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Friction in surface micromachined interfaces

Aug. 18, 2005

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Sandia is a multiprogram laboratory operated by Sandia Corporation, a Lockheed Martin Company,
for the United States Department of Energy's National Nuclear Security Administration
under contract DE-AC04-94AL85000.



Collaborators

Sandia National Labs

- Alex Corwin
- David Luck
- Mike Starr
- Dave Reedy

Auburn U.

- Bob Ashurst

University of Wisconsin

- Rob Carpick
- Erin Flater

Funding:

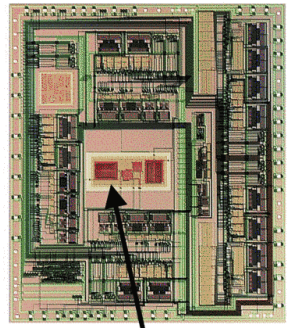
Sandia LDRD programs (DOE)
DOE/BES Center for Excellence in Nanotribology



With polysilicon MEMS we can reliably accomplish electromechanical and optical functions

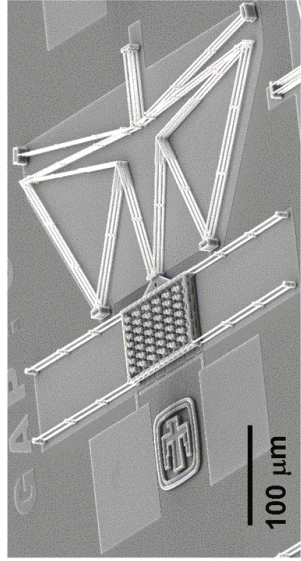
Integrated inertial sensor

- thousands of devices simultaneously
- no assembly required



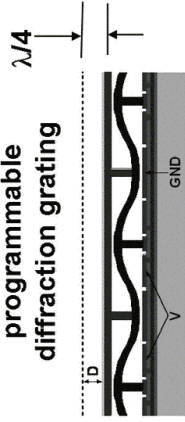
μmachined sensor

High performance comb drive with mechanical amplifier



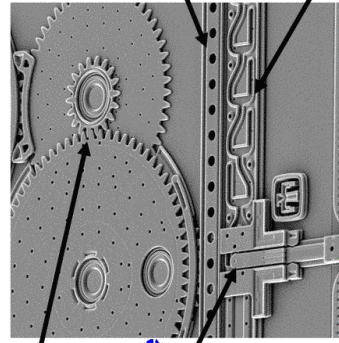
slide 3

Polychromator :
programmable
diffraction grating



Allowing contact between MEMS surfaces significantly broadens the design space

Complex Mechanical Logic



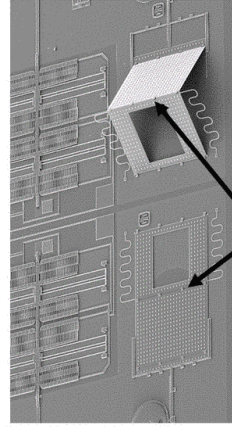
Gears

Pin-in-maze

guides

linear racks

Pop-up Mirrors



hinges

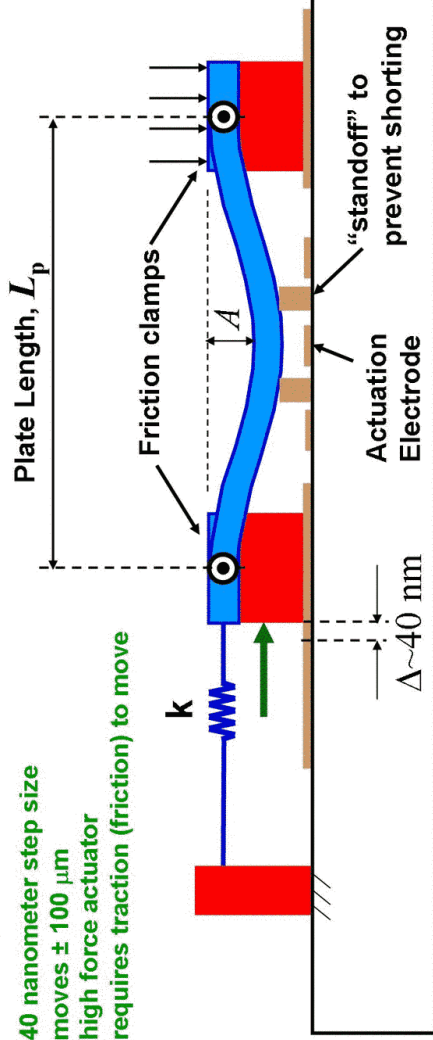
but ...
static friction can dominate the forces required
dynamic friction can dominate energy loss
adhesion, friction and wear become the most important failure mechanisms of contacting MEMS

slide 4



Friction can be good: We developed a high-performance friction-based actuator

- 40 nanometer step size
- moves ± 100 μm
- high force actuator
- requires traction (friction) to move



