

# Biomimetic 4D Printing

Sabetta Matsumoto  
KITP 22 January 2016

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# Shape-morphing Systems



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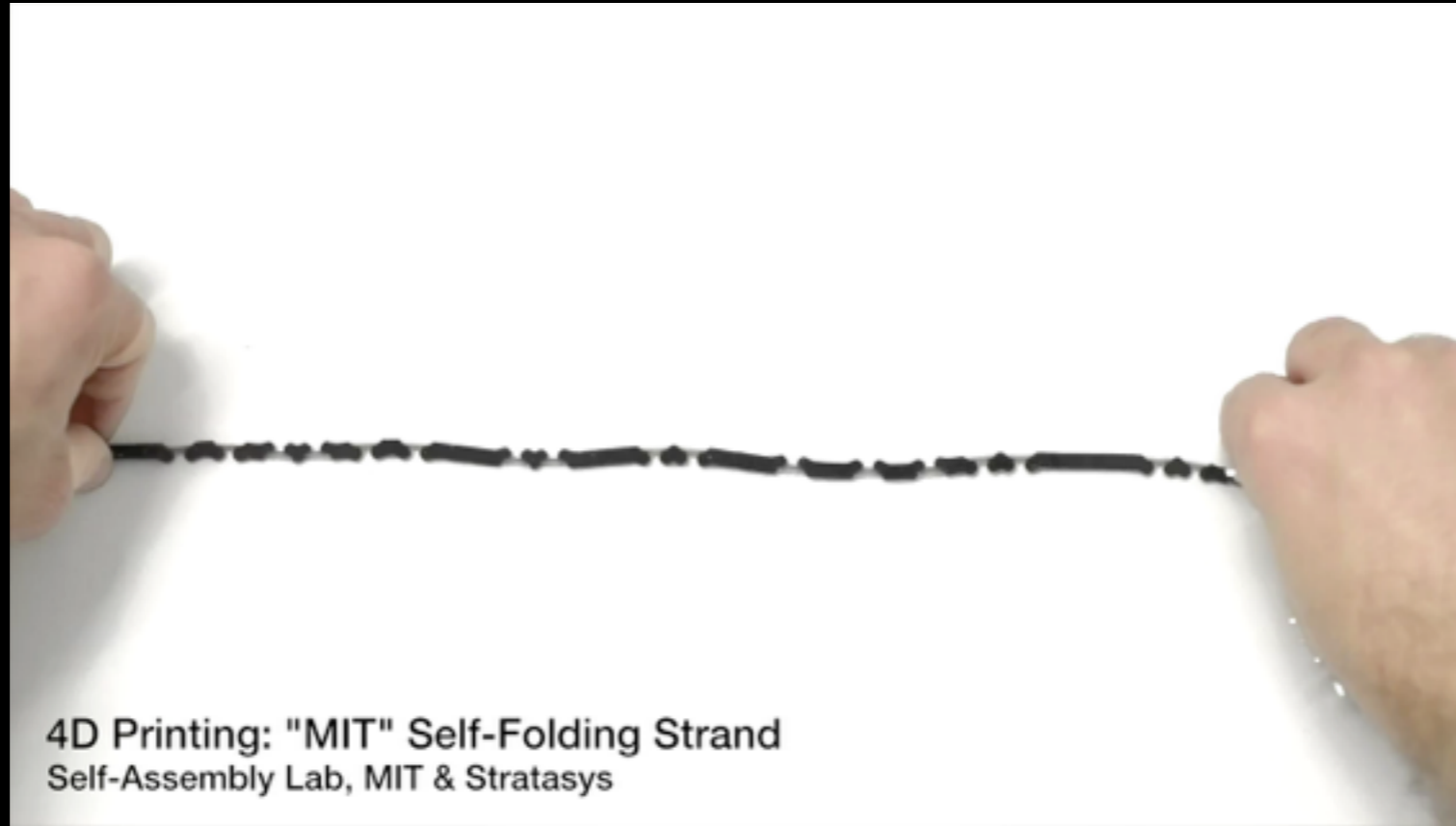
# Shape-morphing Systems



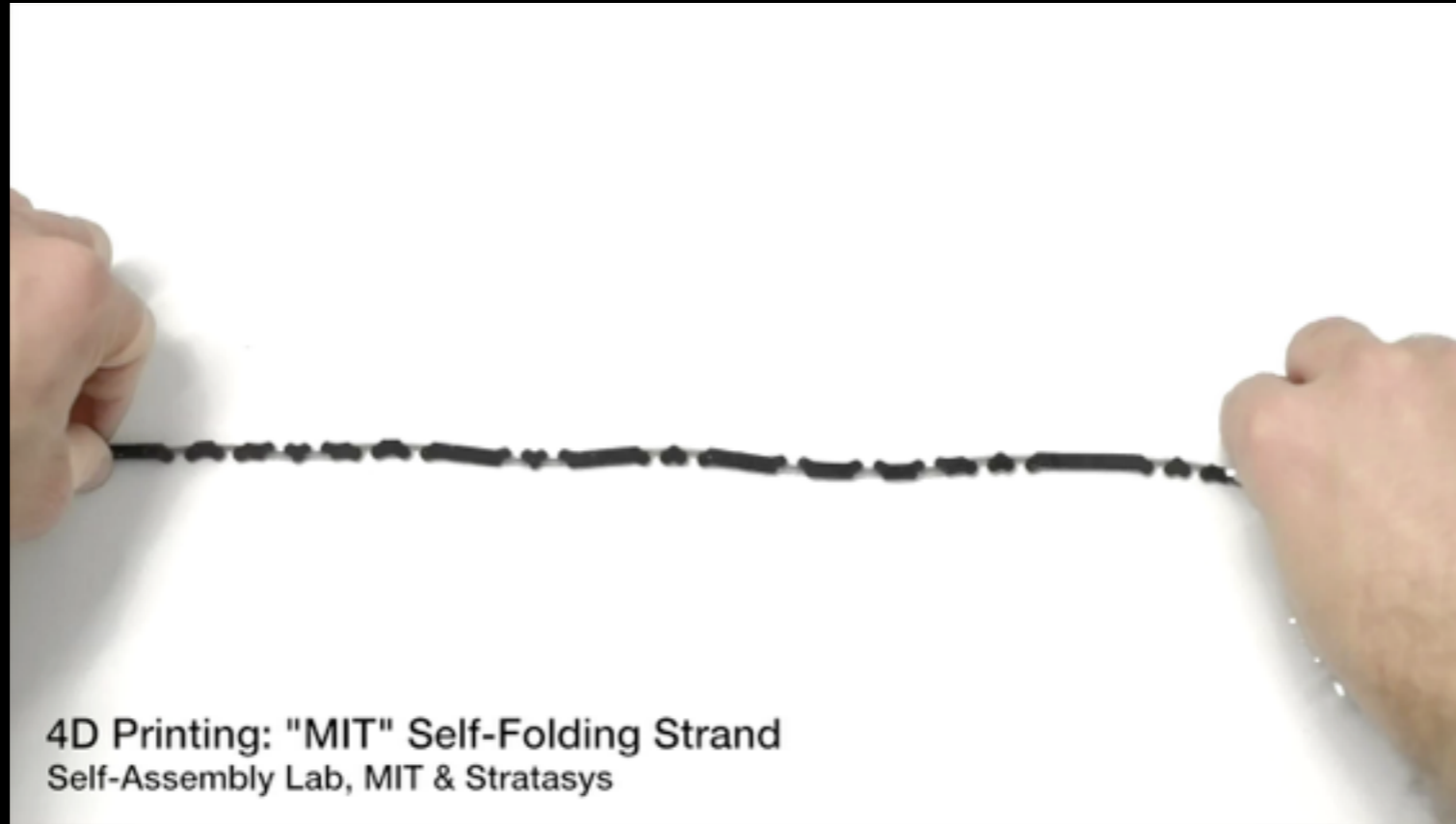
# Shape-morphing Systems



# 4D Printing

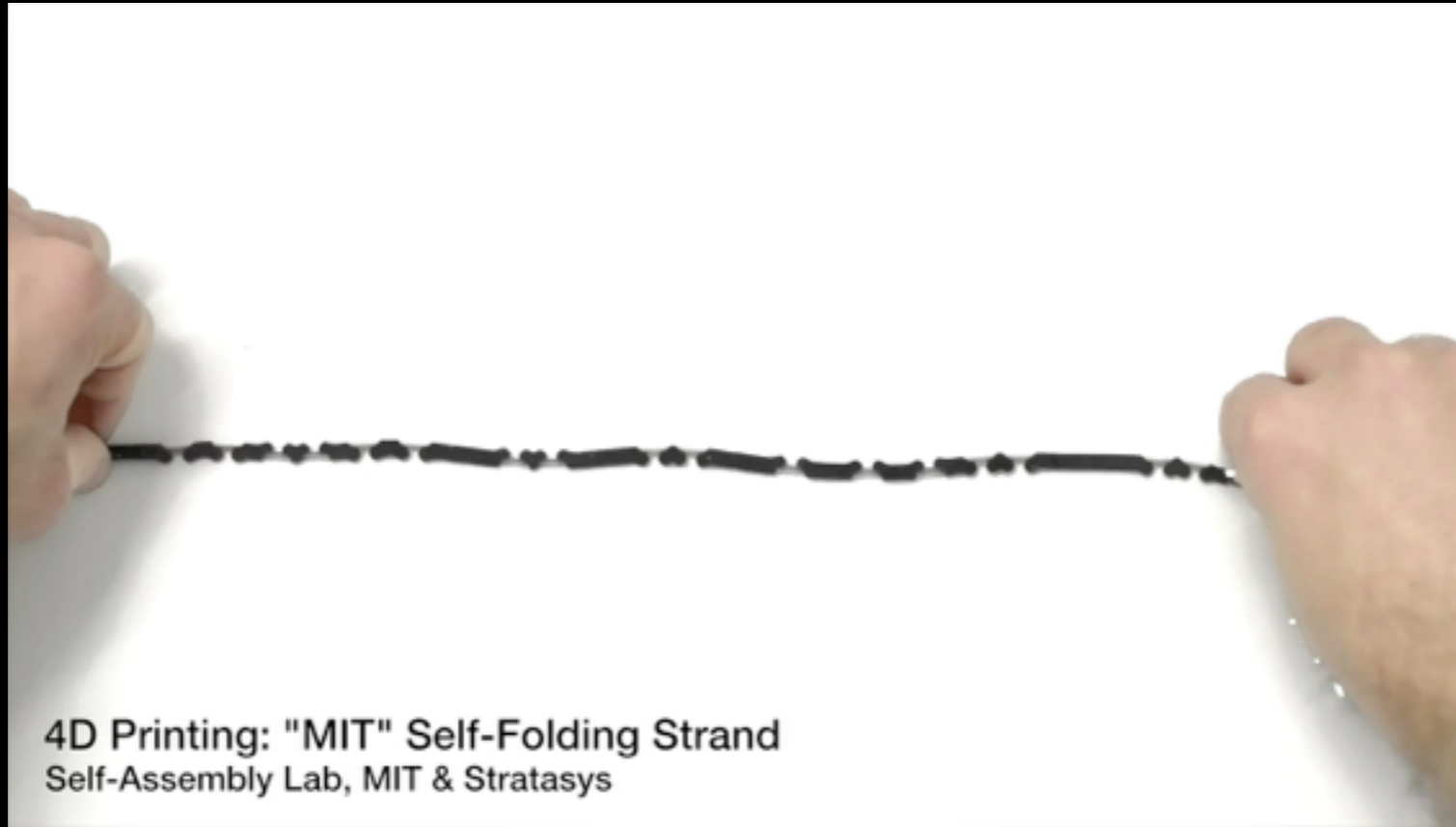


# 4D Printing





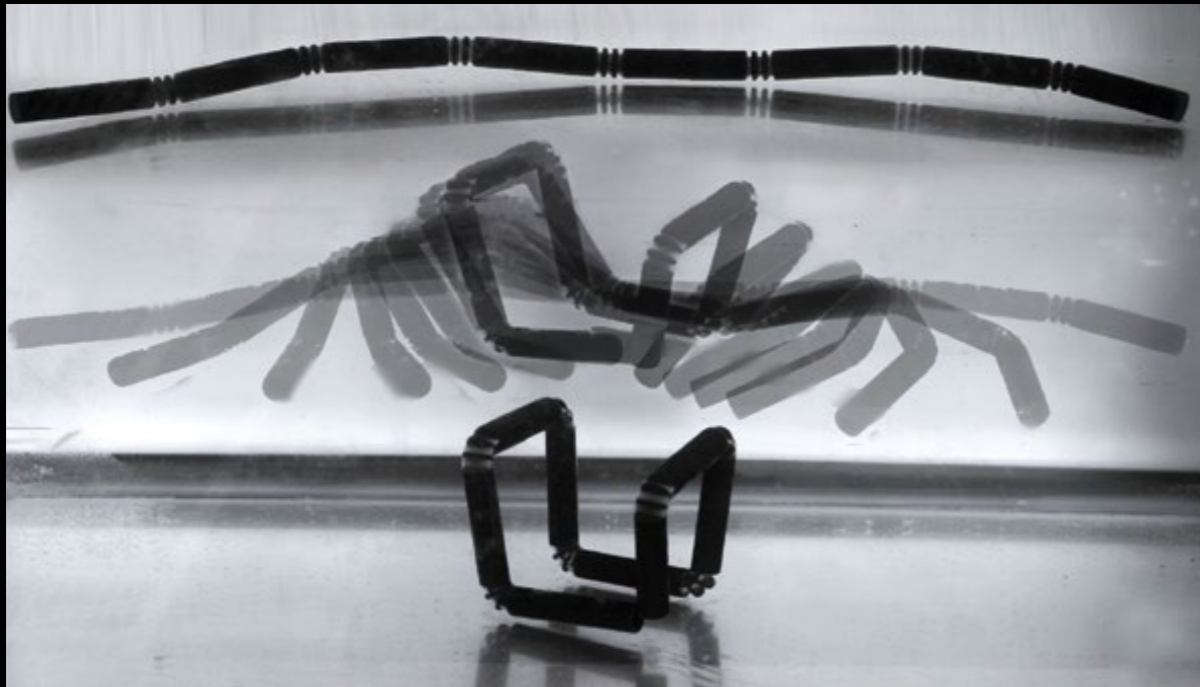
# 4D Printing



4D Printing: "MIT" Self-Folding Strand  
Self-Assembly Lab, MIT & Stratasys

# Shape-morphing Systems

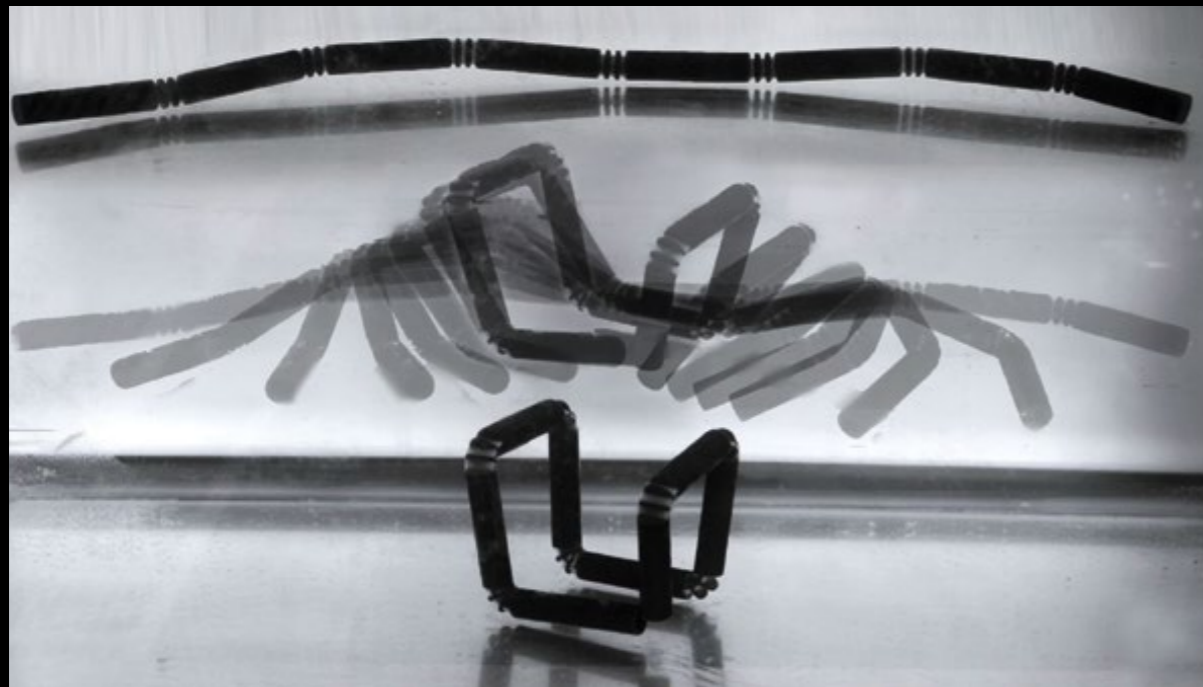
## Mechanical Hinges



Tibbits, *Arch. Design* **84** | 16 (2014)

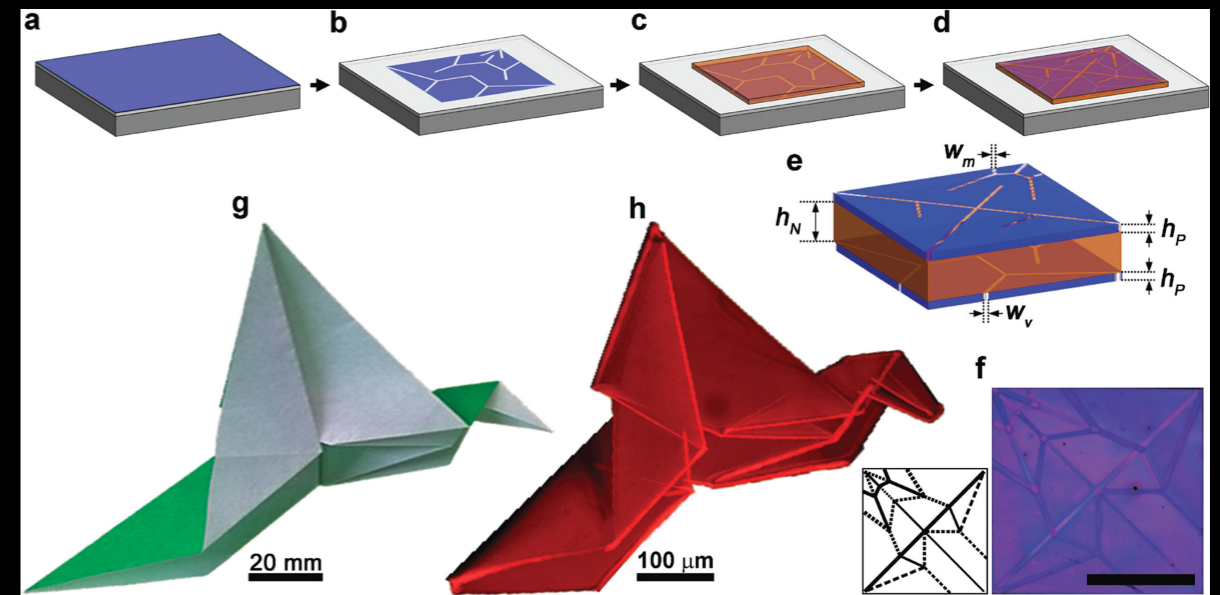
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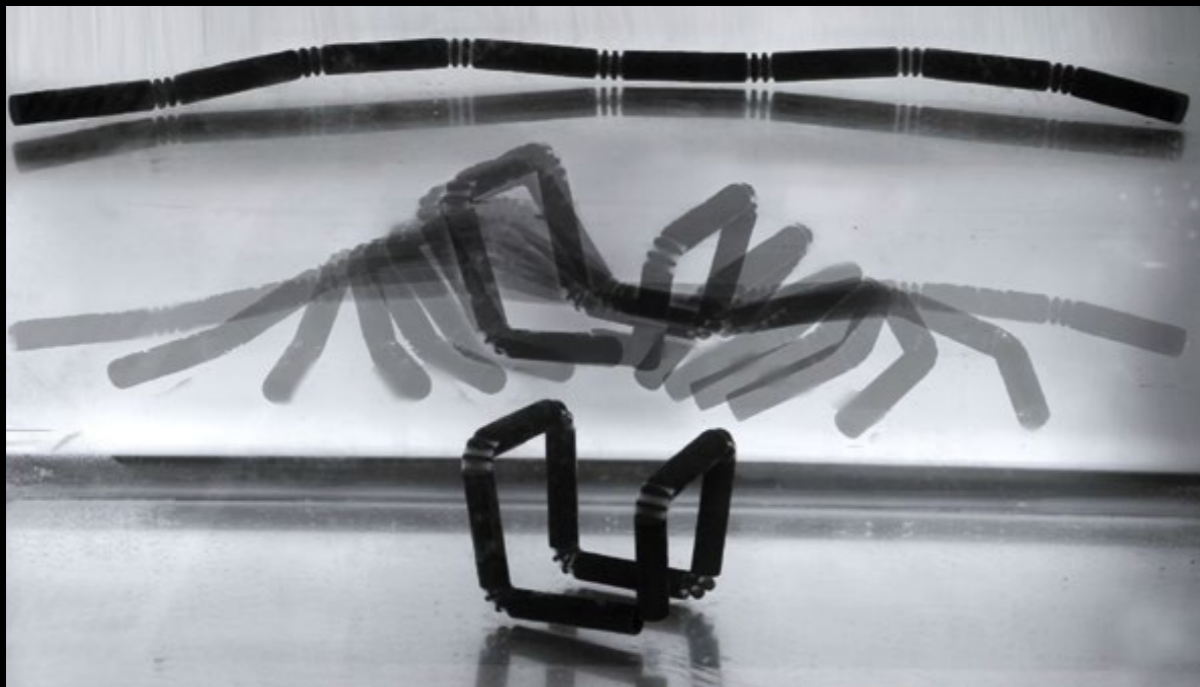
## Origami



Na, et al. *Adv. Mat.* **27** 79 (2015)