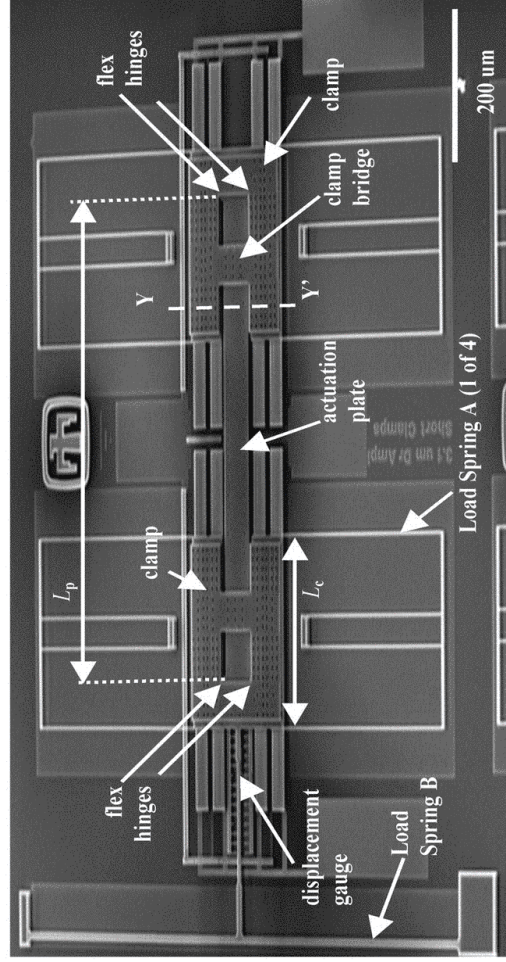
$$F_{\max} \sim 2 \left(\frac{A}{L_p} \right)^2 \approx 1 \text{ mN}$$

large tangential force range



slide 5

Nanotractor - SEM

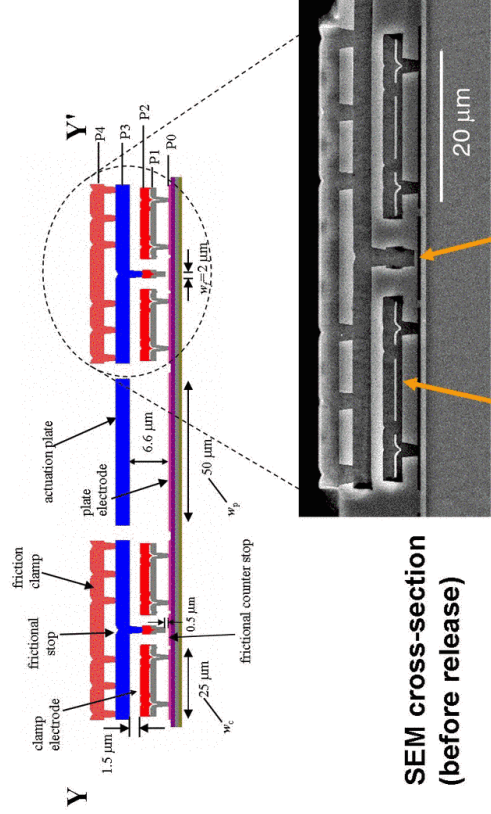


M. P. de Boer, D. L. Luck, W. R. Ashurst et al.
 "High Performance Surface-Micromachined Inchworm Actuator"
 J. MicroElectroMechanical Systems, Feb. 2004



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Nanotractor – clamp cross section



SEM cross-section (before release)

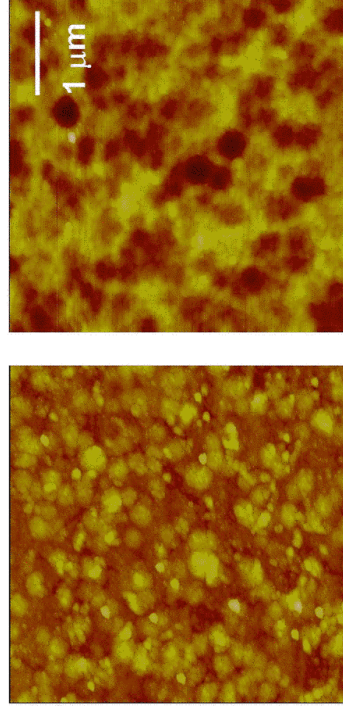
Normal force is applied electrostatically and borne mechanically
 We can apply normal force from 1 μN to 10 mN with this arrangement.

large normal force range



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Surface contact is an aggregate of asperities



bottom counterface (top of P0, 8 nm rms) top counterface (bottom of P12, 5 nm rms)

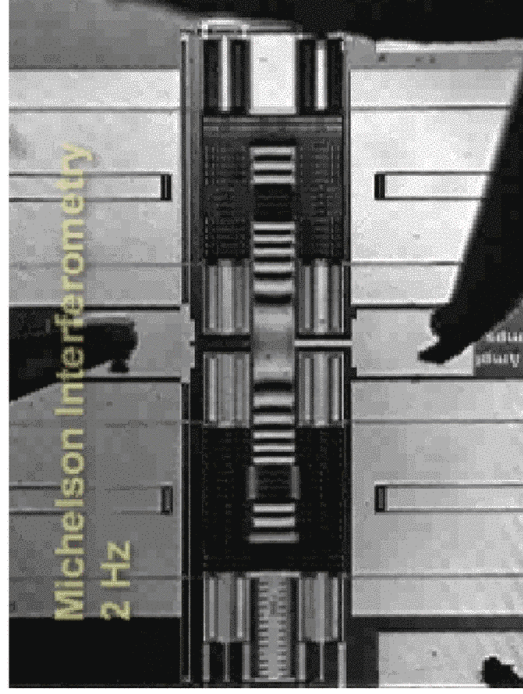
Rough surface contact mechanics considerations ...

- asperity radius of curvature $R \sim 20$ to 500 nm (typically ~ 50 nm)
- rms roughness 1.5 to 10 nm
- contact diameter ~ 10 nm, pressure ~ 10 GPa
- real contact area $\ll 10^{-3}$ (apparent contact area)



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Nanotractor in operation



slide 9



Monolayer coatings strongly affect frictional behavior

FOTAS (tridecafluoro-1,1,2,2-tetrahydrodecyltris(dimethylamino)silane)

vapor deposition

8 carbon chain

van der Waals forces not strong enough to self assemble (tangled)

contact angle $\sim 110^\circ$

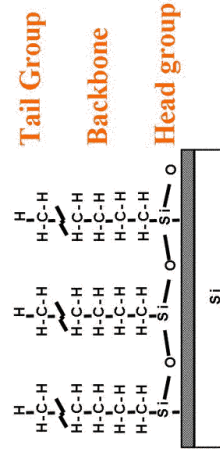
OTS (octadecyltrichlorosilane)

liquid deposition

18 carbon chain

van der Waals forces strong enough to self assemble.

contact angle $\sim 115^\circ$

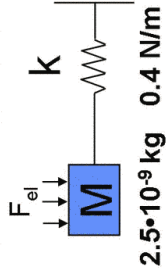
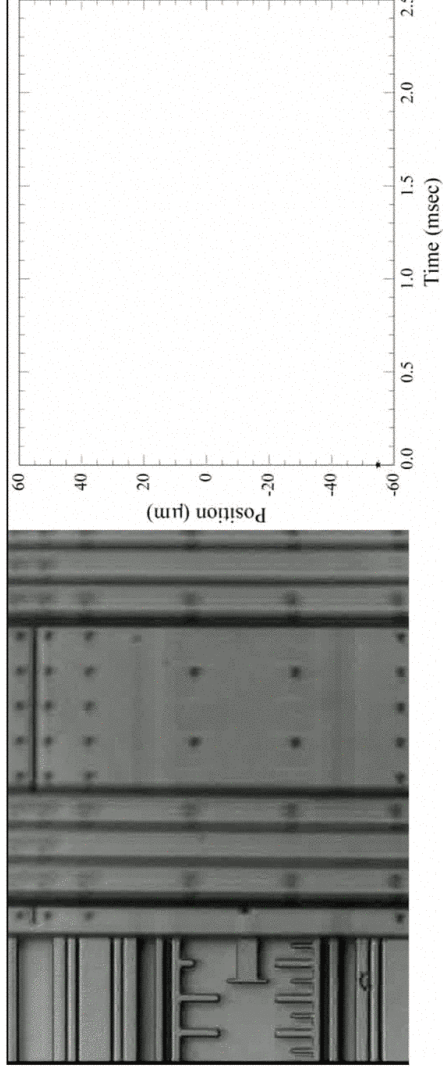