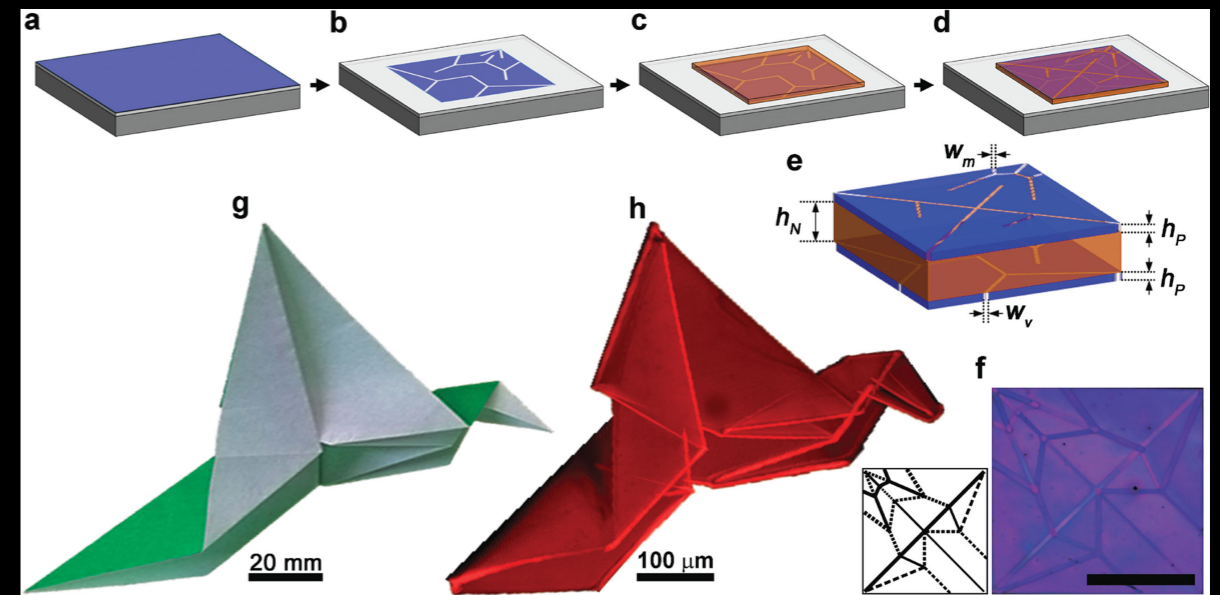
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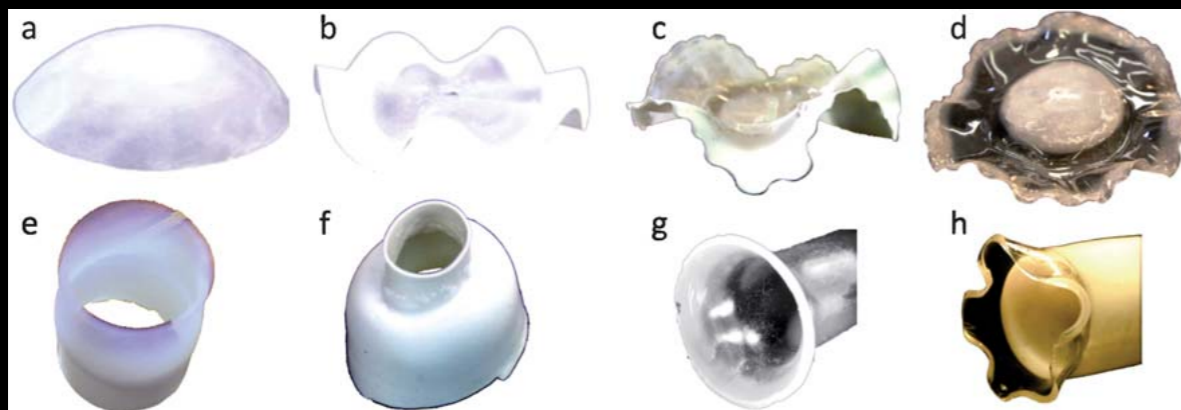
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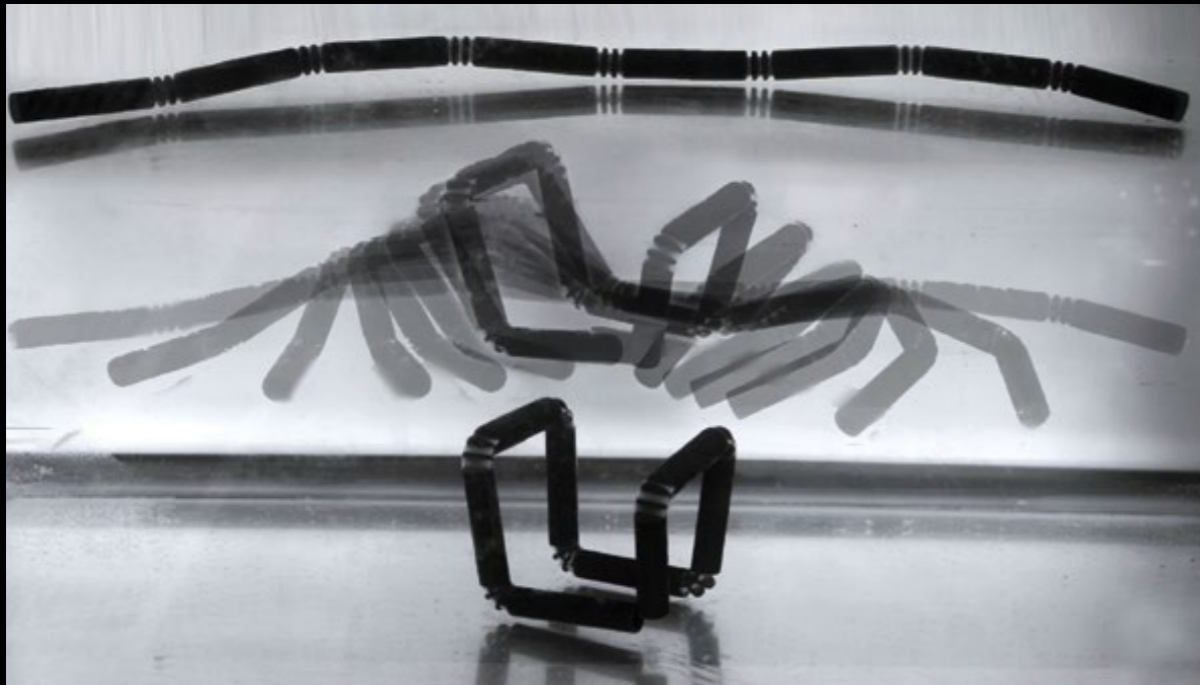
## Swelling Hydrogels



Sharon & Efrati, *Soft Matter* **6** 5693 (2010)

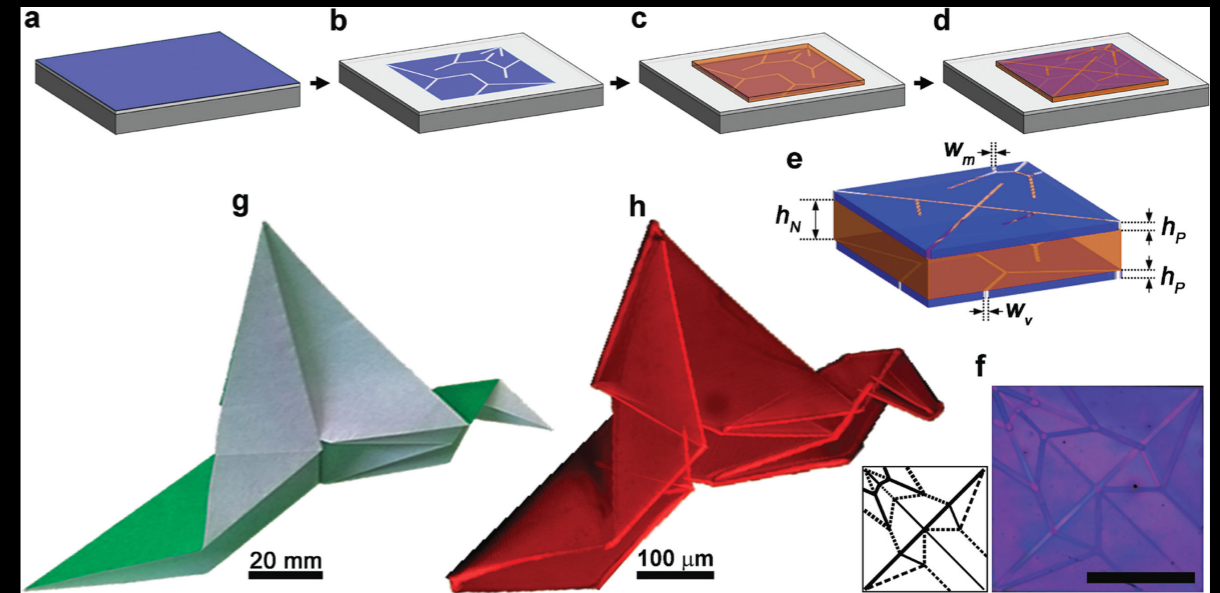
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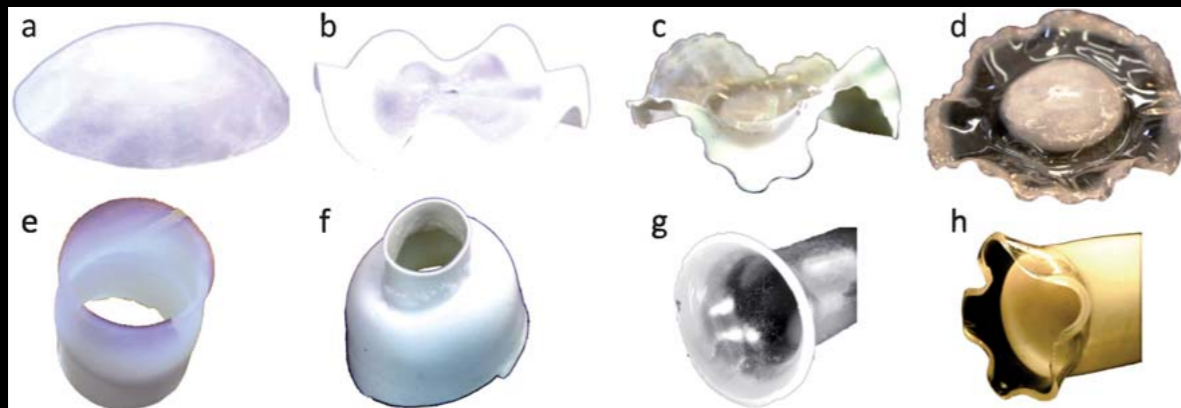
Tibbits, *Arch. Design* **84** 116 (2014)

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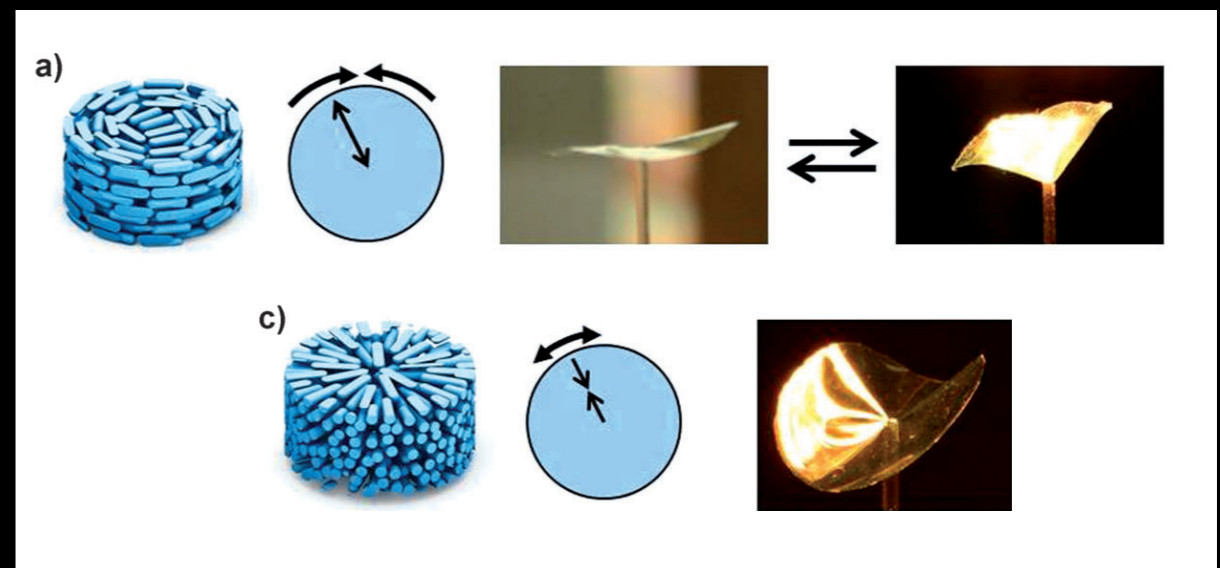
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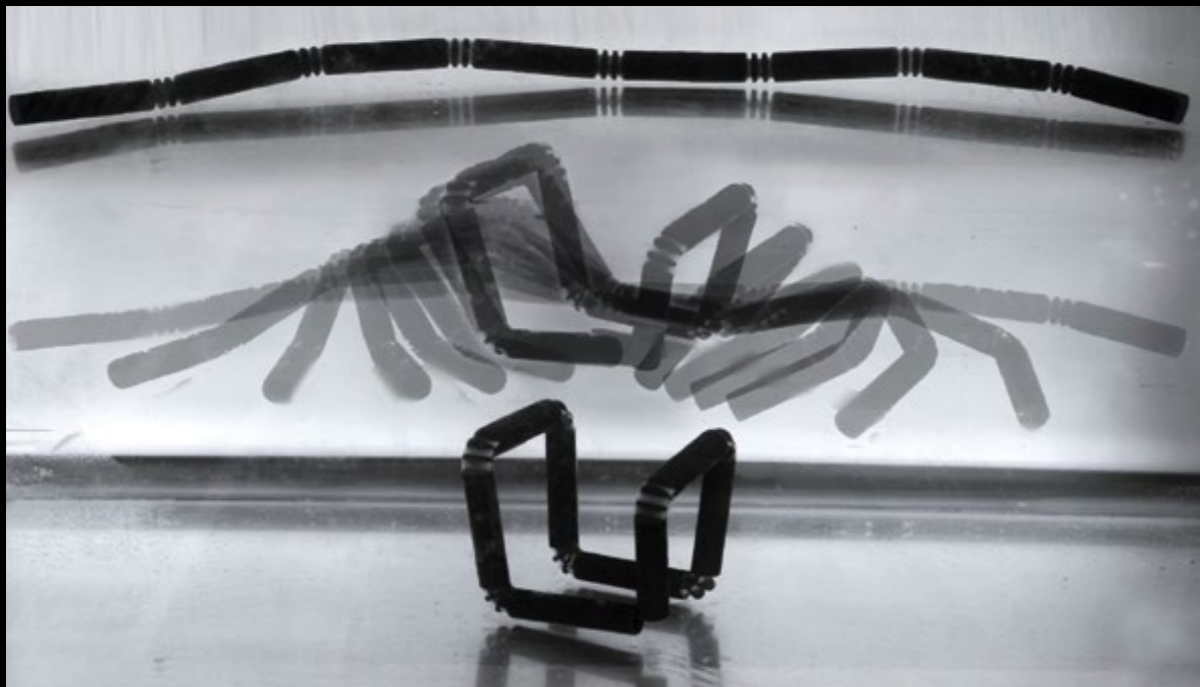
## Liquid Crystal Elastomers



de Haan, et al. *Angew. Chem.* **51** 12469 (2012)

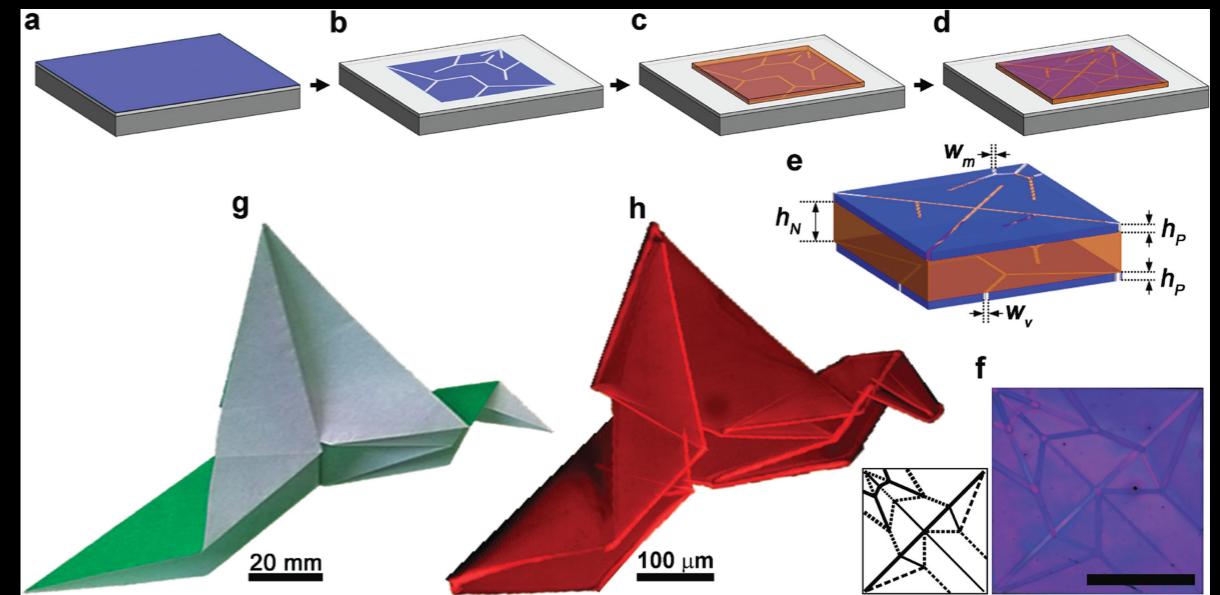
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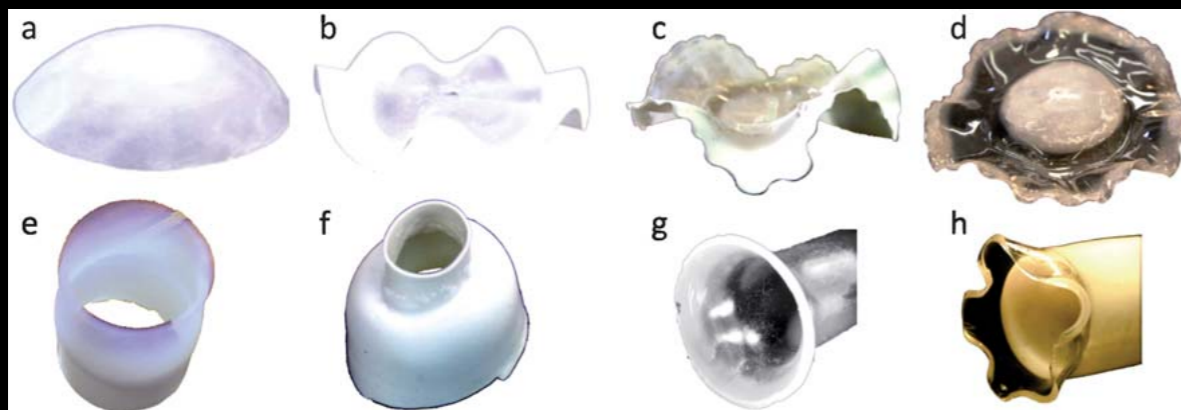
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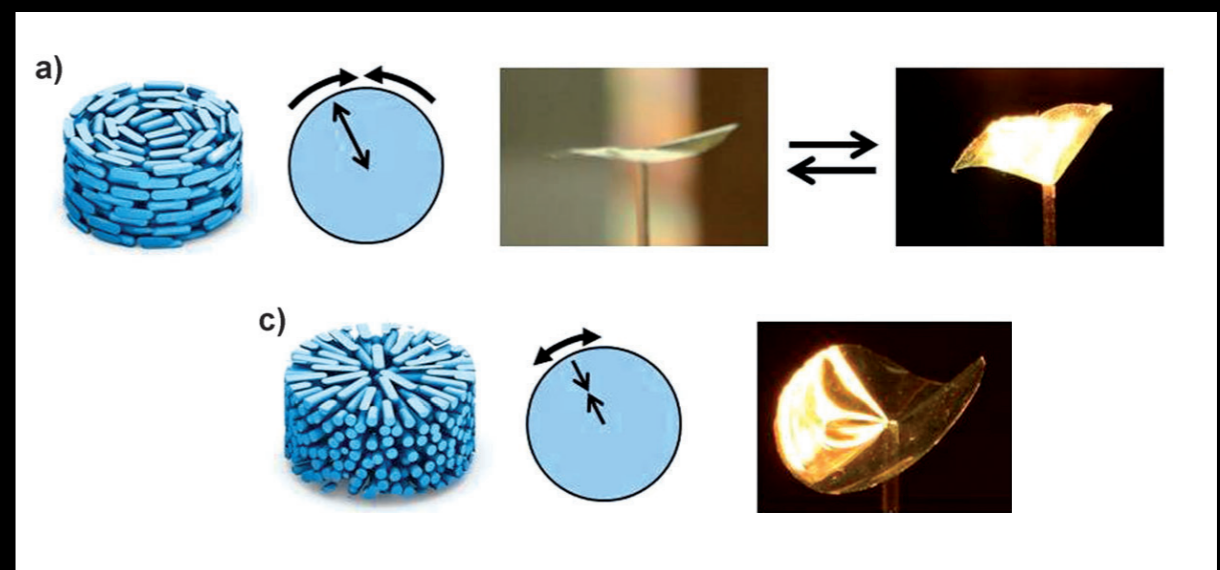
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# Hygroscopic Motion