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Damped oscillator methodology to obtain true dynamic coefficient of friction

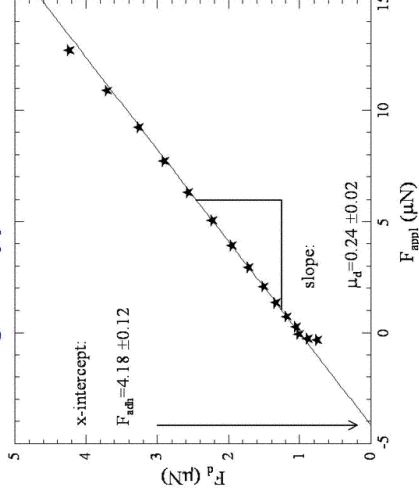
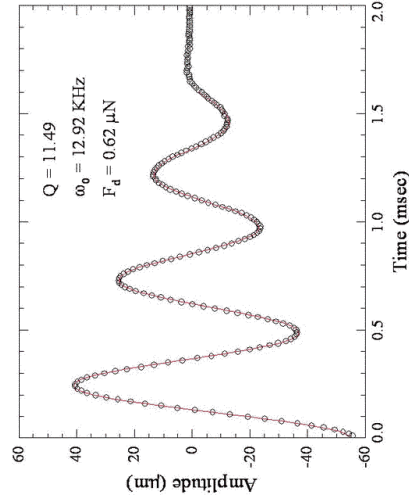
dynamic friction test at small tensile load (FOTAS monolayer):



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There is friction at zero applied load

FOTAS
 Measured and modeled fit for zero applied load
 Dynamic friction over a range of applied loads



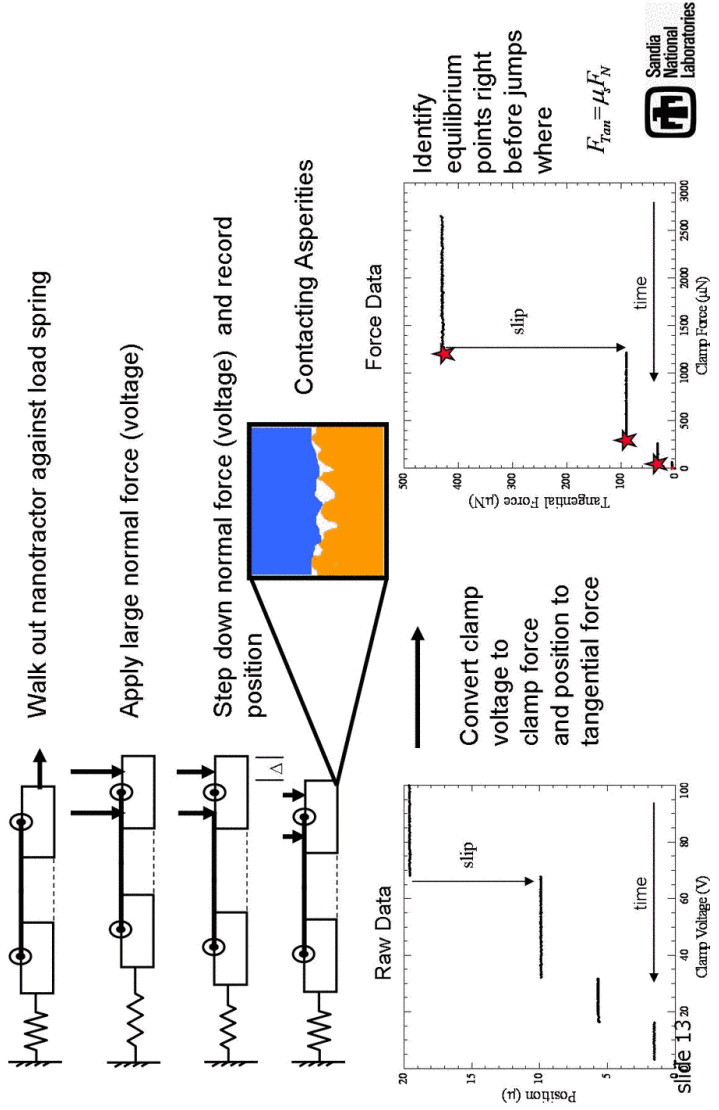
$$F_d = \underbrace{\mu_s(F_c + mg + k_z z)}_{F_{\text{appl}}} + \mu_s F_{\text{adh}}$$

Effect of adhesion on dynamic and static friction in surface micromachining. Conwin, AD, de Boer, MP Applied Physics Letters; 29 March 2004; vol.84, no.13, p.2451-3



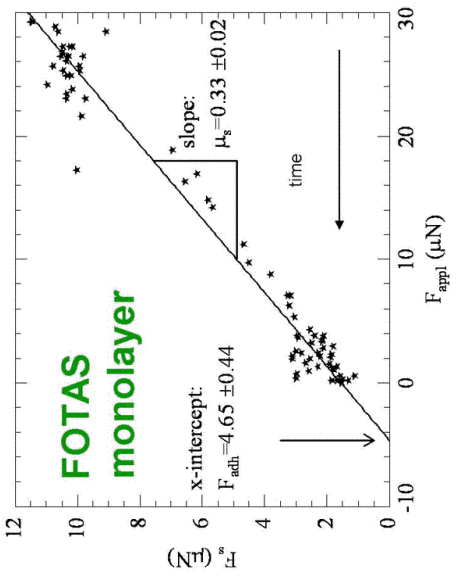
slide 12

Static friction measurement



Static Friction Data and Analysis

Plot tangential force as function of applied electrostatic force
 Can fit this with Amontons' law if we include adhesive term:



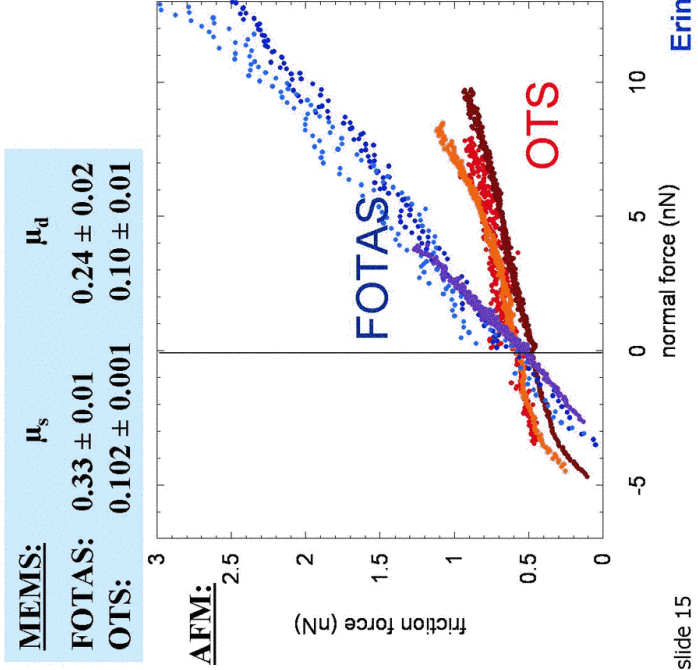
$\mu_s = 0.33 \pm 0.02$
 $F_{adh} = 4.6 \pm 0.5 \mu\text{N}$

Under static conditions, adhesion also gives rise to friction
Adhesive contribution to force is ~4.7 μN (~1 nN/ μm^2)

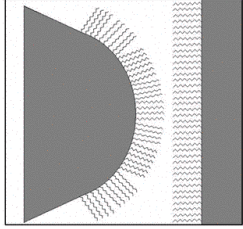


Multi-asperity (MEMS) to single-asperity (AFM) comparison: qualitative agreement

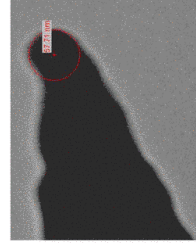
| MEMS: | μ_s | μ_d |
|--------|-------------------|-----------------|
| FOTAS: | 0.33 ± 0.01 | 0.24 ± 0.02 |
| OTS: | 0.102 ± 0.001 | 0.10 ± 0.01 |