Pine Cone

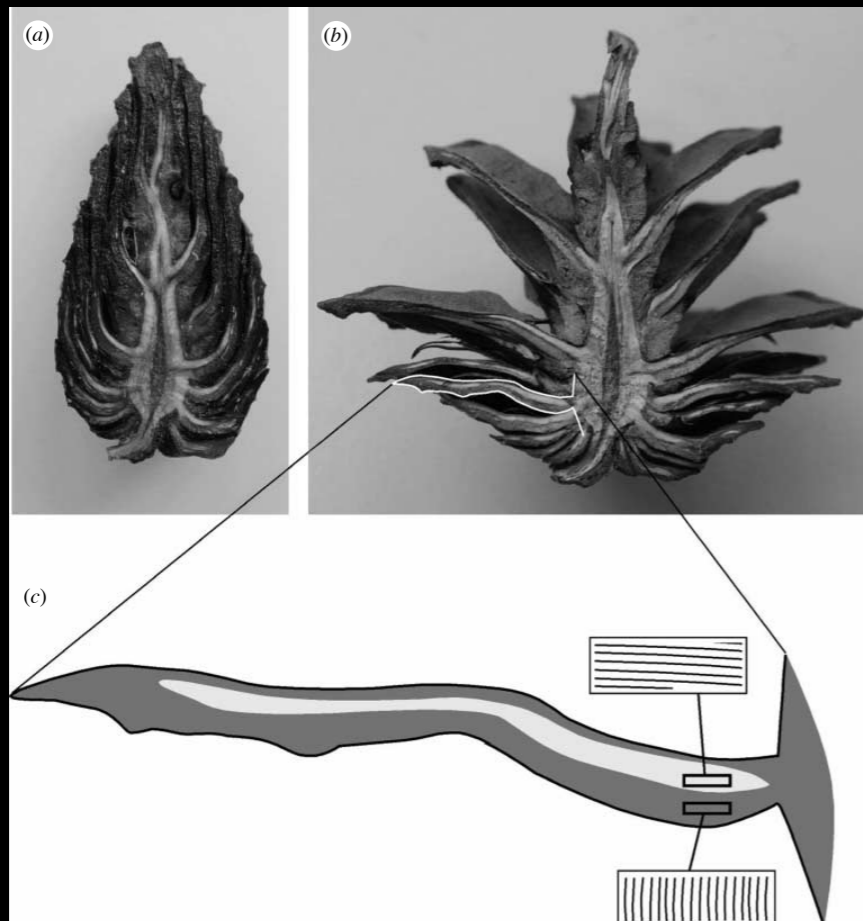
Erodium Awn



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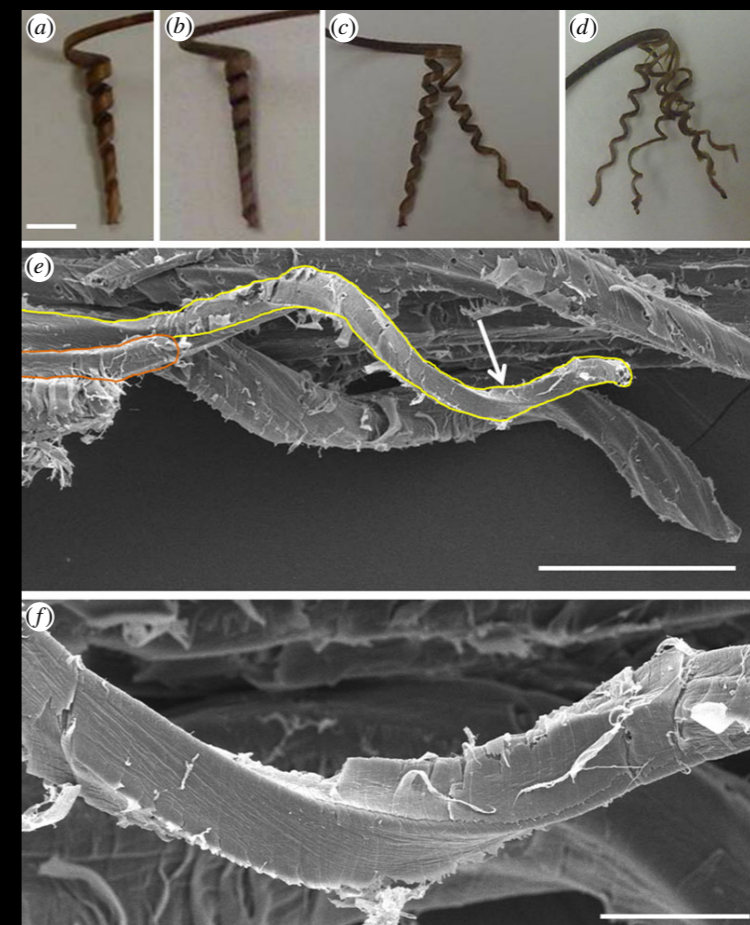
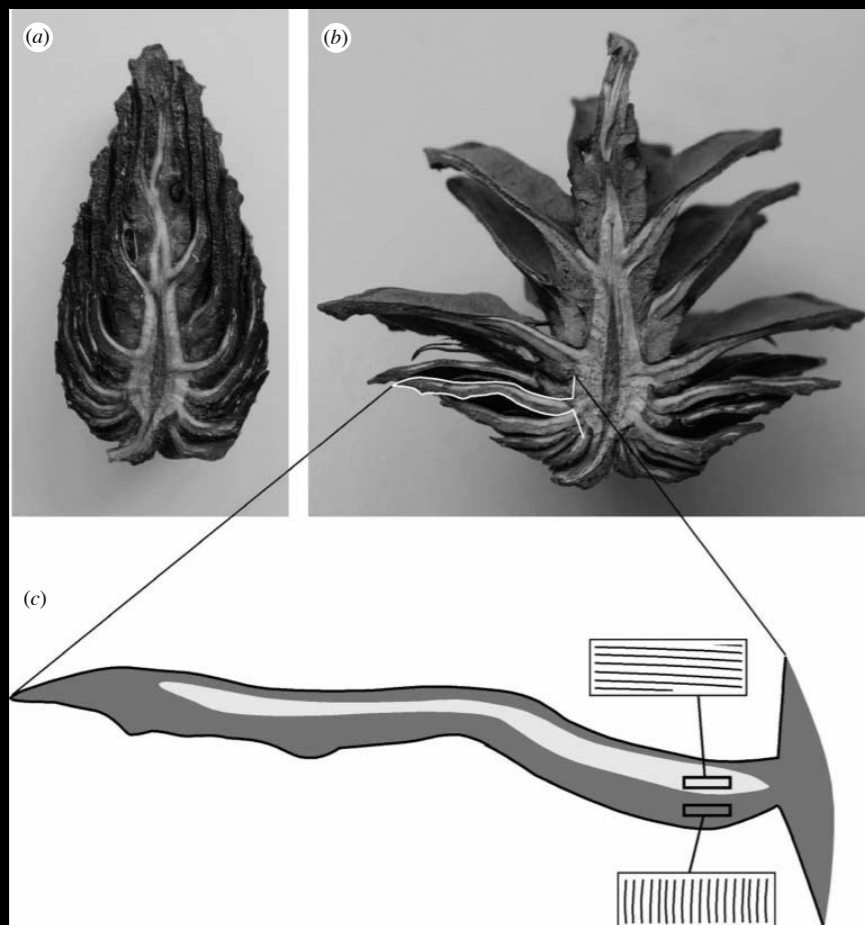


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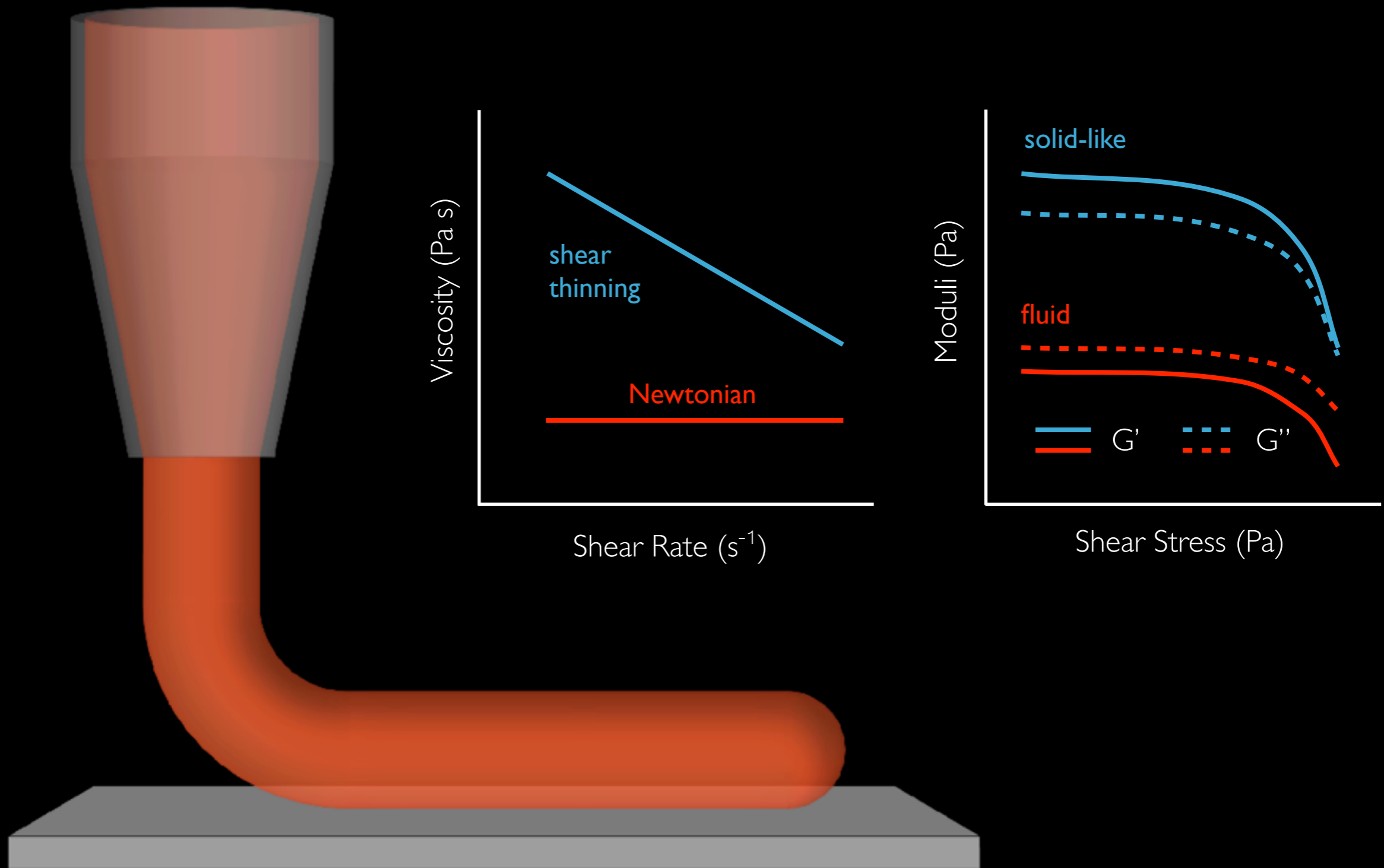
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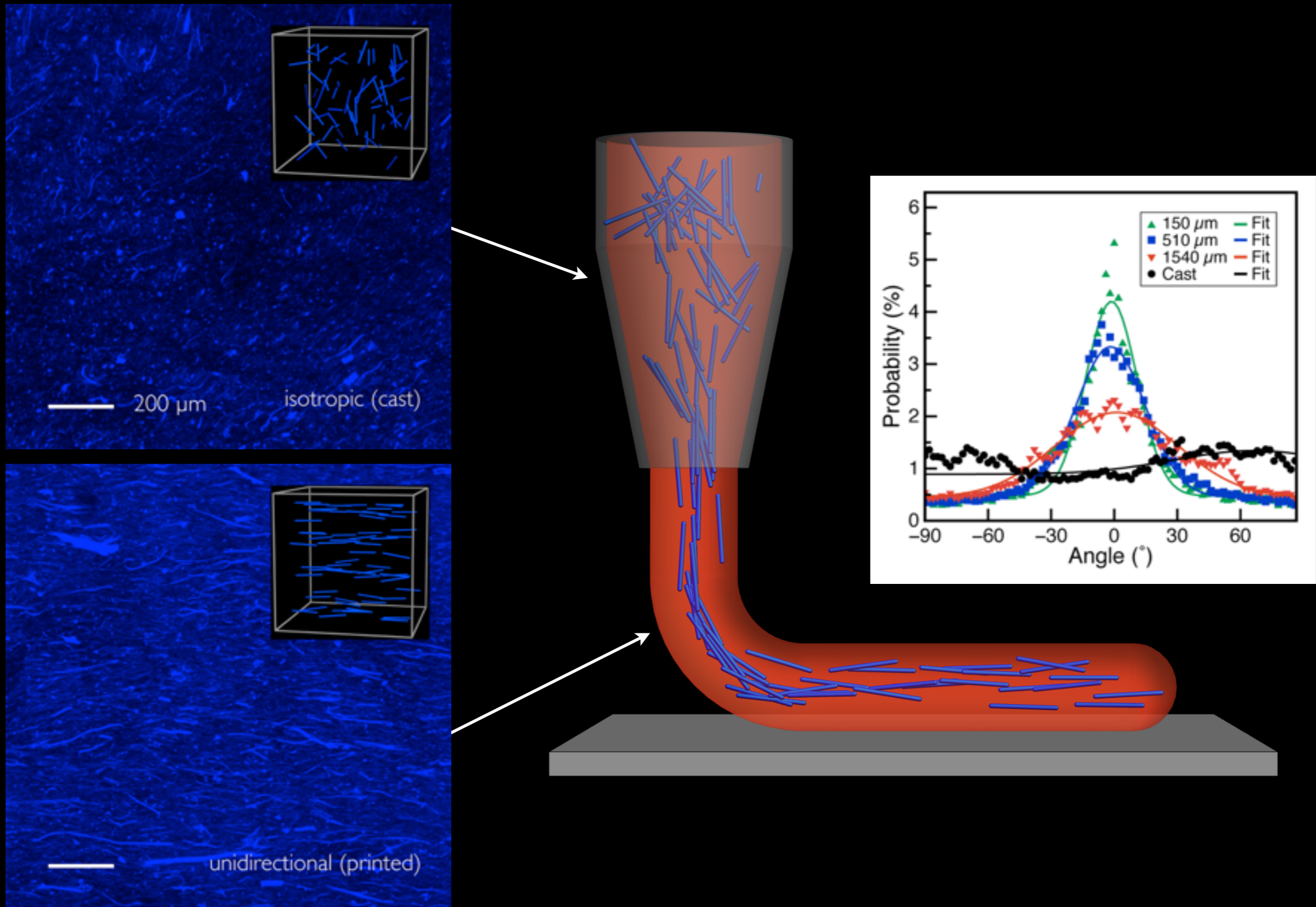


# 3D Printing



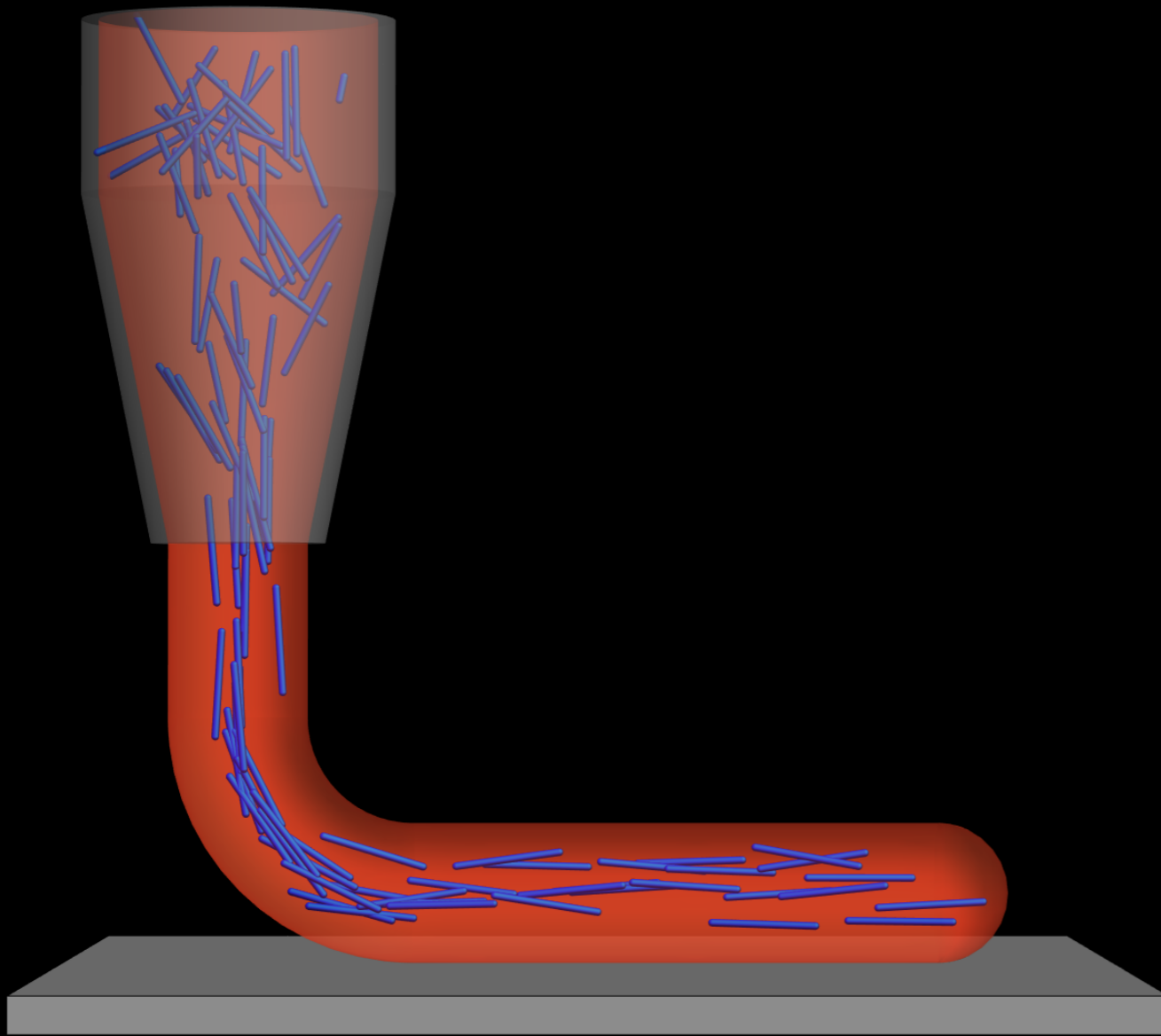
# Our Ink

Cellulose Nanofibrils + Acrylamide Monomers + Clay = Composite Ink



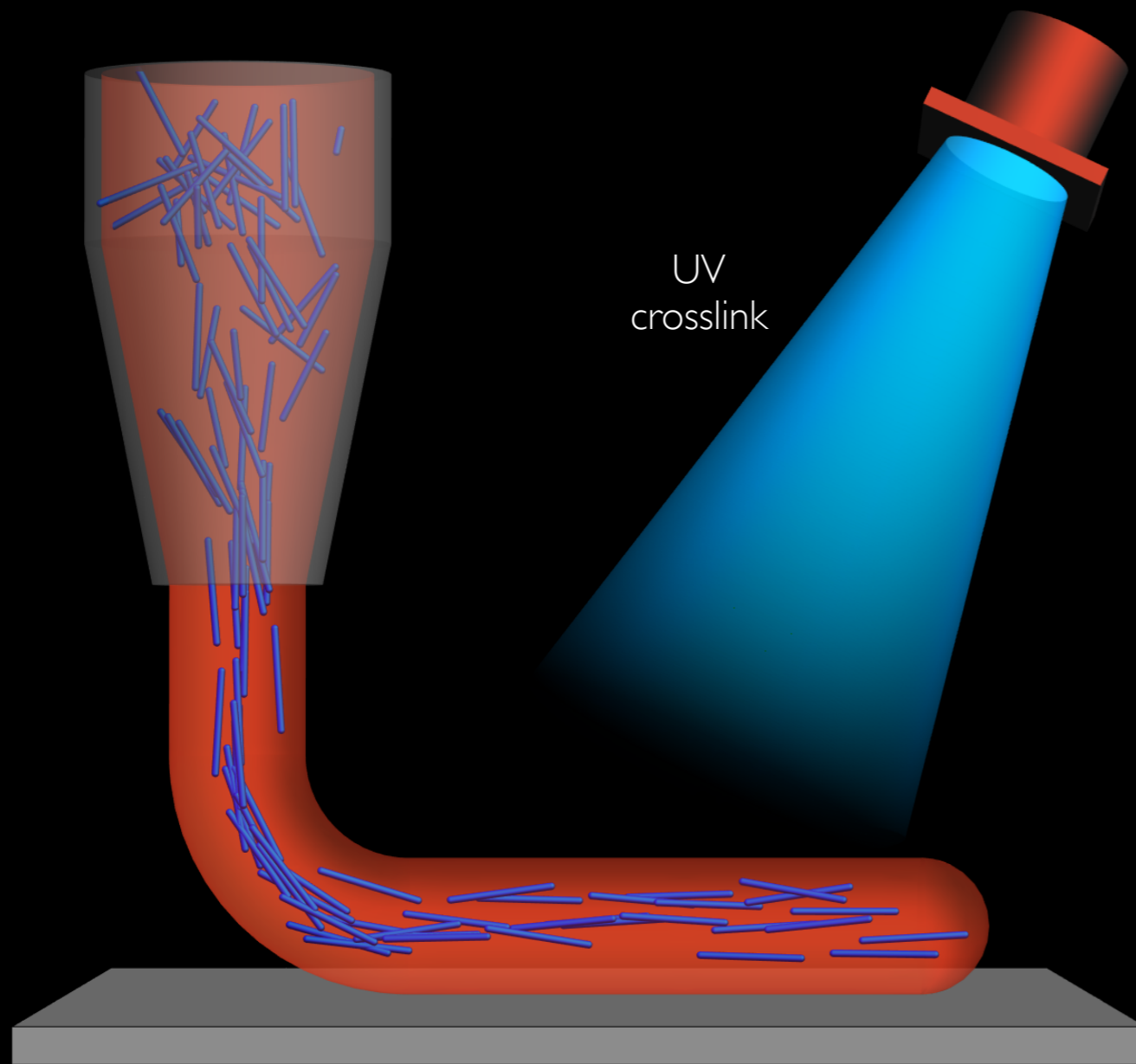
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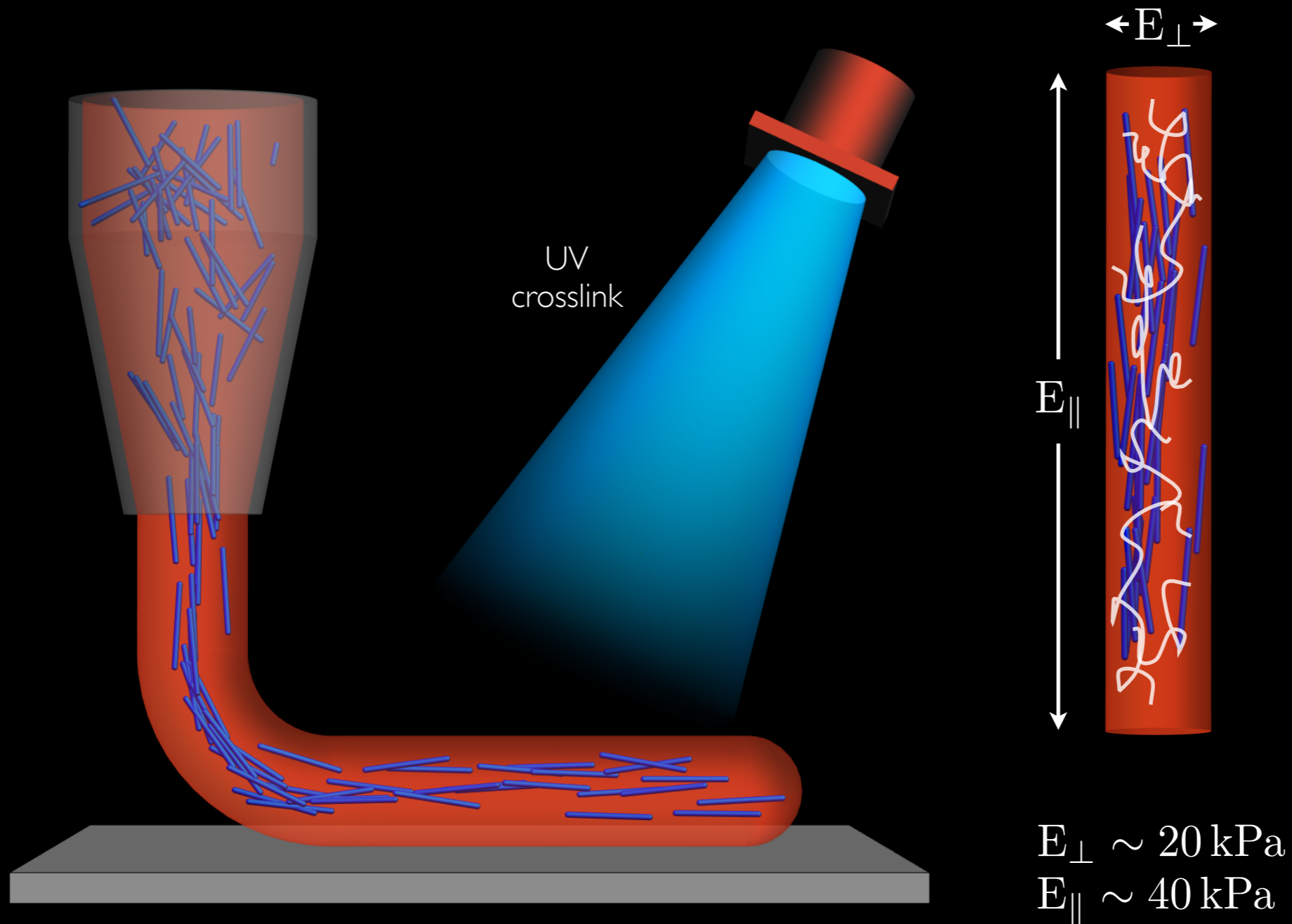
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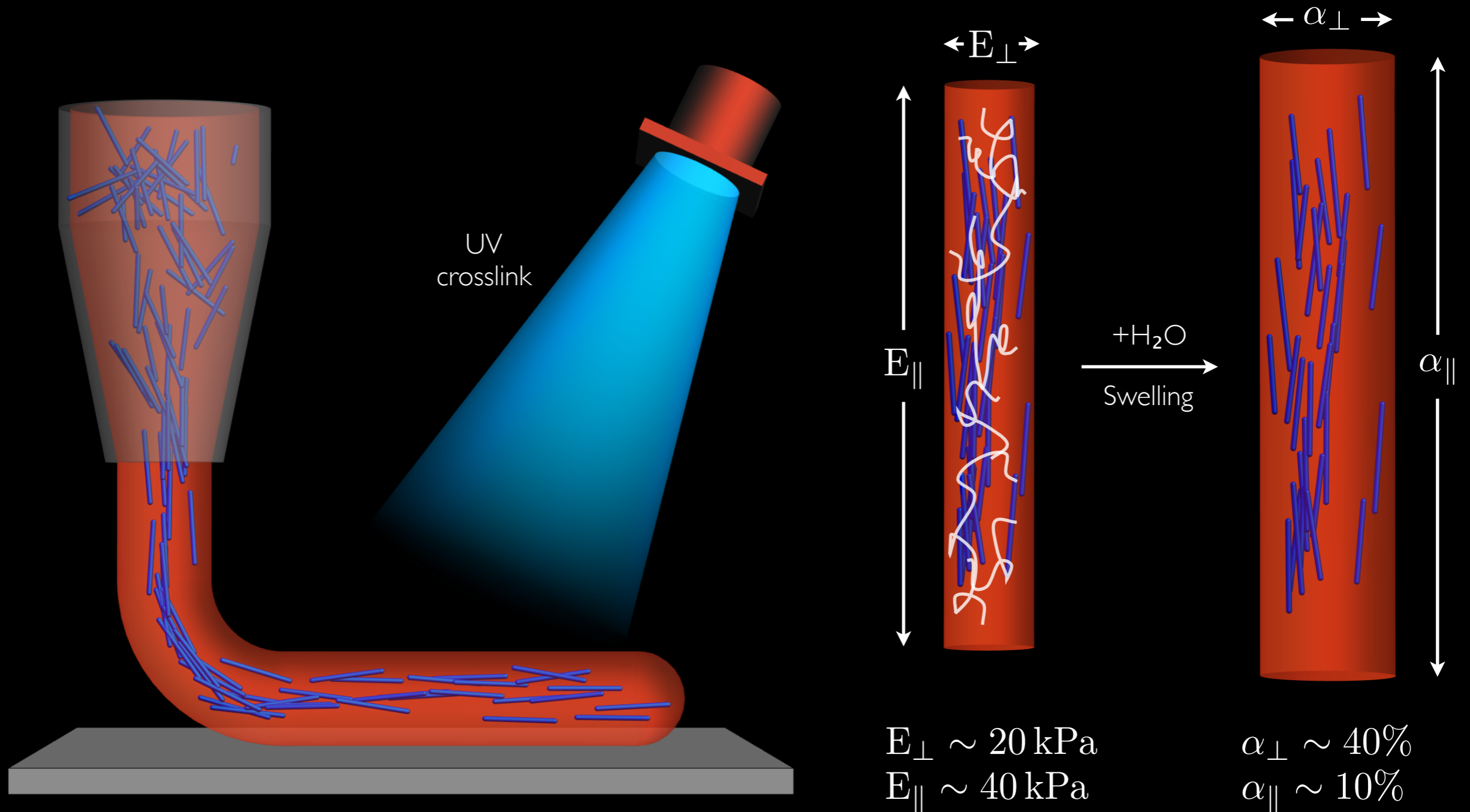
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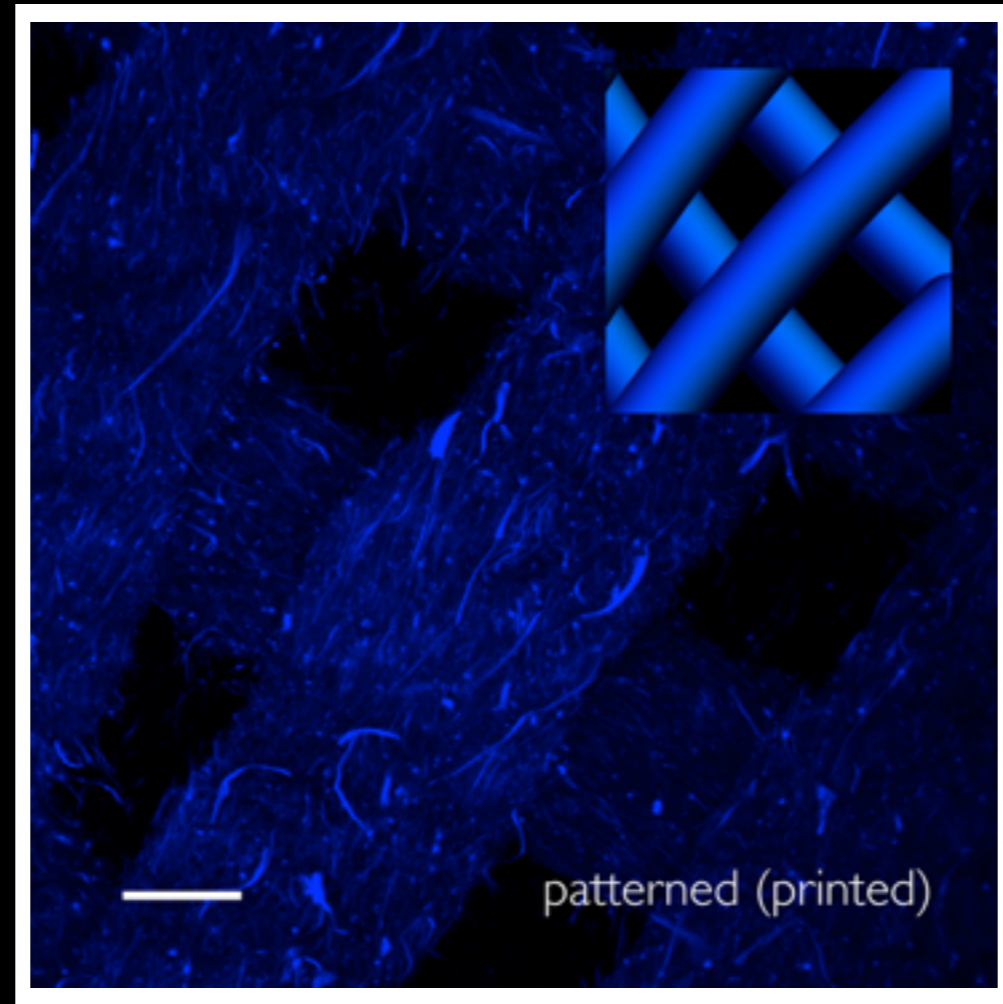
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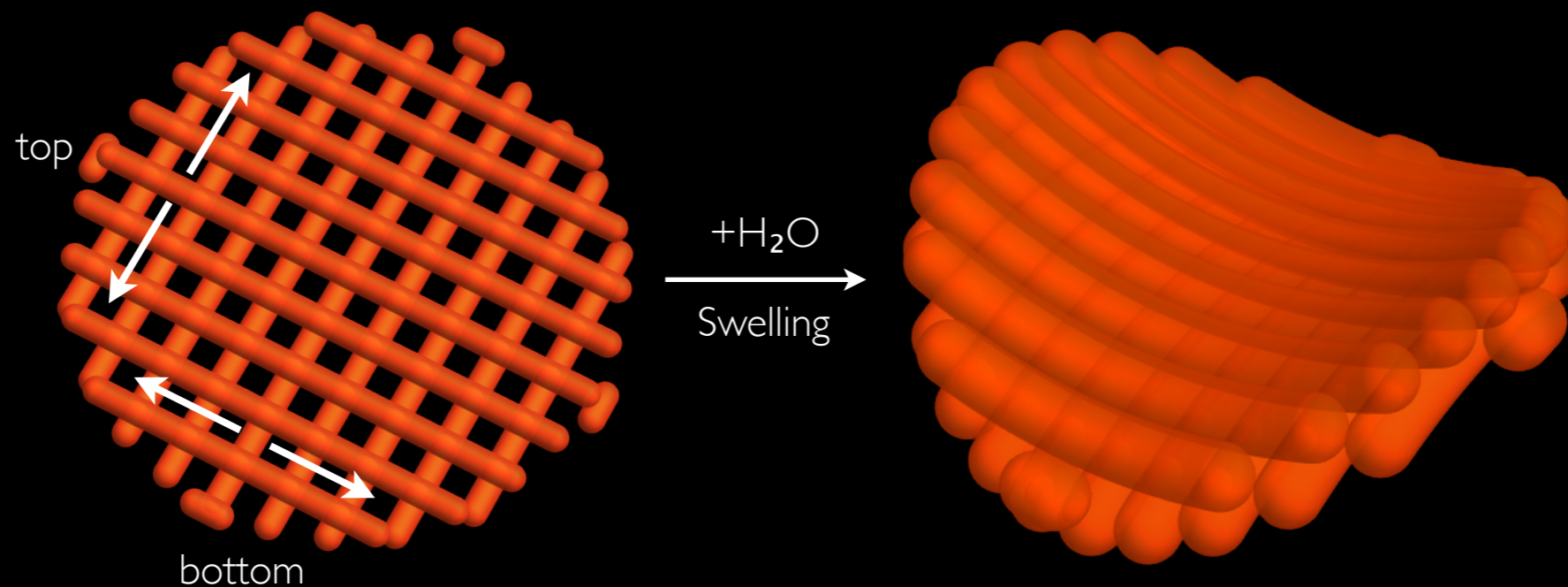
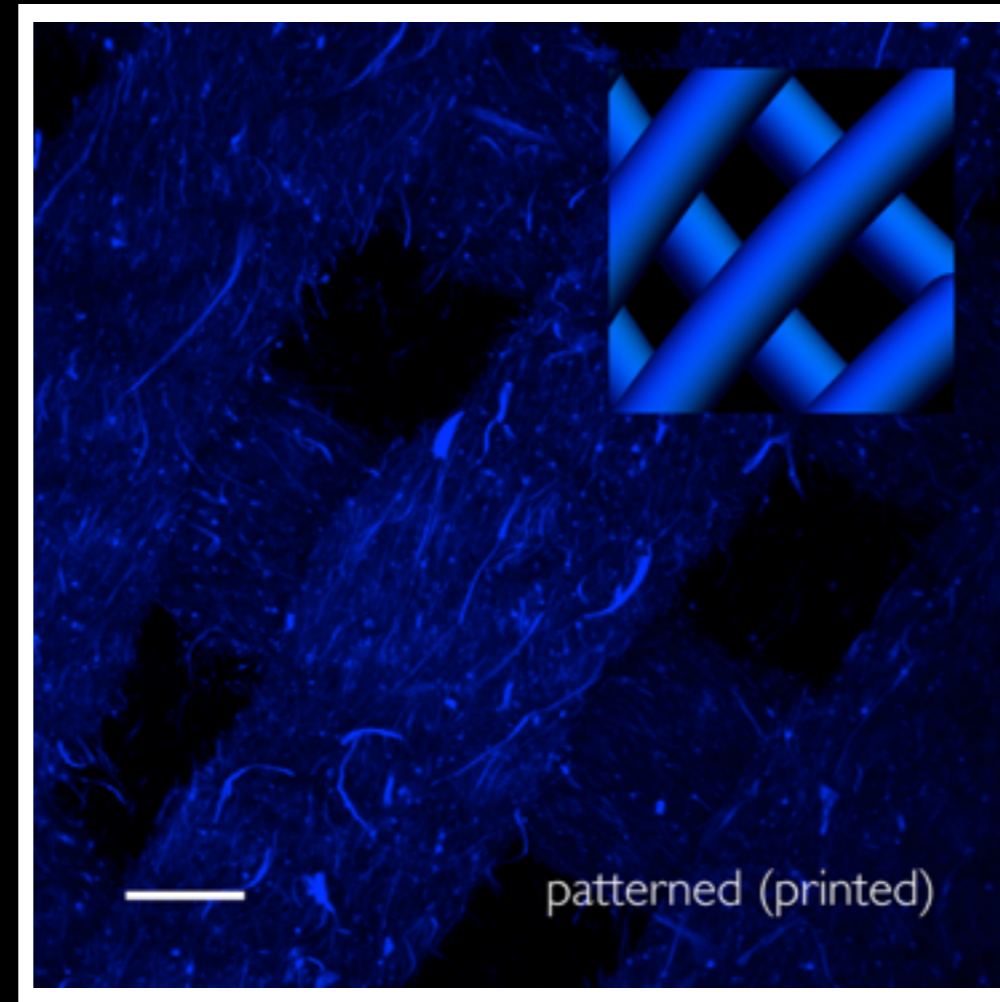
Elastic Anisotropy leads to swelling anisotropy

# Encoding Local Anisotropy





# Encoding Local Anisotropy



# Bi-Metallic Strips

## I. Equilibrium Condition

“Due to the fact that there are not external forces acting on the strip, all forces acting over any cross-section of the strip must be in equilibrium”

Stress from  
fictitious force

$$\sigma = \frac{P^{\text{eff}}}{h}$$

## ANALYSIS OF BI-METAL THERMOSTATS

By S. TIMOSHENKO

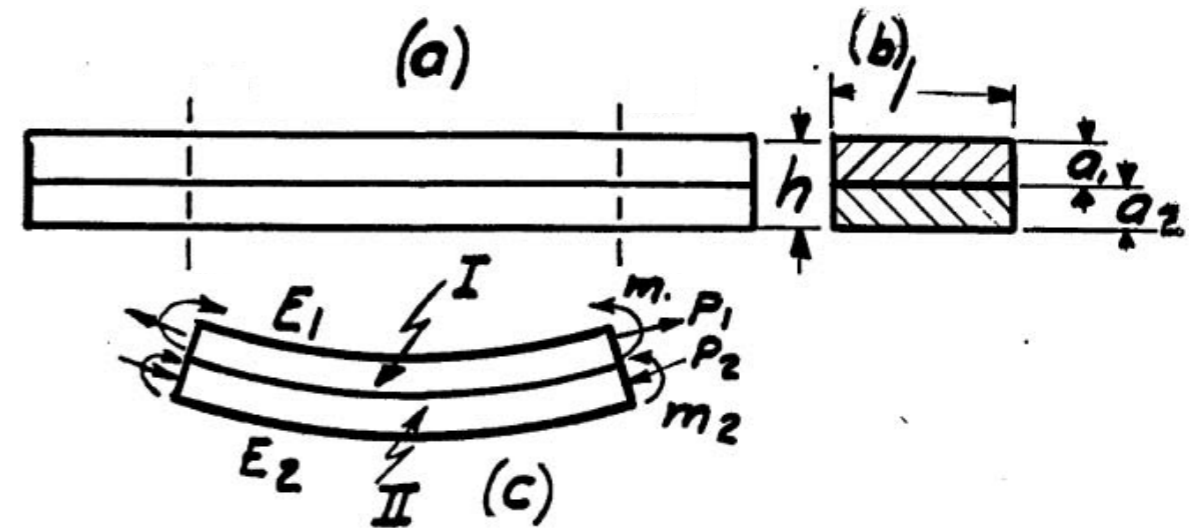


FIG. 1. Deflection of a bi-metal strip while uniformly heated.

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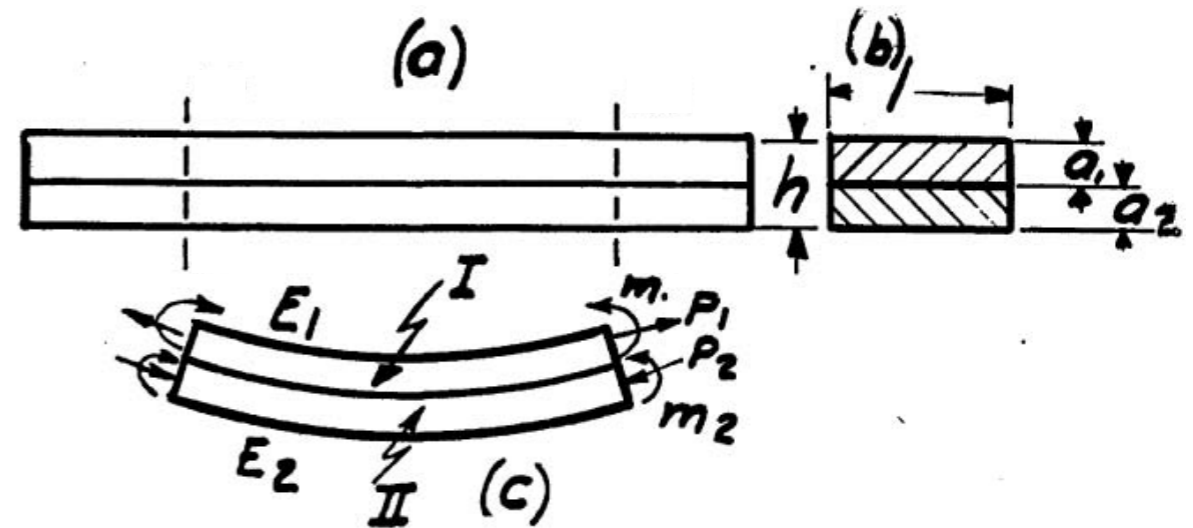


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$$\begin{aligned} M^{\text{tot}} &= \int_{-a_2}^{a_1} zE(z)\varepsilon dz \\ &= \kappa E_1 \int_0^{a_1} z^2 dz + \kappa E_2 \int_{-a_2}^0 z^2 dz \\ &= \frac{\kappa}{3} (E_1 a_1^3 + E_2 a_2^3) \end{aligned}$$

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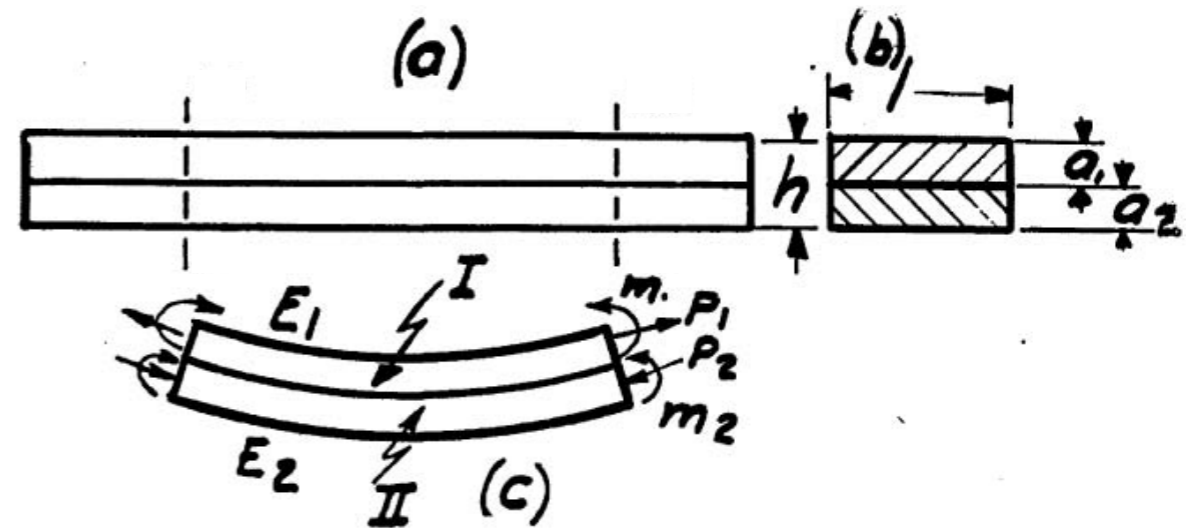


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Moment-stress relationship

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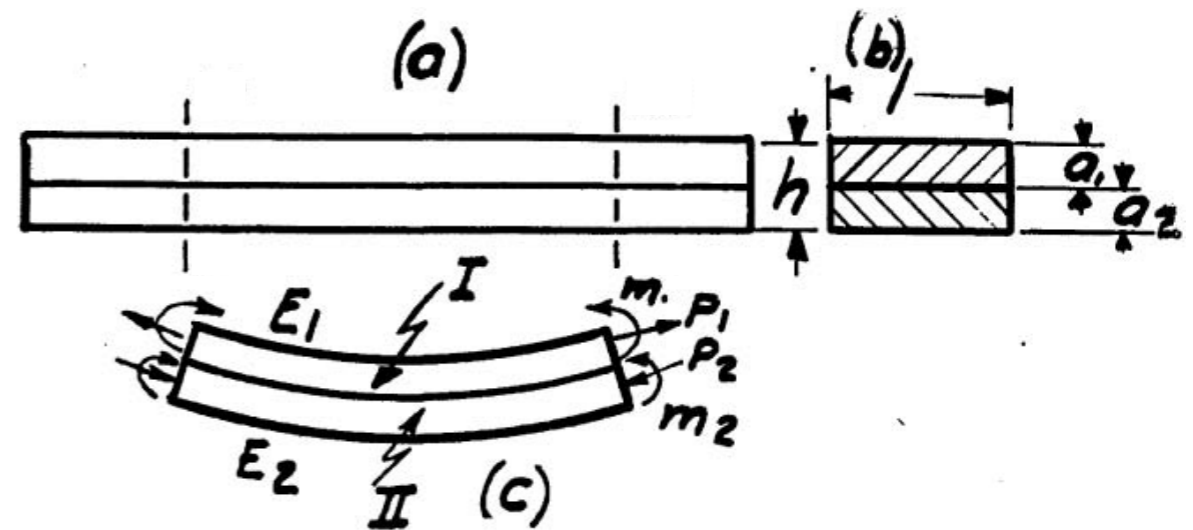