AFM:
both tip and substrate are coated



TEM of AFM tip

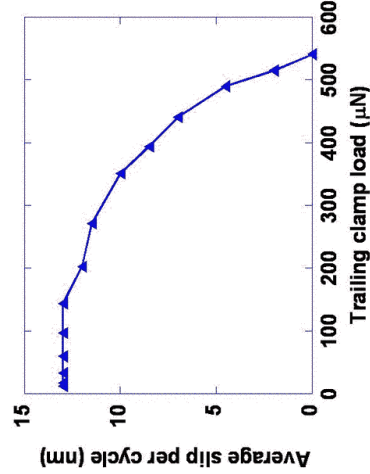
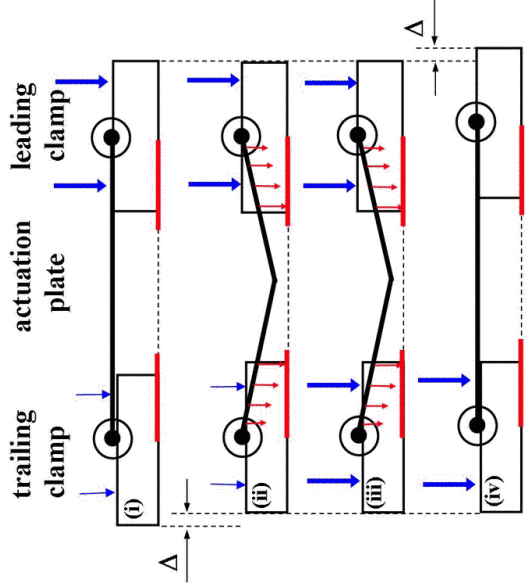


Erin Flater & Rob Carpick, U. Wisc.

Do μ_s and μ_d describe the nanotractor under operational conditions?

FOTAS
monolayer

Measured Data



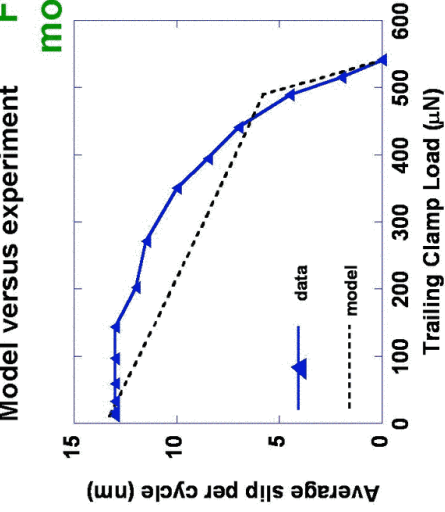
slide 16



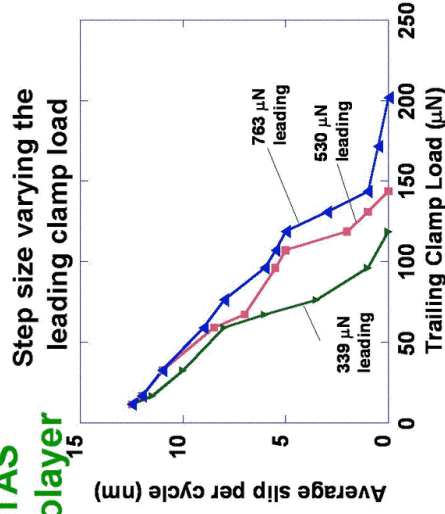
Modeling suggests that μ_s and μ_d CANNOT describe device operation

Model versus experiment

FOTAS monolayer



Although agreement may appear reasonable, axial hinge compliance value is unrealistic (value used to fit data is 50 to 100 times too large)