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# Bi-Metallic Strips

## 2. Compatibility Condition

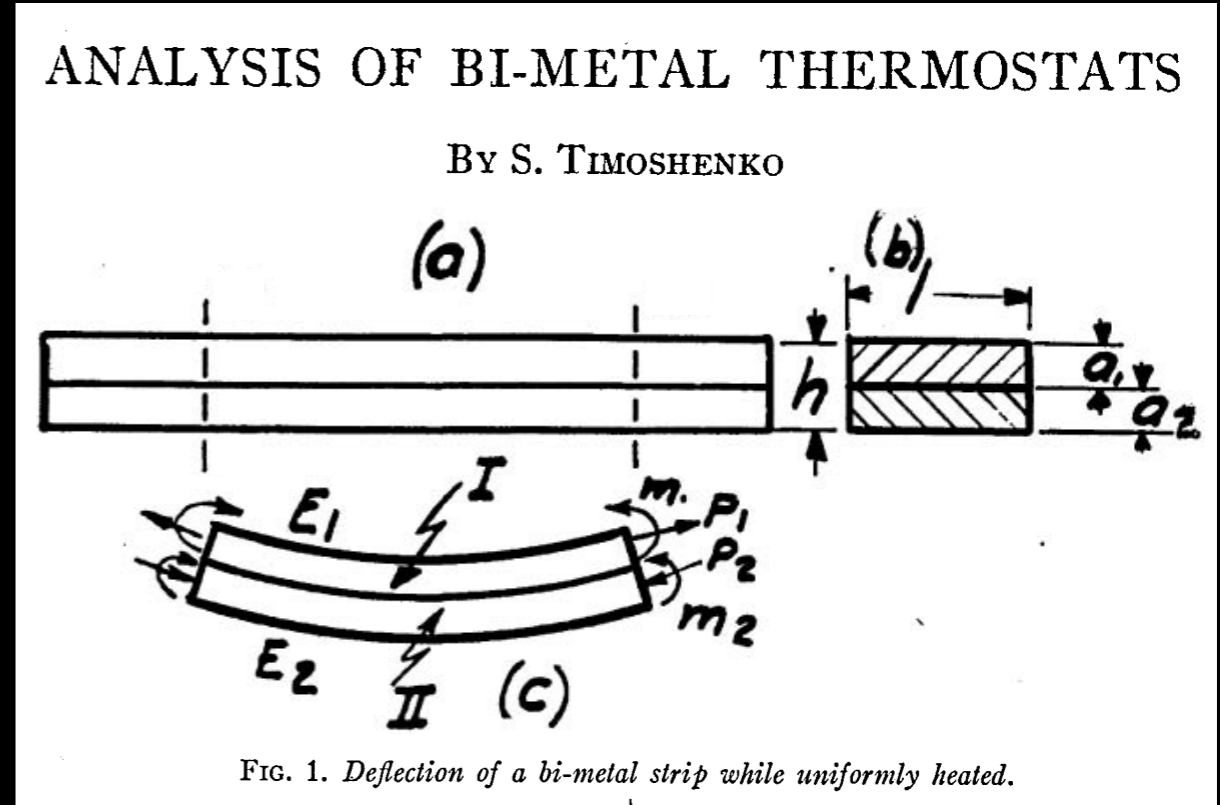
“On the bearing surface of both metals the unit elongation occurring in the longitudinal fibres of metals (1) and (2) must be equal.”

$$\varepsilon^{(1)} = \varepsilon^{(2)}$$

Strain from swelling  $\varepsilon^s = \alpha$

Strain from curvature  $\varepsilon^s = z\kappa$

Strain from stress  $\varepsilon = E^{-1}\sigma^{\text{tot}} = \frac{1}{E} \frac{M^{\text{tot}}}{2h}$



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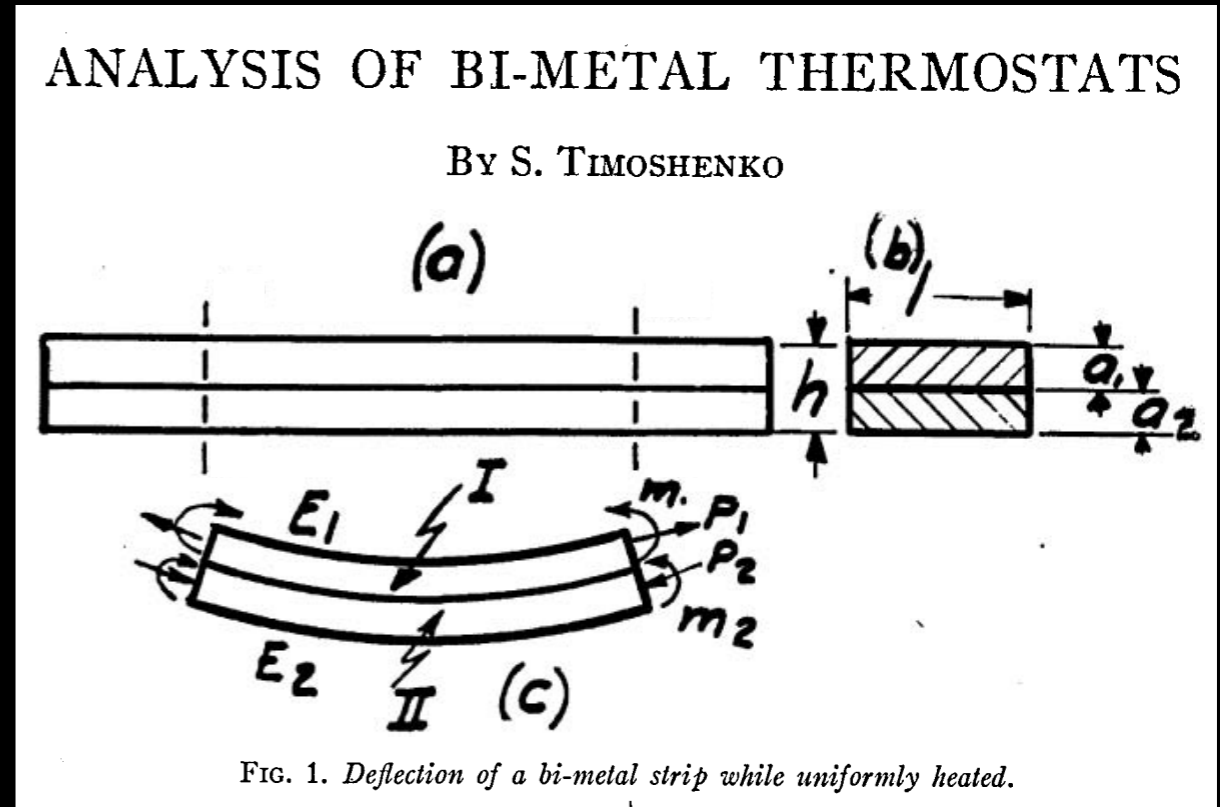
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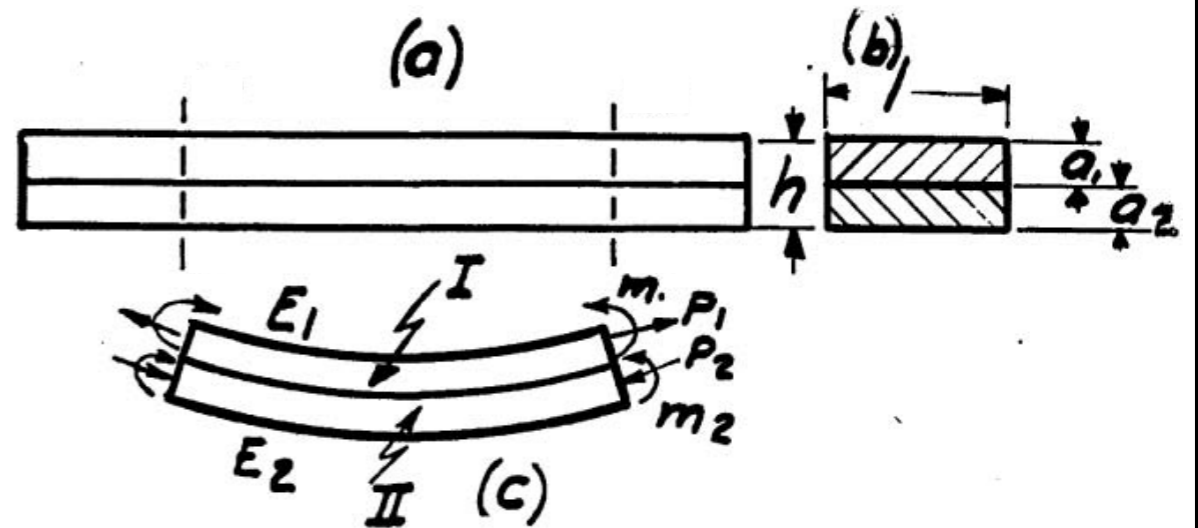


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$$\kappa = \frac{6(\alpha_2 - \alpha_1)(1 + m)^2}{h \left( 3(1 + m)^2 + (1 + mn) \left( m^2 + \frac{1}{mn} \right) \right)}, \quad m = \frac{a_1}{a_2}, \quad n = \frac{E_1}{E_2}$$

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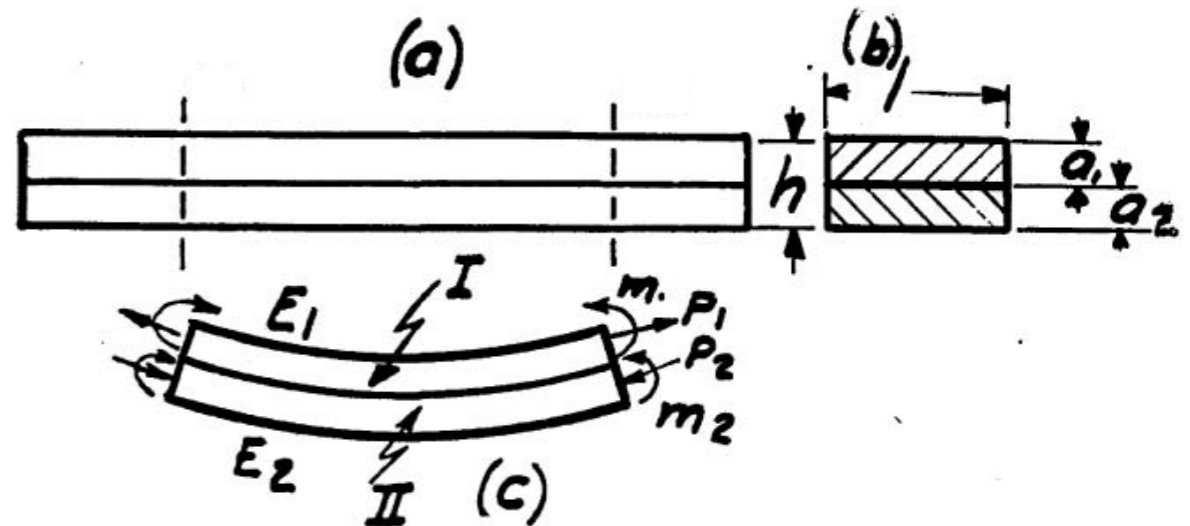
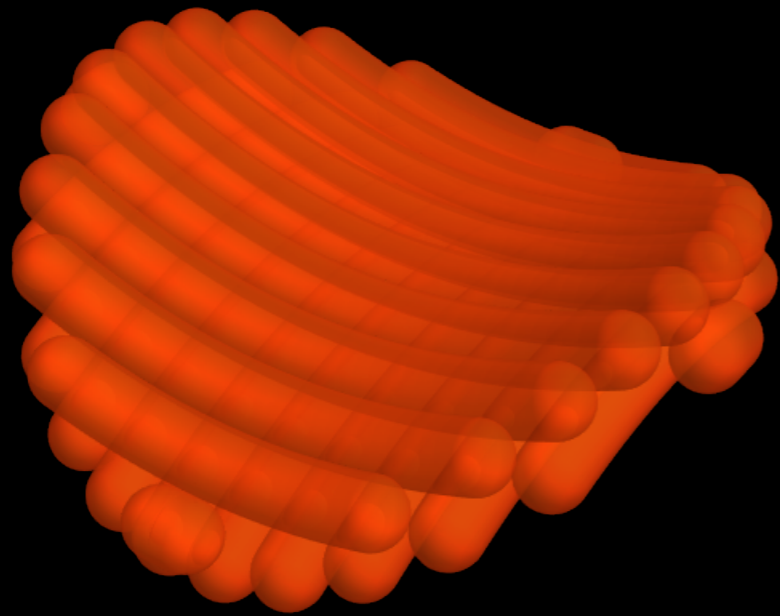
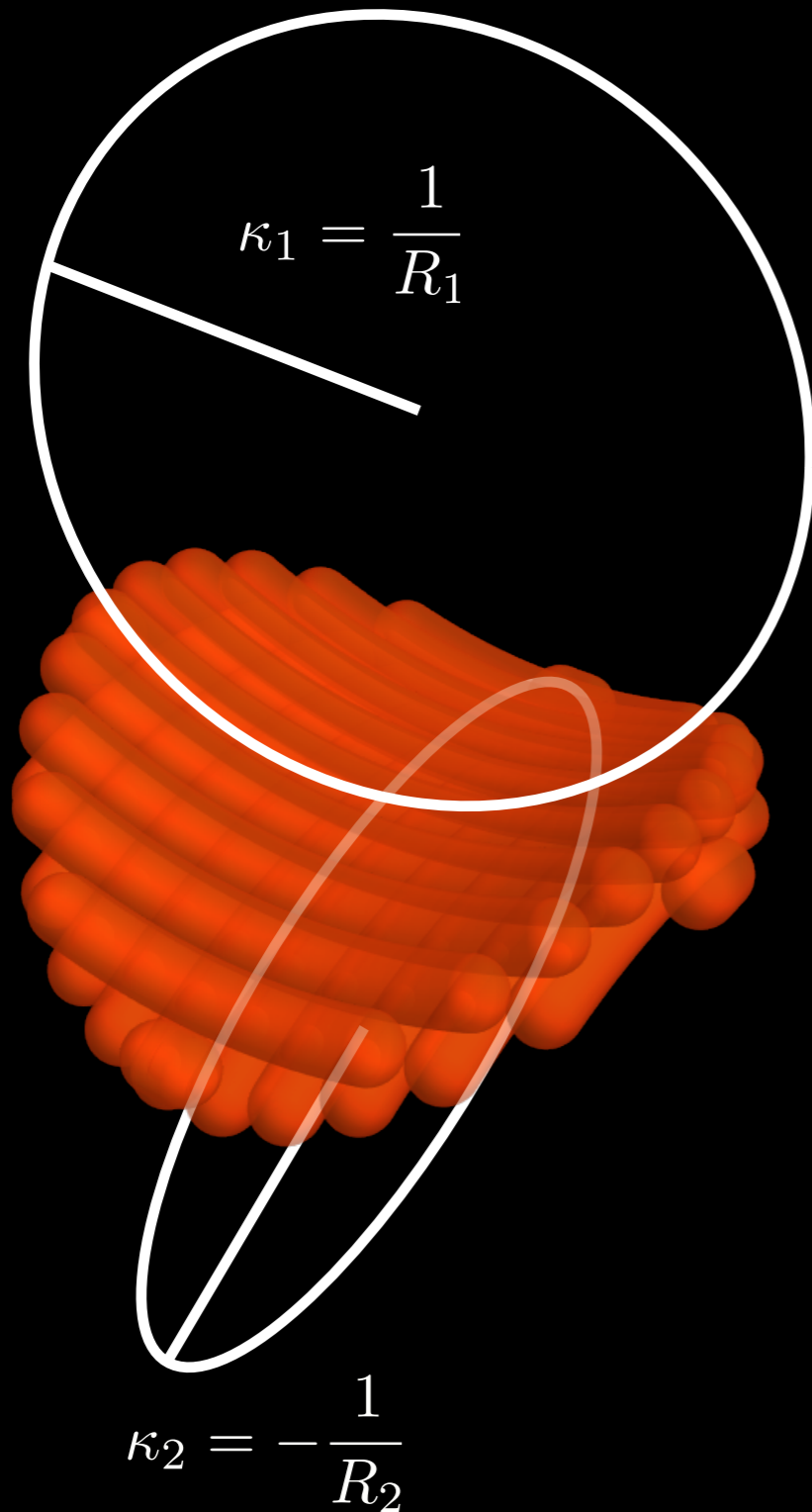


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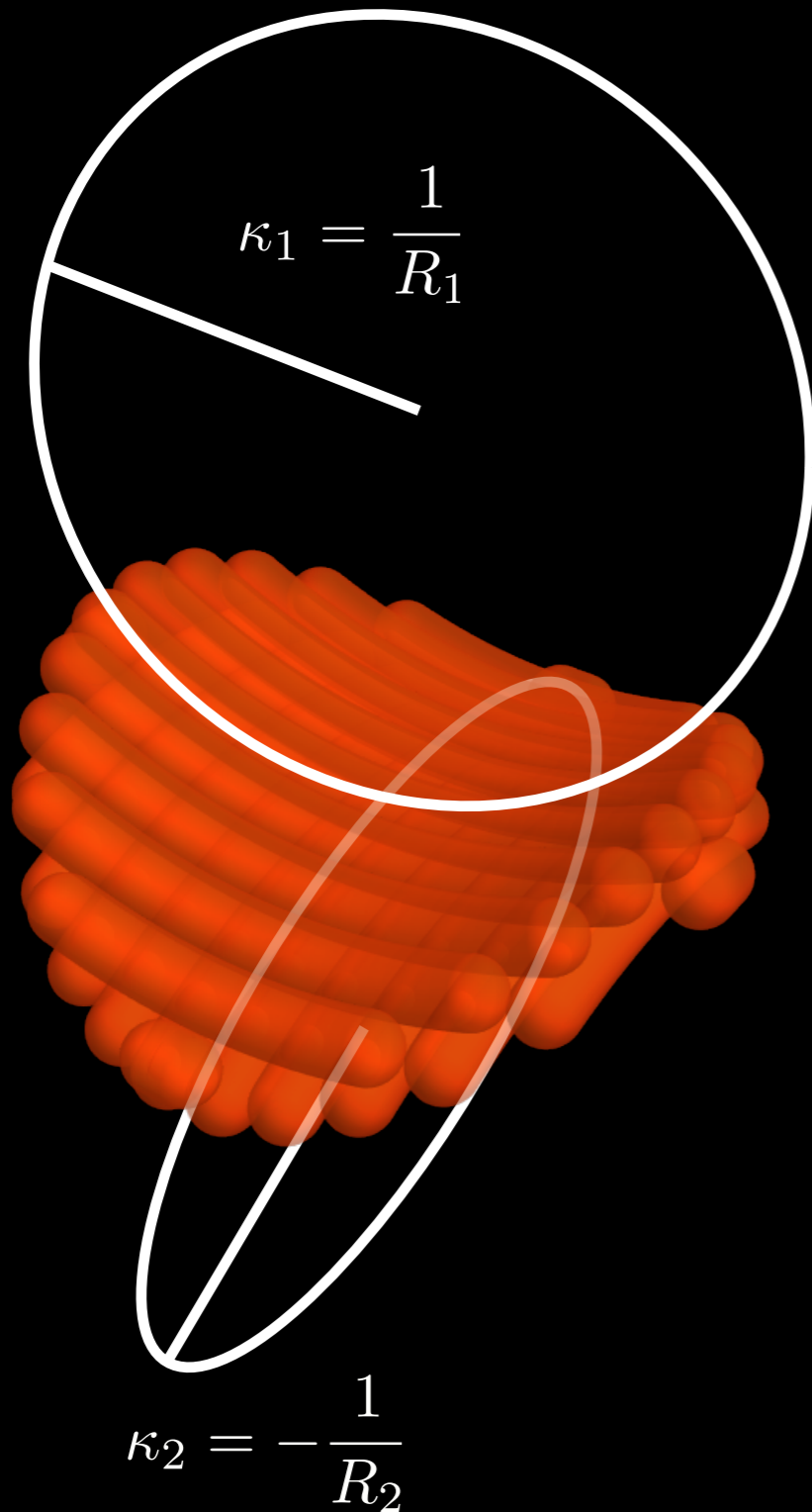
# A Brief Primer on Curvature



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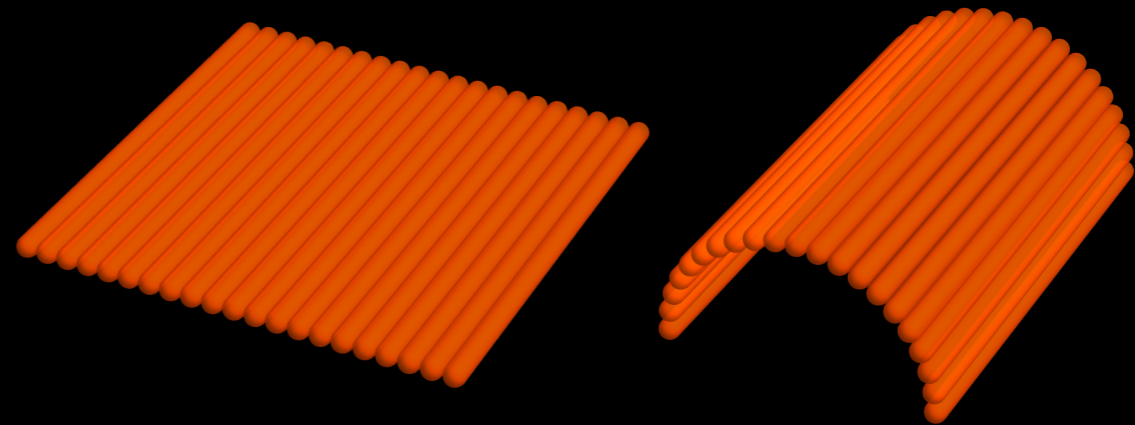


Mean (Extrinsic) Curvature:  $H = \frac{1}{2}(\kappa_1 + \kappa_2)$

Bending energy

$H=0$

$H<0$

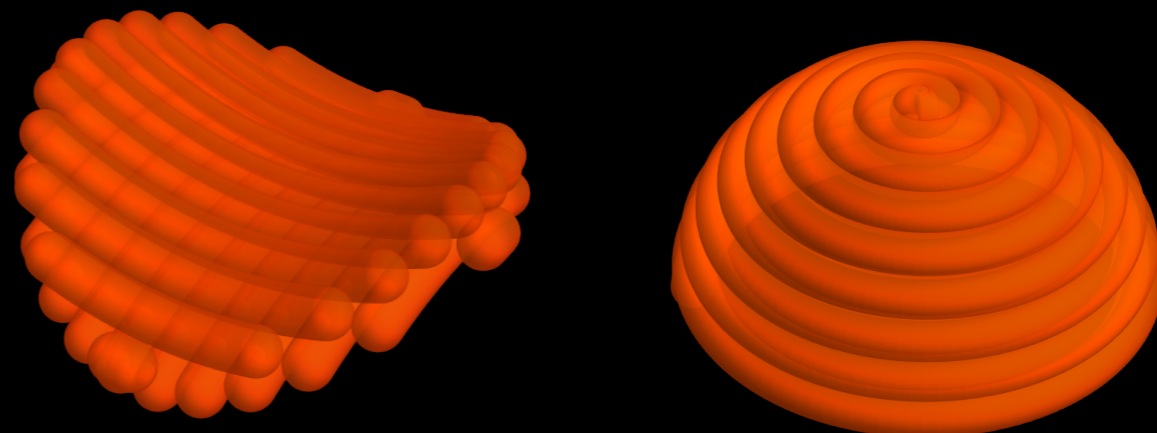


Gaussian (Intrinsic) Curvature:  $K = \kappa_1 \kappa_2$

Stretching energy

$K<0$

$K>0$

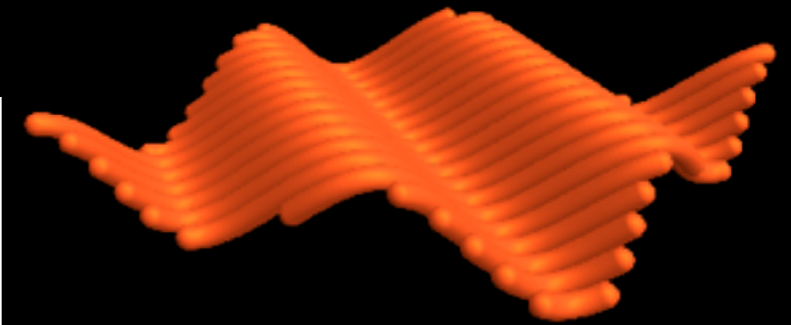
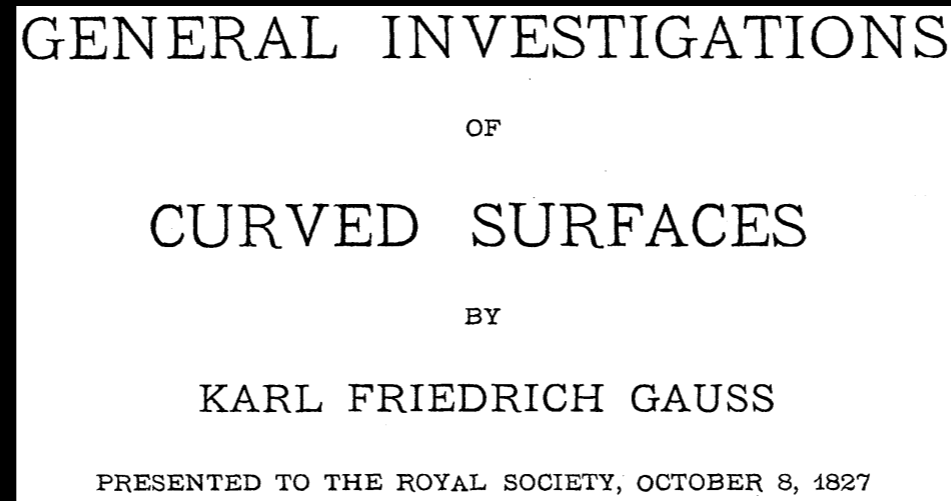


# A Geometric Model

$$g(x, y) = R[\theta(x, y)] \begin{bmatrix} \alpha_{\parallel} & 0 \\ 0 & \alpha_{\perp} \end{bmatrix} R^T[\theta(x, y)]$$



## Gauss's Theorema Egregium



Thus the formula of the preceding article leads of itself to the remarkable  
**THEOREM.** *If a curved surface is developed upon any other surface whatever, the measure of curvature in each point remains unchanged.*

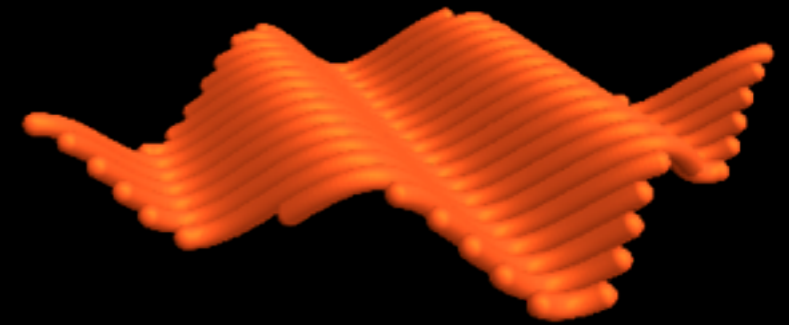
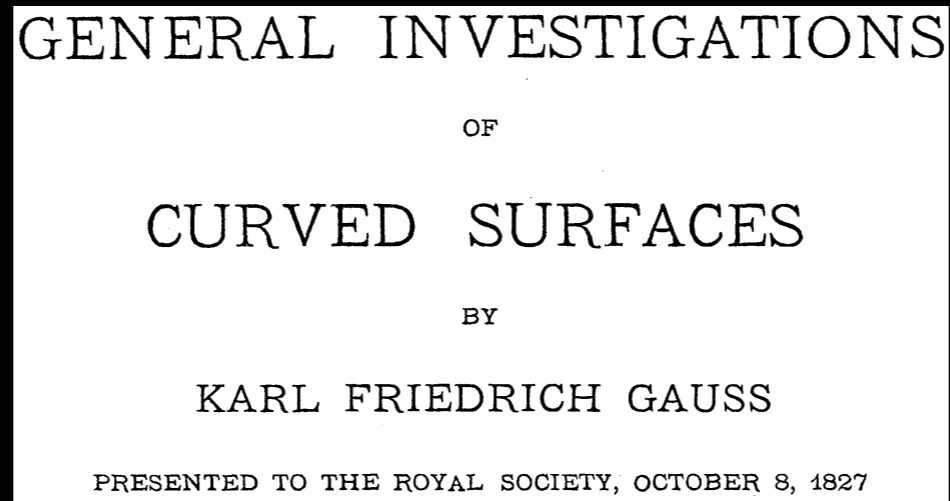
$$K(x, y) = K(g, \partial_x g, \partial_y g, \partial_{xx} g, \partial_{xy} g, \partial_{yy} g)$$

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$$K(x, y) = (\alpha_{\parallel} - \alpha_{\perp}) \left[ \frac{(\phi^2 - 1)\phi_{xy} - \phi(\phi_{xx} - \phi_{yy})}{(\phi^2 + 1)^2} + \frac{(3\phi^2 - 1)(\phi_x^2 - \phi_y^2) - 2\phi(\phi^2 - 3)\phi_x\phi_y}{(\phi^2 + 1)^3} \right]$$

# The Model

## Theory of Anisotropic Plates and Shells

Curvature in Monge Gauge  $\kappa_{ij} = \partial_i \partial_j H(x, y)$

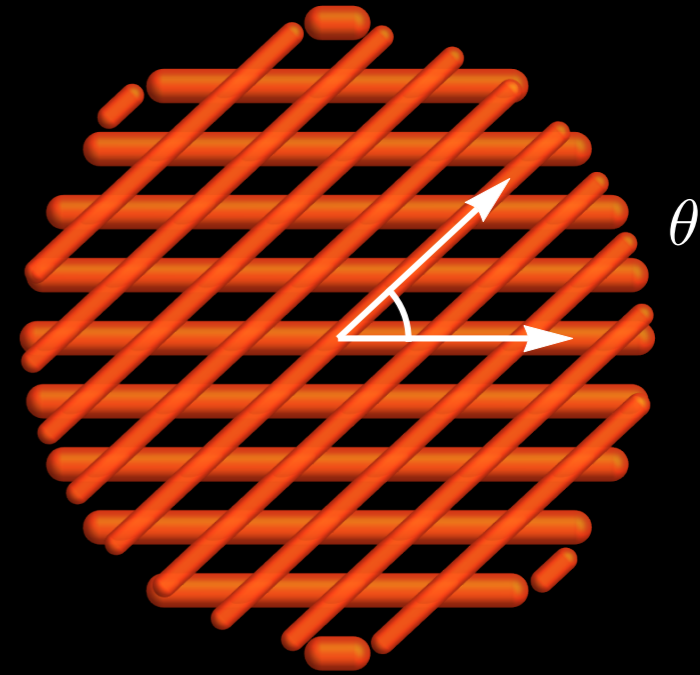
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Elastic Strain  $\varepsilon_{ij}^e = -z \kappa_{ij}$

Strain Tensor  $\varepsilon = \varepsilon^s + \varepsilon^e \quad \varepsilon_{ij}(\theta) = R_{im}(\theta) \varepsilon_{mn} R_{jn}^T(\theta)$

Elastic Modulus Tensor  $\mathbf{E}_{ijkl}(\theta) = R_{im}(\theta) R_{kp}(\theta) \mathbf{E}_{mnpq} R_{jn}^T(\theta) R_{lq}^T(\theta)$

Stress-Strain Relation  $\sigma_{ij} = \mathbf{E}_{ijkl} \varepsilon_{kl}^e$



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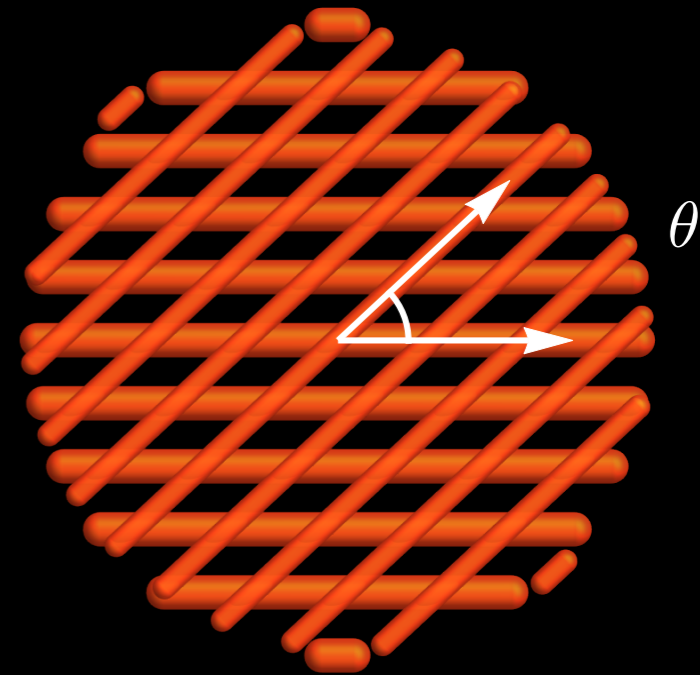
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Bending Moments 
$$M_{ij} = \int_{-a_2}^{a_1} z\sigma_{ij}dz = - \int_{-a_2}^{a_1} z^2\mathbf{E}_{ijkl}\kappa_{kl}dz$$

$$= - \int_0^{a_1} \mathbf{E}_{ijkl}(0)\kappa_{kl}z^2dz - \int_{-a_2}^0 \mathbf{E}_{ijkl}(\theta)\kappa_{kl}z^2dz$$



### 1. Equilibrium Condition

“Due to the fact that there are not external forces acting on the strip, all forces acting over any cross-section of the strip must be in equilibrium”

### 2. Compatibility Condition

“On the bearing surface of both metals the unit elongation occurring in the longitudinal fibres of metals (1) and (2) must be equal.”



# The Model

## Theory of Anisotropic Plates and Shells

Curvature in Monge Gauge  $\kappa_{ij} = \partial_i \partial_j H(x, y)$

Swelling Strain  $\varepsilon^s = \begin{bmatrix} \alpha_{\parallel} & 0 \\ 0 & \alpha_{\perp} \end{bmatrix}$

Elastic Strain  $\varepsilon_{ij}^e = -z\kappa_{ij}$

Strain Tensor  $\varepsilon = \varepsilon^s + \varepsilon^e \quad \varepsilon_{ij}(\theta) = R_{im}(\theta)\varepsilon_{mn}R_{jn}^T(\theta)$

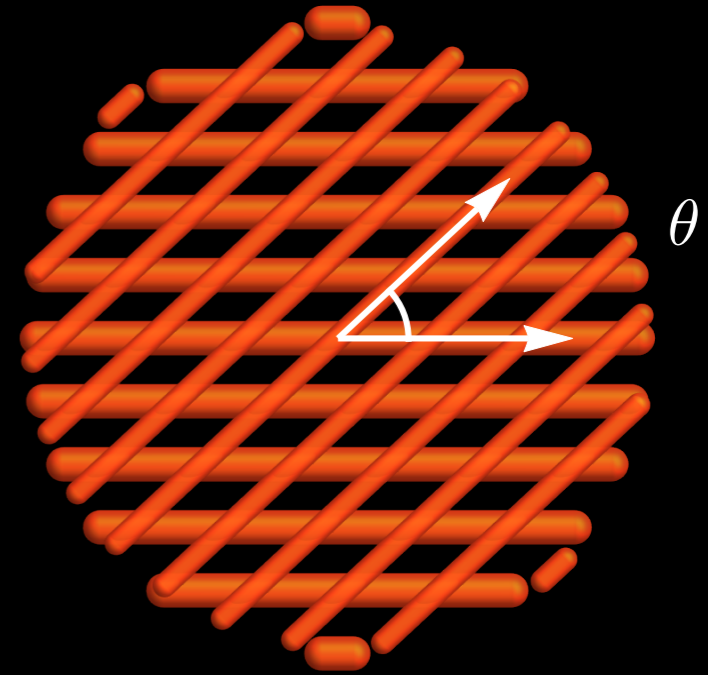
Elastic Modulus Tensor  $\mathbf{E}_{ijkl}(\theta) = R_{im}(\theta)R_{kp}(\theta)\mathbf{E}_{mnpq}R_{jn}^T(\theta)R_{lq}^T(\theta)$

Stress-Strain Relation  $\sigma_{ij} = \mathbf{E}_{ijkl}\varepsilon_{kl}^e$

Bending Moments 
$$M_{ij} = \int_{-a_2}^{a_1} z\sigma_{ij}dz = - \int_{-a_2}^{a_1} z^2\mathbf{E}_{ijkl}\kappa_{kl}dz$$

$$= - \int_0^{a_1} \mathbf{E}_{ijkl}(0)\kappa_{kl}z^2dz - \int_{-a_2}^0 \mathbf{E}_{ijkl}(\theta)\kappa_{kl}z^2dz$$

$$\frac{1}{\alpha_1} \int_0^{a_1} \left( \varepsilon_{ij}^{(1)} + \frac{\mathbf{E}_{ijkl}^{-1}}{a_1} M_{kl}(\theta) \right) dz = \frac{1}{\alpha_2} \int_{-a_2}^0 \left( \varepsilon_{ij}^{(2)}(\theta) + \frac{\mathbf{E}_{ijkl}^{-1}(\theta)}{a_2} M_{kl}(\theta) \right) dz$$



# The Model

$$\frac{1}{\alpha_1} \int_0^{a_1} \left( \varepsilon_{ij}^{(1)} + \frac{\mathbf{E}_{ijkl}^{-1}}{a_1} M_{kl}(\theta) \right) dz = \frac{1}{\alpha_2} \int_{-a_2}^0 \left( \varepsilon_{ij}^{(2)}(\theta) + \frac{\mathbf{E}_{ijkl}^{-1}(\theta)}{a_2} M_{kl}(\theta) \right) dz$$

Given:  $\alpha_{\parallel}, \alpha_{\perp}, \mathbf{E}_{ijkl}, a_1, a_2, \theta$       Solve for:  $\kappa_{ij}$

# The Model

$$\frac{1}{\alpha_1} \int_0^{a_1} \left( \varepsilon_{ij}^{(1)} + \frac{\mathbf{E}_{ijkl}^{-1}}{a_1} M_{kl}(\theta) \right) dz = \frac{1}{\alpha_2} \int_{-a_2}^0 \left( \varepsilon_{ij}^{(2)}(\theta) + \frac{\mathbf{E}_{ijkl}^{-1}(\theta)}{a_2} M_{kl}(\theta) \right) dz$$

Given:  $\alpha_{\parallel}, \alpha_{\perp}, \mathbf{E}_{ijkl}, a_1, a_2, \theta$       Solve for:  $\kappa_{ij}$

$$H = \frac{\alpha_{\perp} - \alpha_{\parallel}}{h} \frac{c_1 \sin^2(\theta)}{c_2 - c_3 \cos(2\theta) + m^4 \cos(4\theta)}, \quad c_i = c_i(\mathbf{E}^{(1)}, \mathbf{E}^{(2)}, m = a_1/a_2)$$

$$K = -\frac{(\alpha_{\perp} - \alpha_{\parallel})^2}{h^2} \frac{c_4 \sin^2(\theta)}{c_5 - c_6 \cos(2\theta) + m^4 \cos(4\theta)}$$

# The Model

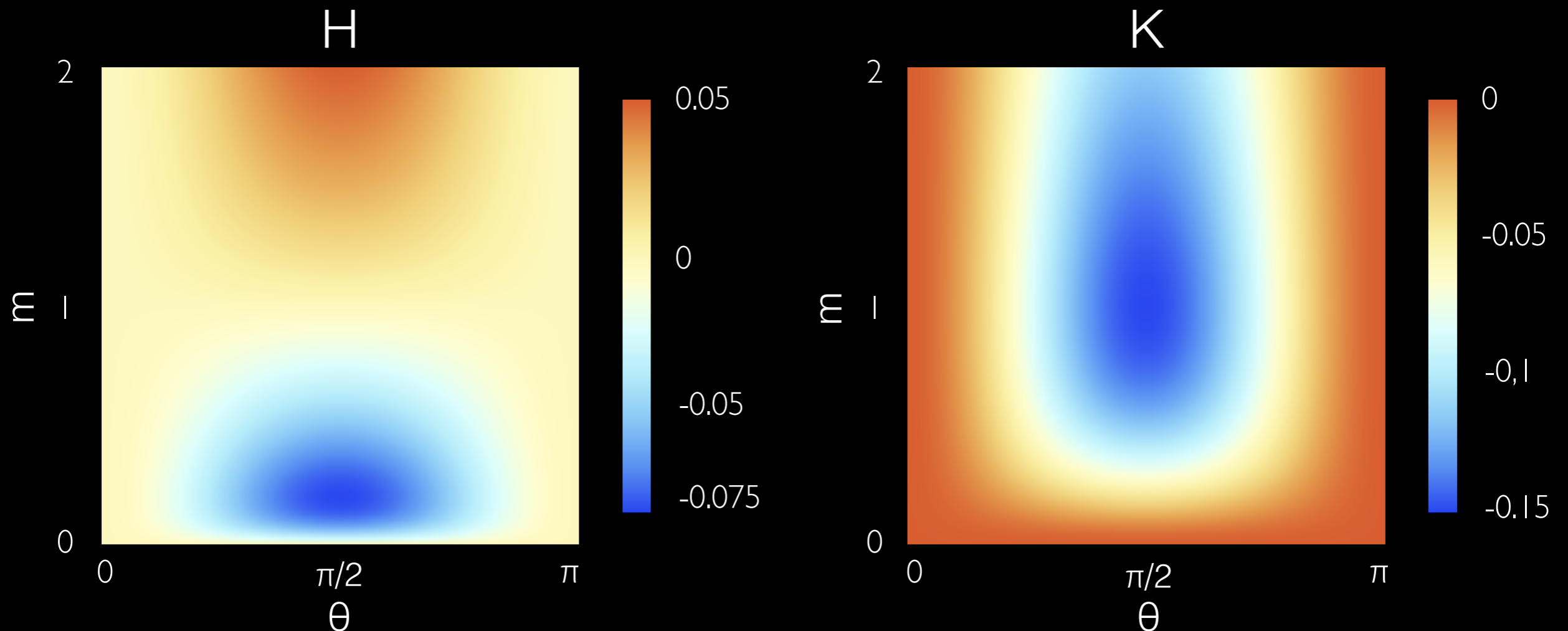
$$\frac{1}{\alpha_1} \int_0^{a_1} \left( \varepsilon_{ij}^{(1)} + \frac{\mathbf{E}_{ijkl}^{-1}}{a_1} M_{kl}(\theta) \right) dz = \frac{1}{\alpha_2} \int_{-a_2}^0 \left( \varepsilon_{ij}^{(2)}(\theta) + \frac{\mathbf{E}_{ijkl}^{-1}(\theta)}{a_2} M_{kl}(\theta) \right) dz$$

Given:  $\alpha_{\parallel}, \alpha_{\perp}, \mathbf{E}_{ijkl}, a_1, a_2, \theta$       Solve for:  $\kappa_{ij}$

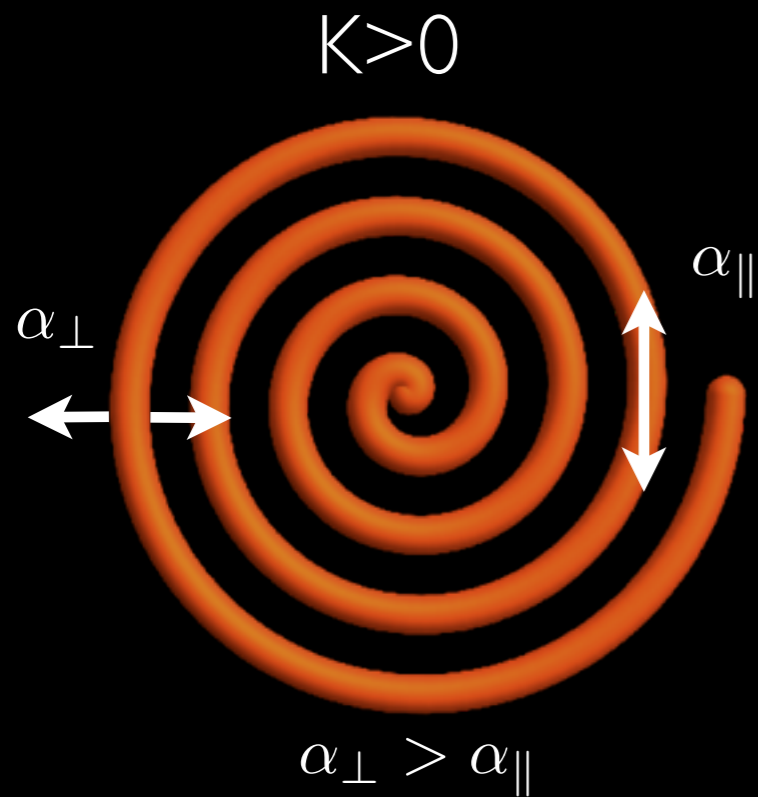
$$H = \frac{\alpha_{\perp} - \alpha_{\parallel}}{h} \frac{c_1 \sin^2(\theta)}{c_2 - c_3 \cos(2\theta) + m^4 \cos(4\theta)}$$