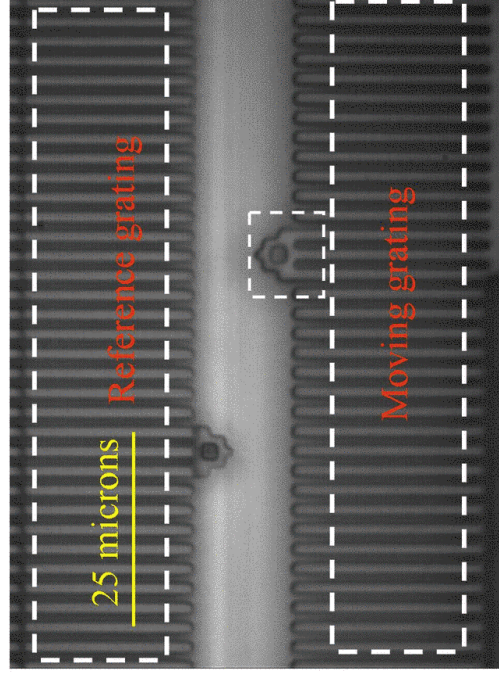
The leading clamp must be slipping backwards!!

D. L. Luck, M. P. de Boer, W. R. Ashurst, A. D. Corwin and M. S. Baker
 "Evidence for pre-sliding tangential deflections in MEMS"
 Transducers 2003 Conference

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Highly resolved slip measurements: we measure position to 1 nm

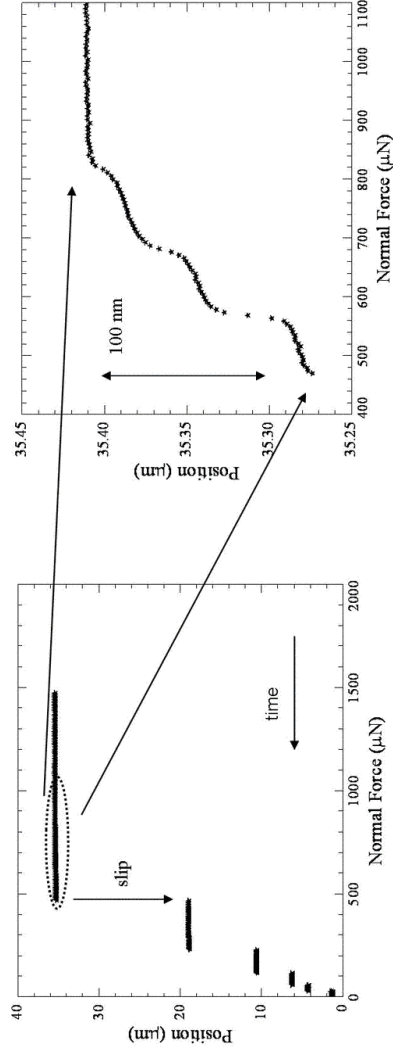
Use periodic grating and measure relative phase to 1 part in 2500



slide 18

Pre-sliding tangential deflections (PSTD)

FOTAS monolayer coating

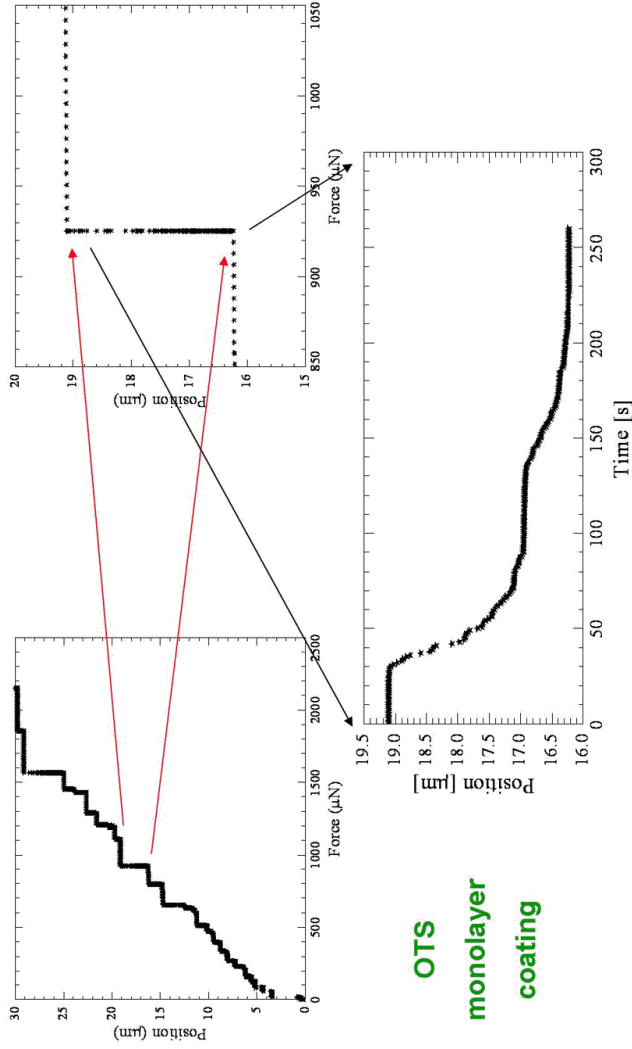


Stable in-plane deflections occur before gross sliding event.



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OTS shows CREEP rather than PSTD

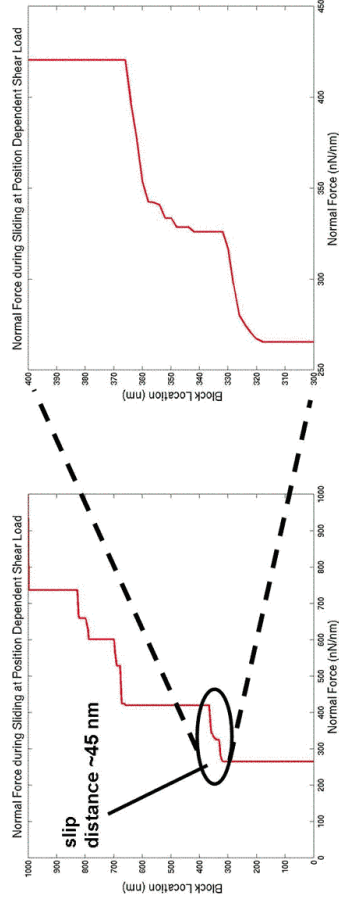


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Wait for equilibrium

Simulations indicate PSTD may be stochastic

FOTAS monolayer simulation



Simulation:

2-D linescan of the surface (from real AFM data)

- Calculate real contact area at a large normal load (Hertz only)
- Reduce load -> contact area reduces until frictional force is surmounted
- Clamp moves forward (1 nm), and tangential load reduces slightly
- If new position has enough contact area, then new position is stable
- Otherwise get a large position jump

slide 21



Mike Starr, Sandia

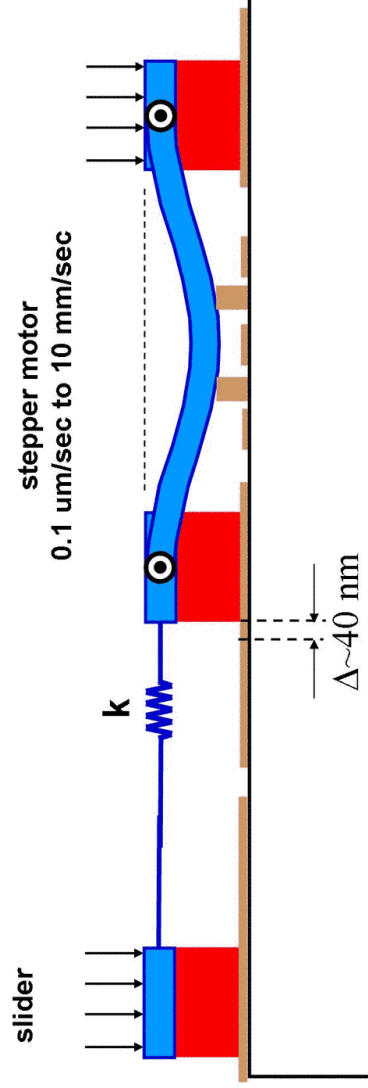
Comparison to Rate & State Phenomenology

| <u>Rate & State</u> | <u>FOTAS</u> | <u>OTS</u> |
|-------------------------|---------------|-----------------|
| Do/a~1 | pstd: Do/a~10 | creep: Do/a~300 |
| us > ud | us > ud | us~ud |
| us=us(t) | unclear | unclear |

slide 22



More direct comparison to rate/state experiments is possible



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Summary & Conclusions

- MEMS can serve as a testbed for multi-asperity friction experiments
- At low loads, normal force is dominated by adhesion
- Monolayer static friction correlates from nano to microscale
- PSTD events can govern operation of MEMS nanostructure
- PSTD may be a stochastic effect
- Creep can also dominate and depends on monolayer lubricant

slide 24