,  $c_i = c_i(\mathbf{E}^{(1)}, \mathbf{E}^{(2)}, m = a_1/a_2)$

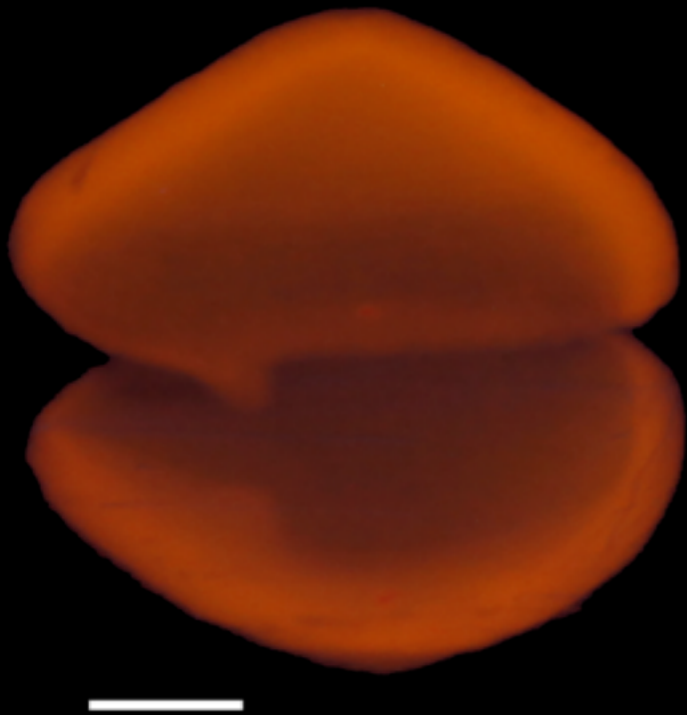
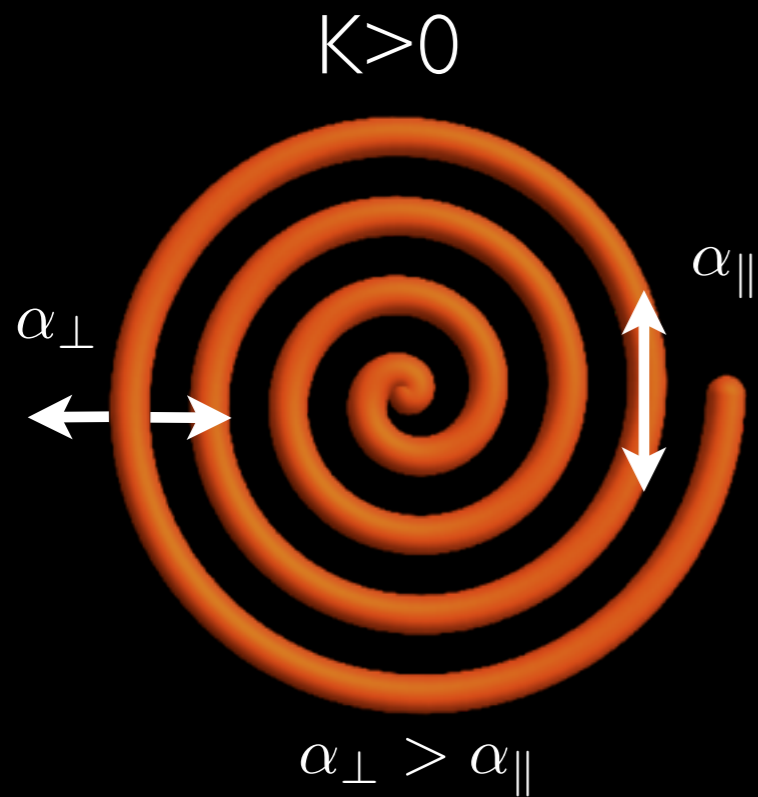
$$K = -\frac{(\alpha_{\perp} - \alpha_{\parallel})^2}{h^2} \frac{c_4 \sin^2(\theta)}{c_5 - c_6 \cos(2\theta) + m^4 \cos(4\theta)}$$



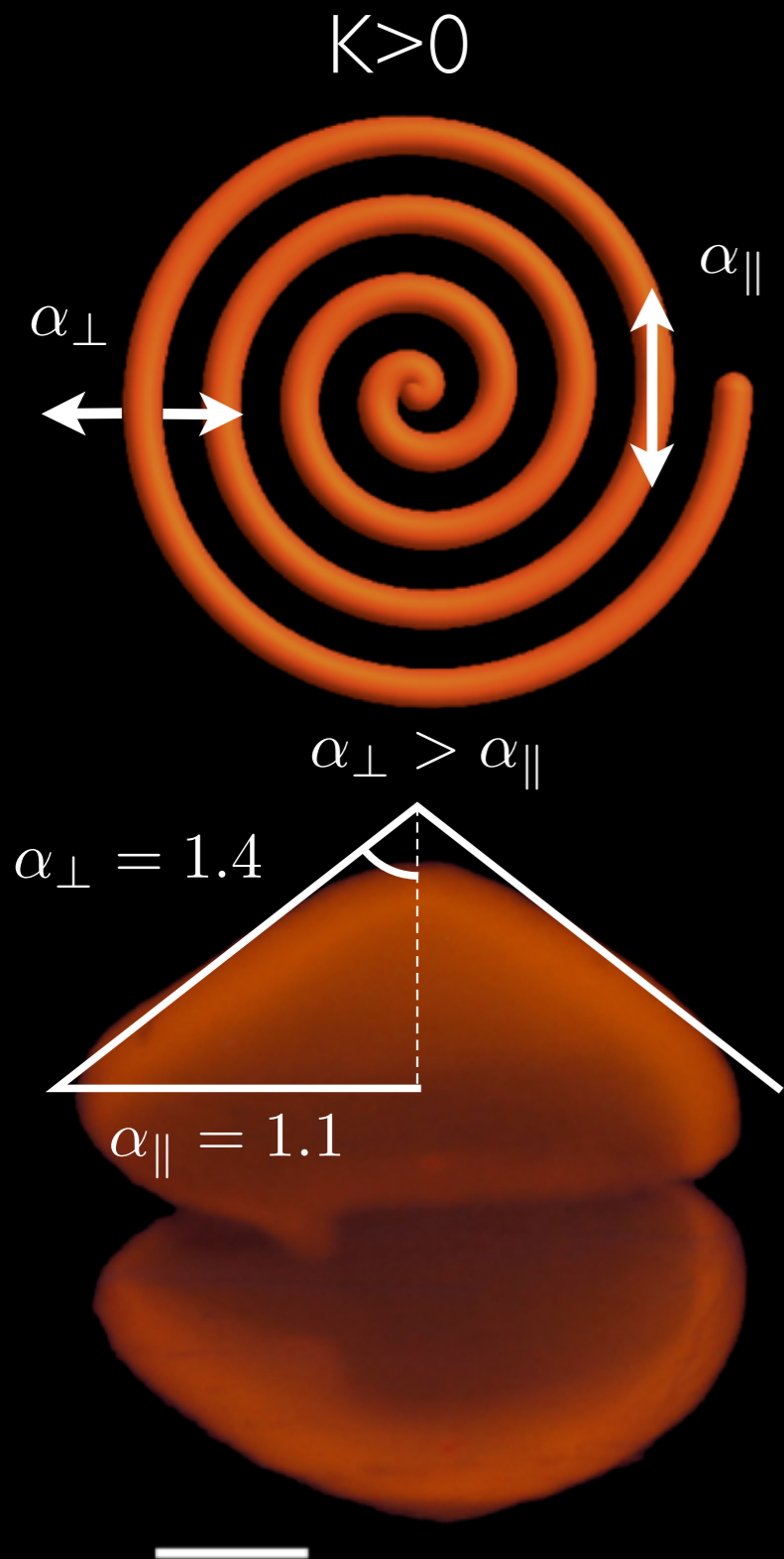
# Controlling Gaussian Curvature



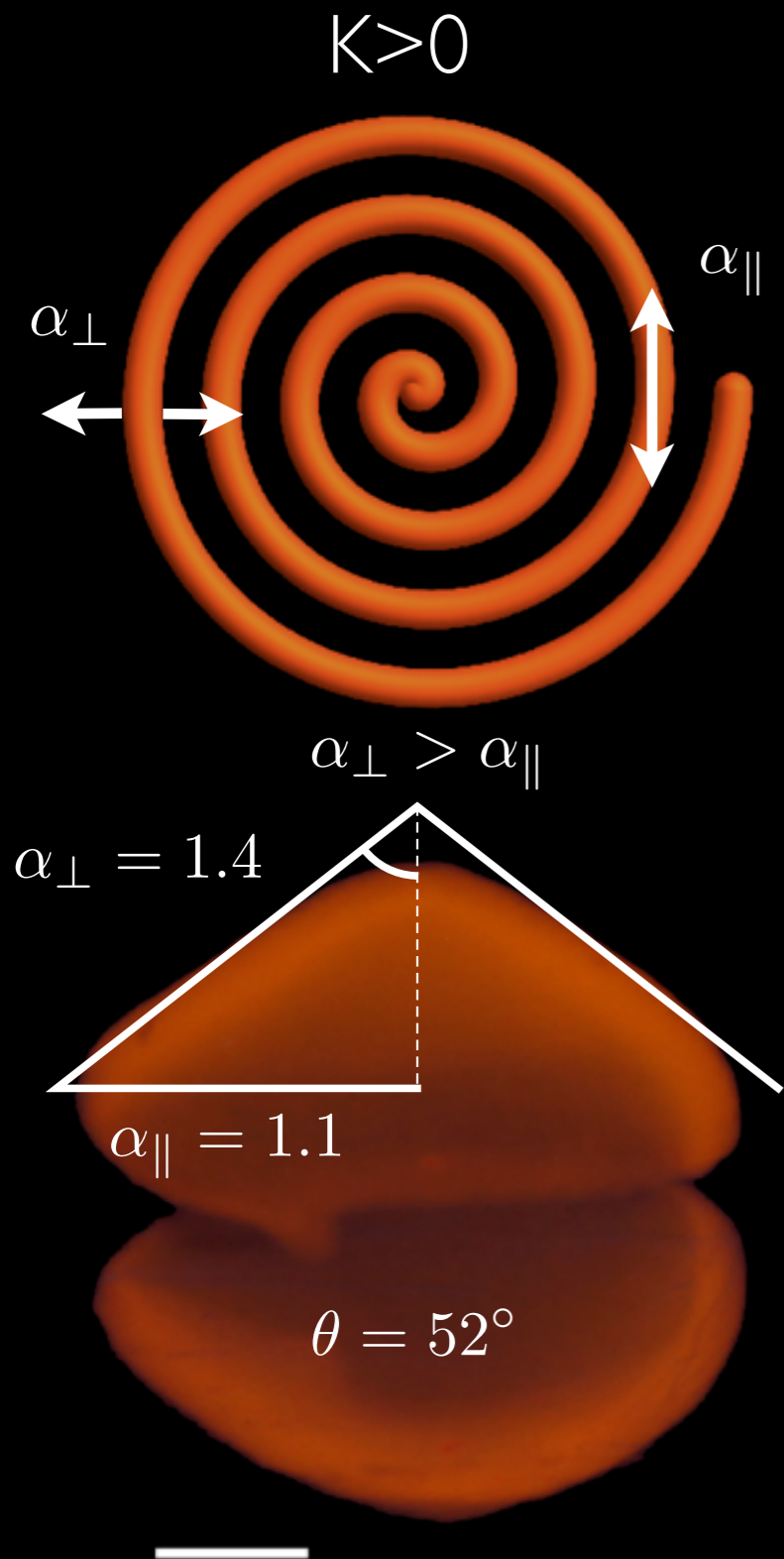
# Controlling Gaussian Curvature



# Controlling Gaussian Curvature

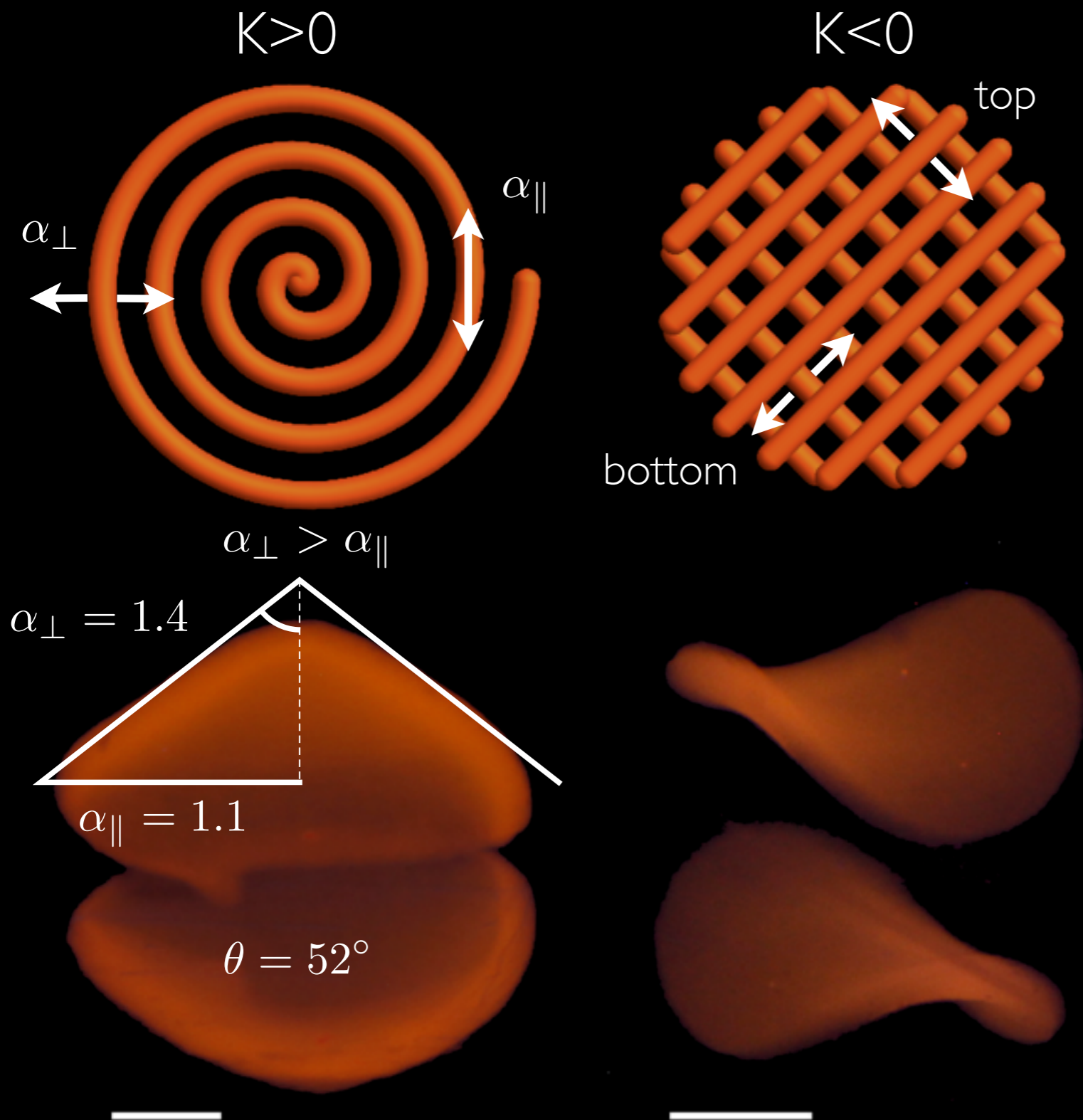


# Controlling Gaussian Curvature

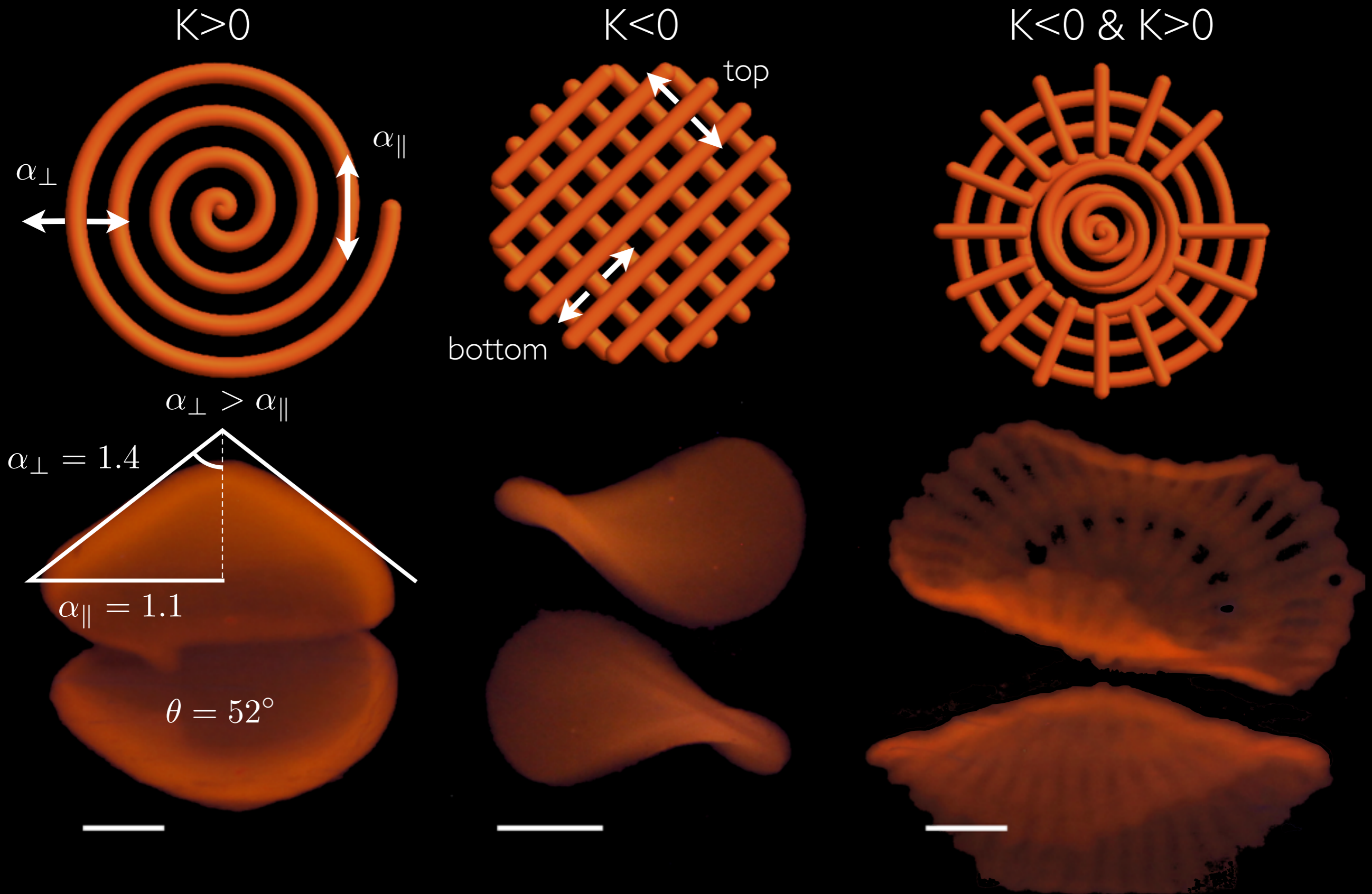




# Controlling Gaussian Curvature



# Controlling Gaussian Curvature



# Controlling Mean Curvature

$$\kappa = 3/2(\alpha_1 - \alpha_2)/h = 0.45/h \text{ mm}^{-1}$$

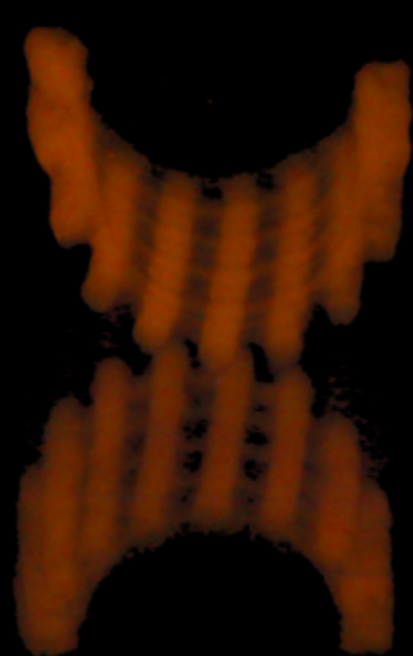
$h = 1.25 \text{ mm}$

$h = 0.75 \text{ mm}$

$h = 0.5 \text{ mm}$



$\kappa = 0.36 \text{ mm}^{-1}$



predicted

$\kappa = 0.6 \text{ mm}^{-1}$



$\kappa = 0.9 \text{ mm}^{-1}$

# Controlling Mean Curvature

$$\kappa = 3/2(\alpha_1 - \alpha_2)/h = 0.45/h \text{ mm}^{-1}$$

$h = 1.25 \text{ mm}$

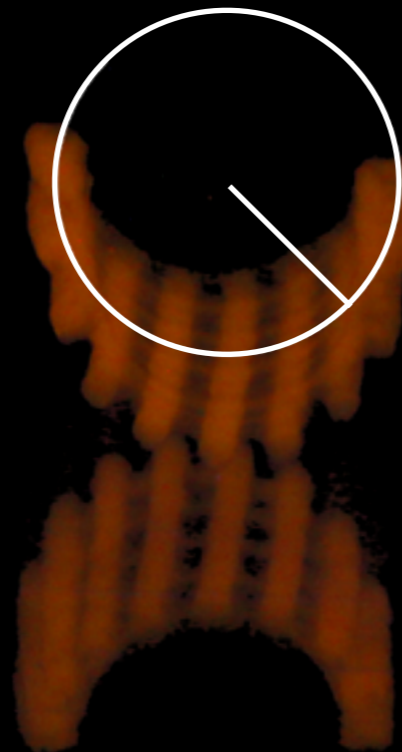
$h = 0.75 \text{ mm}$

$h = 0.5 \text{ mm}$



$\kappa = 0.36 \text{ mm}^{-1}$

$\kappa = 0.34 \text{ mm}^{-1}$



$\kappa = 0.6 \text{ mm}^{-1}$

measured

$\kappa = 0.61 \text{ mm}^{-1}$



$\kappa = 0.85 \text{ mm}^{-1}$

# Controlling Mean Curvature

$$\kappa = 3/2(\alpha_1 - \alpha_2)/h = 0.45/h \text{ mm}^{-1}$$

$h = 1.25 \text{ mm}$

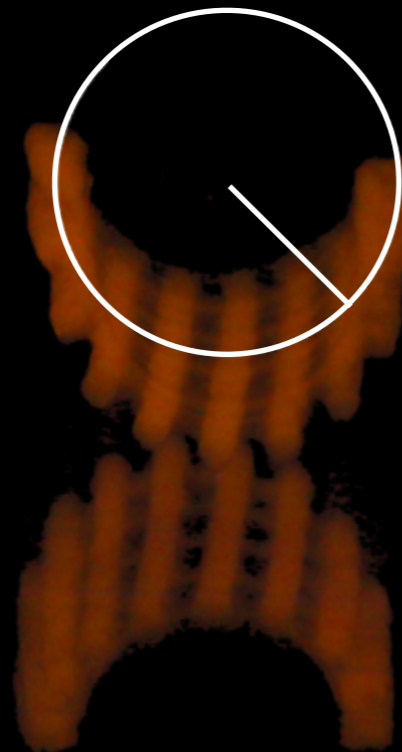
$h = 0.75 \text{ mm}$

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predicted

$\kappa = 0.6 \text{ mm}^{-1}$

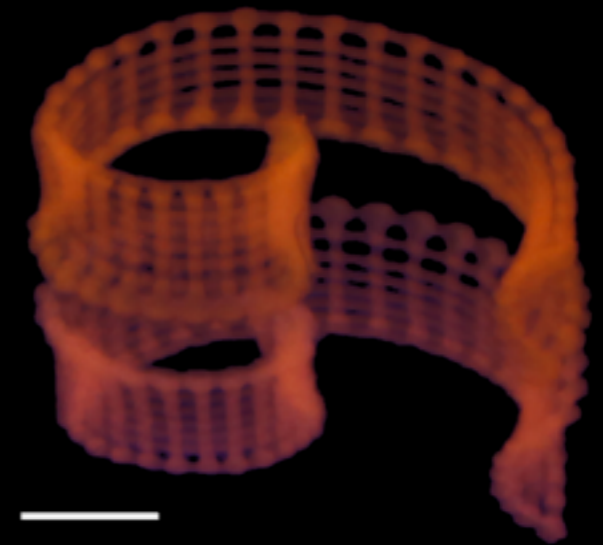
measured

$\kappa = 0.61 \text{ mm}^{-1}$



$\kappa = 0.9 \text{ mm}^{-1}$

$\kappa = 0.85 \text{ mm}^{-1}$



# To Twist or Not To Twist, That is the Question



Bottom Layer:  $0^\circ$   
Top Layer:  $90^\circ$

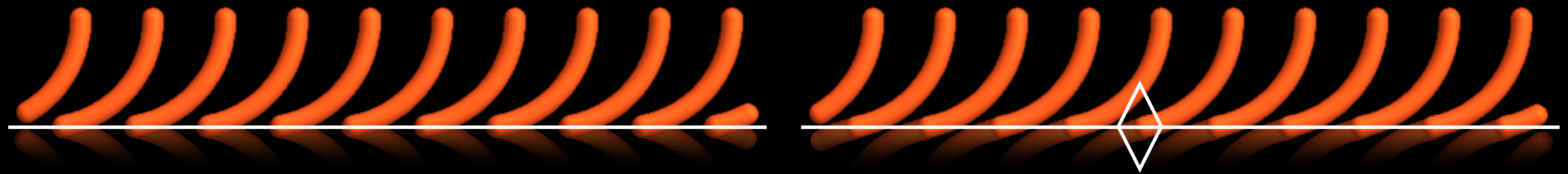


Bottom Layer:  $-45^\circ$   
Top Layer:  $45^\circ$

# To Twist or Not To Twist, That is the Question



# To Twist or Not To Twist, That is the Question

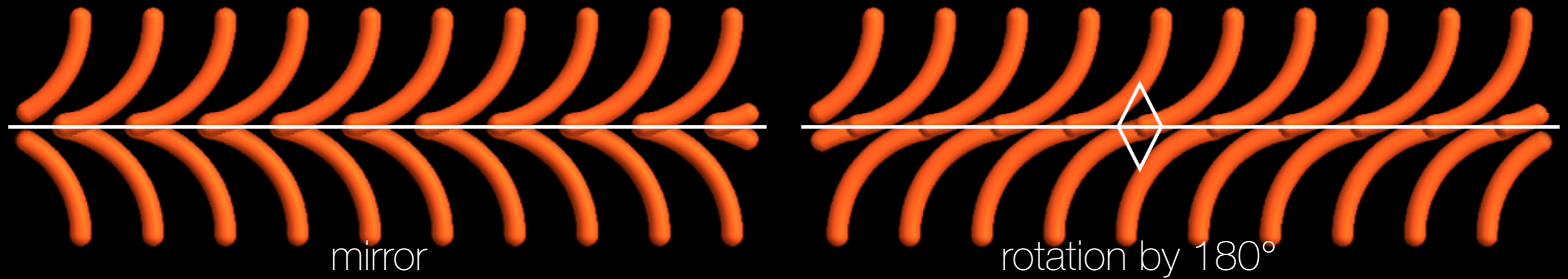


mirror

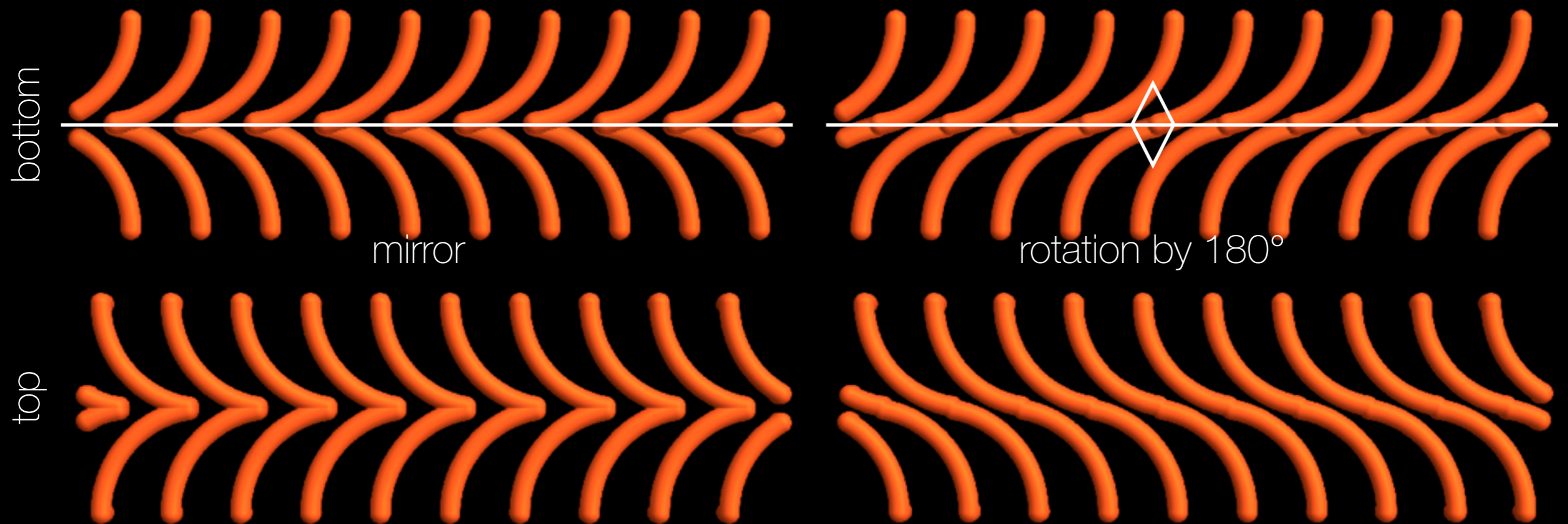
rotation by  $180^\circ$



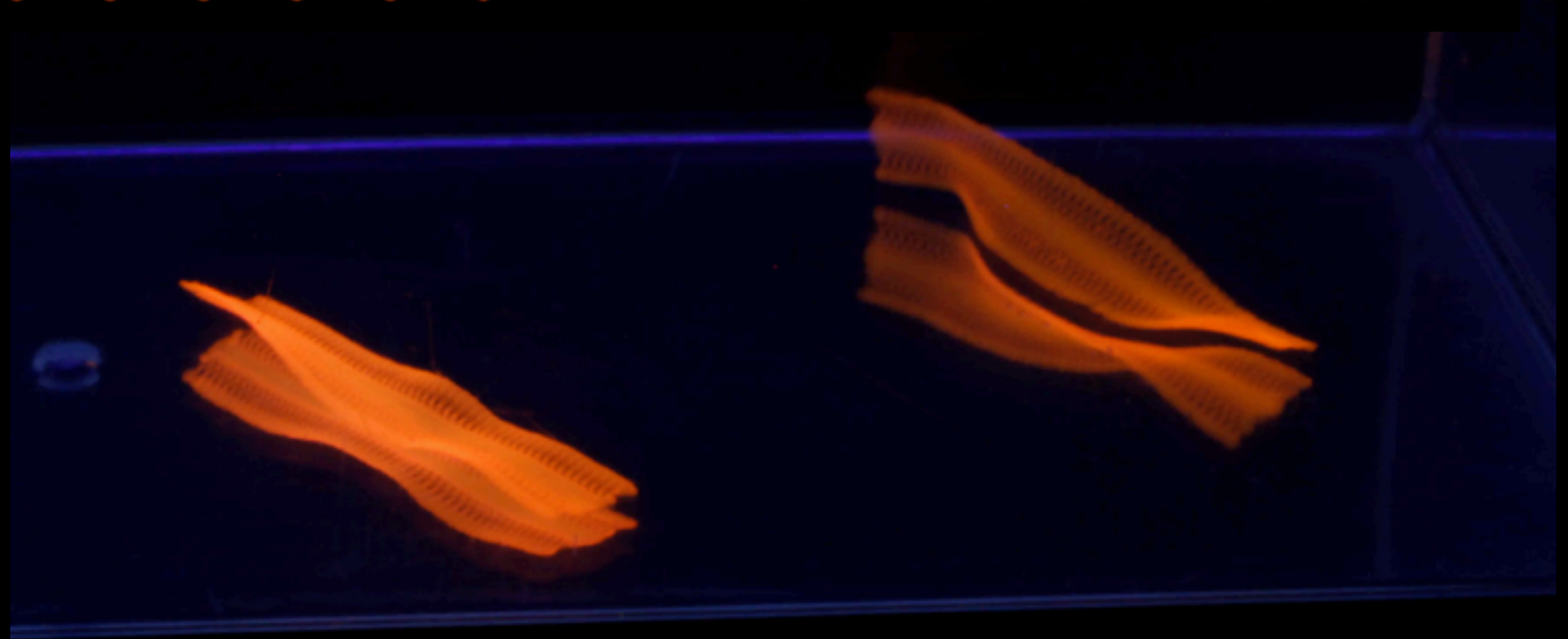
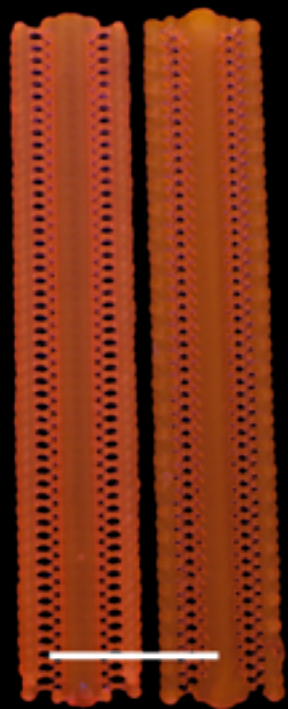
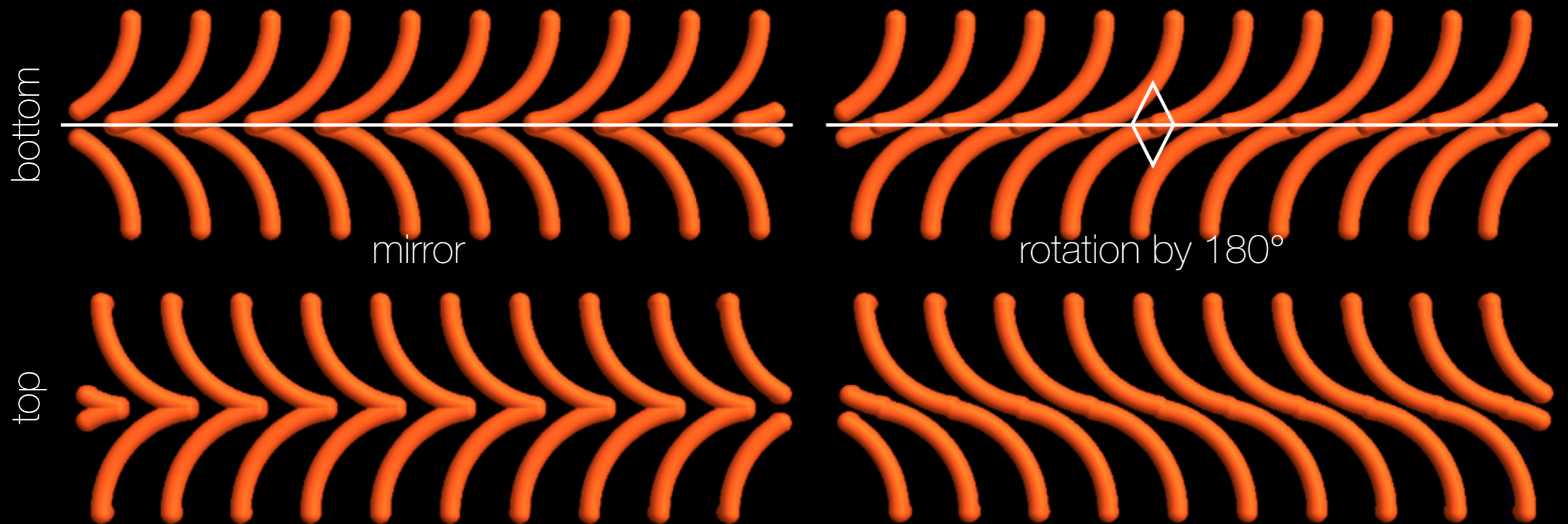
# To Twist or Not To Twist, That is the Question



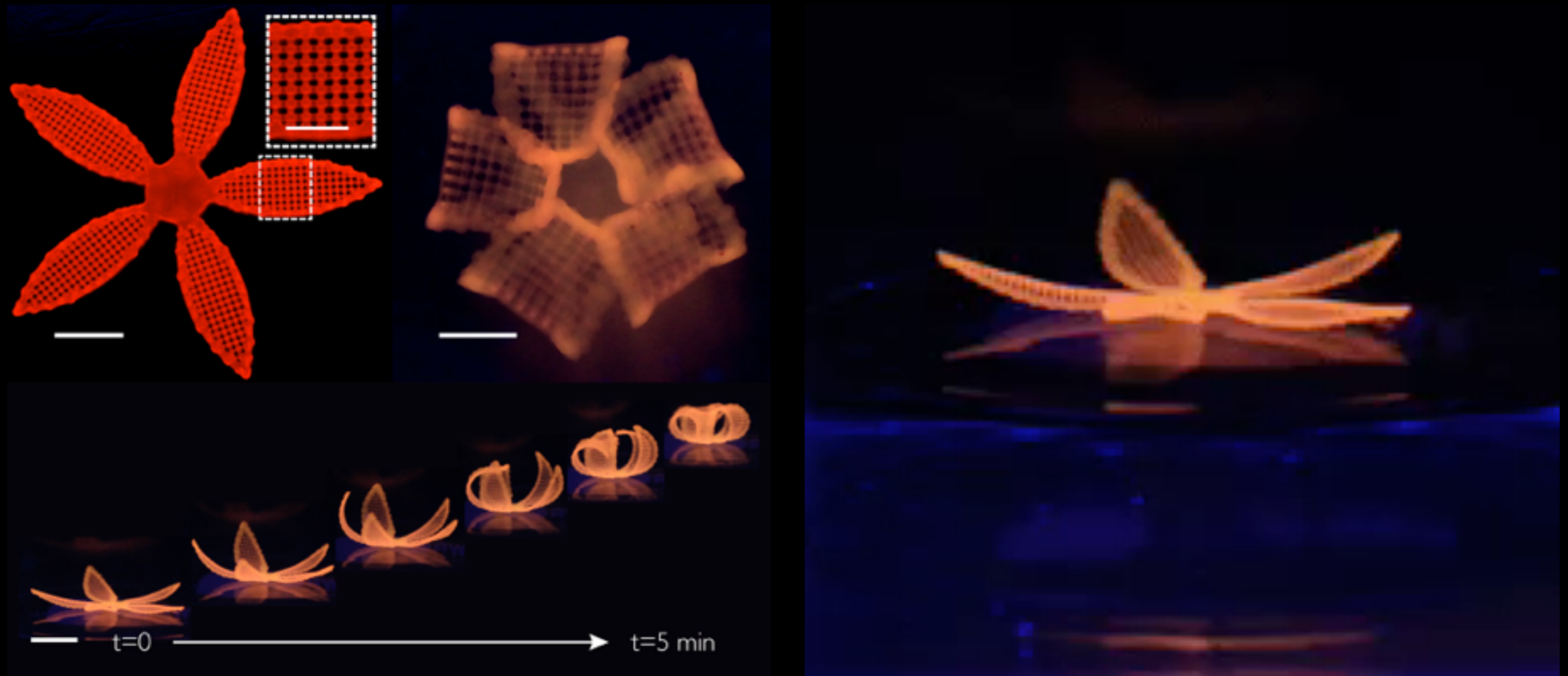
# To Twist or Not To Twist, That is the Question



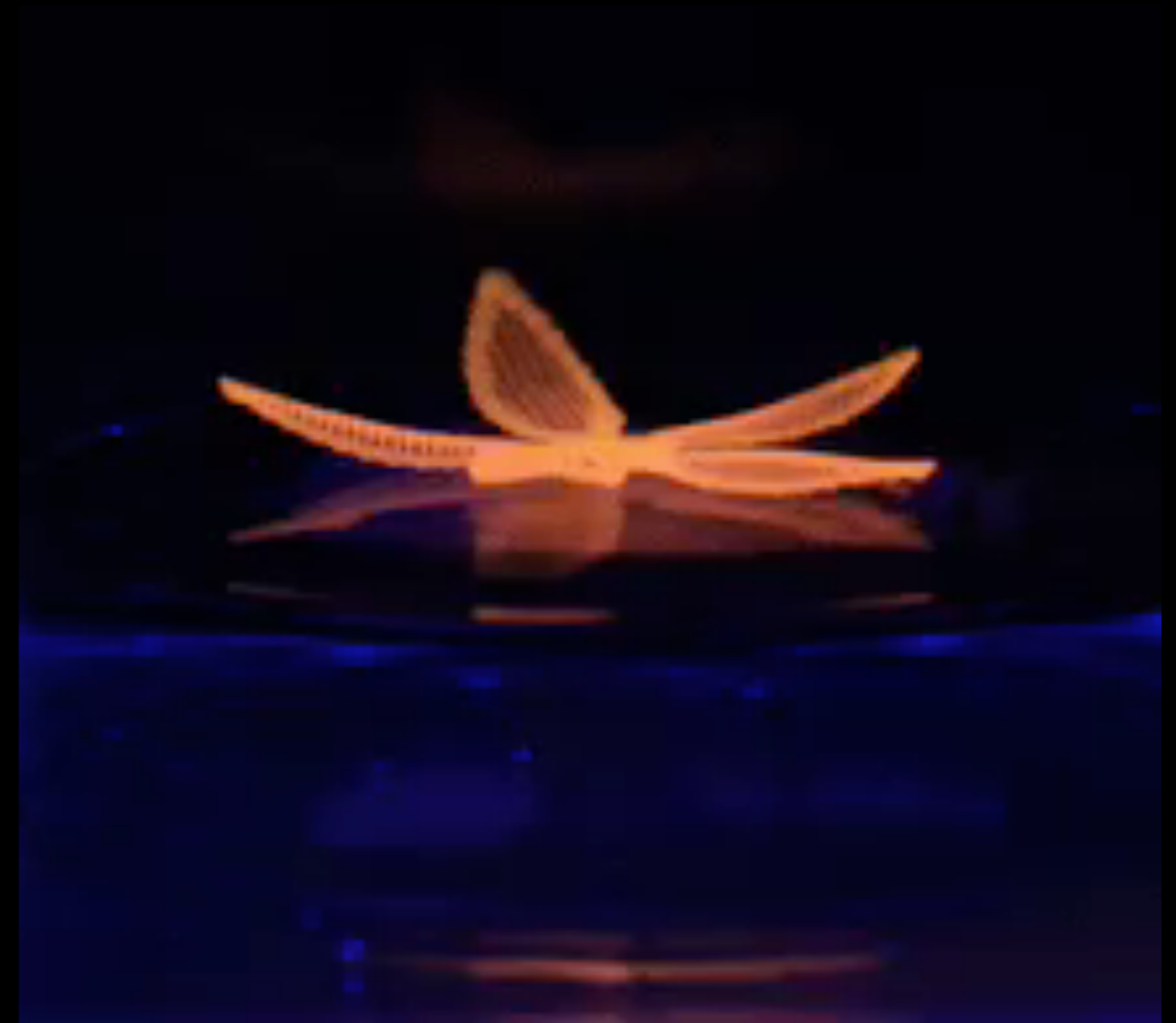
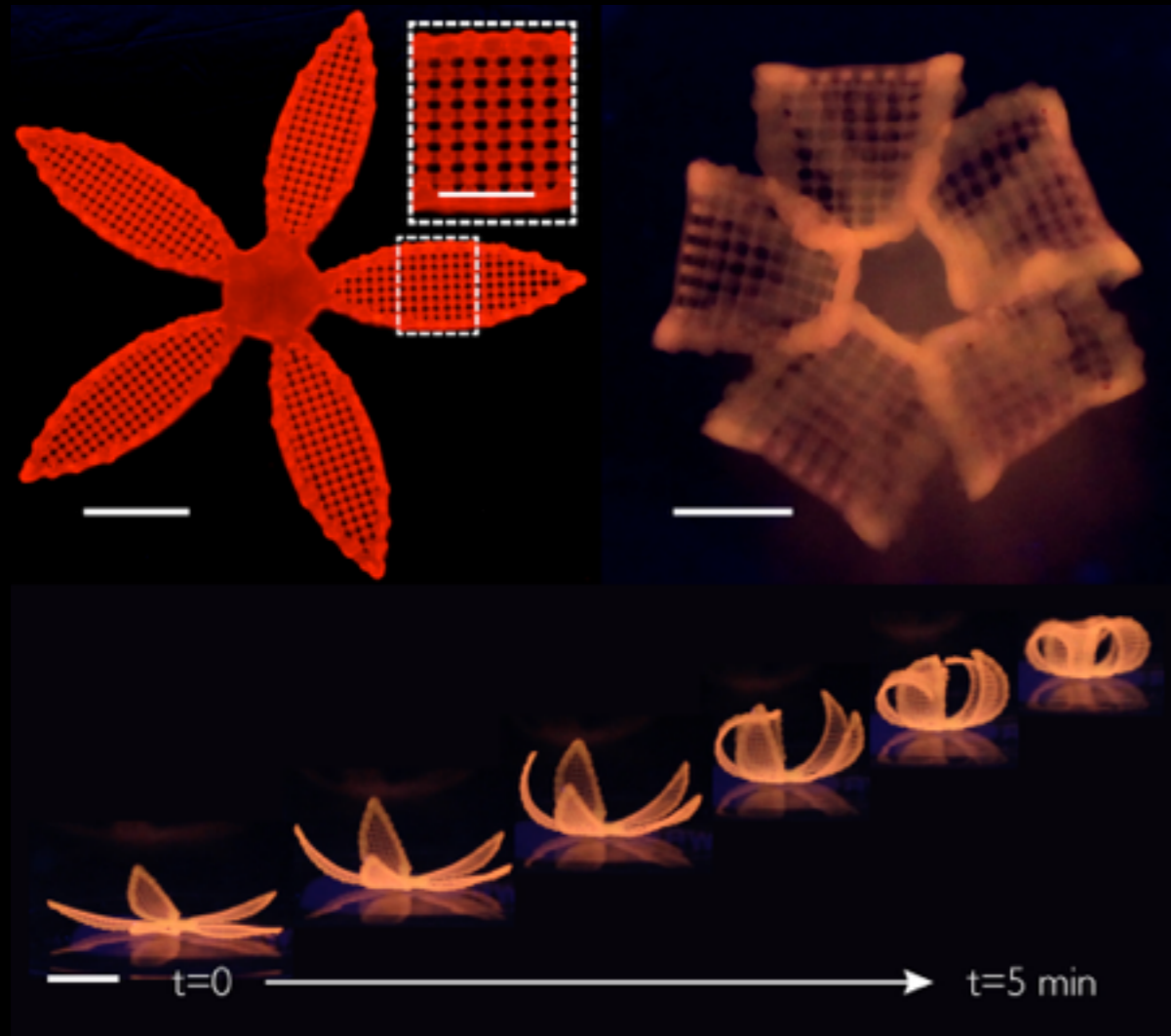
# To Twist or Not To Twist, That is the Question



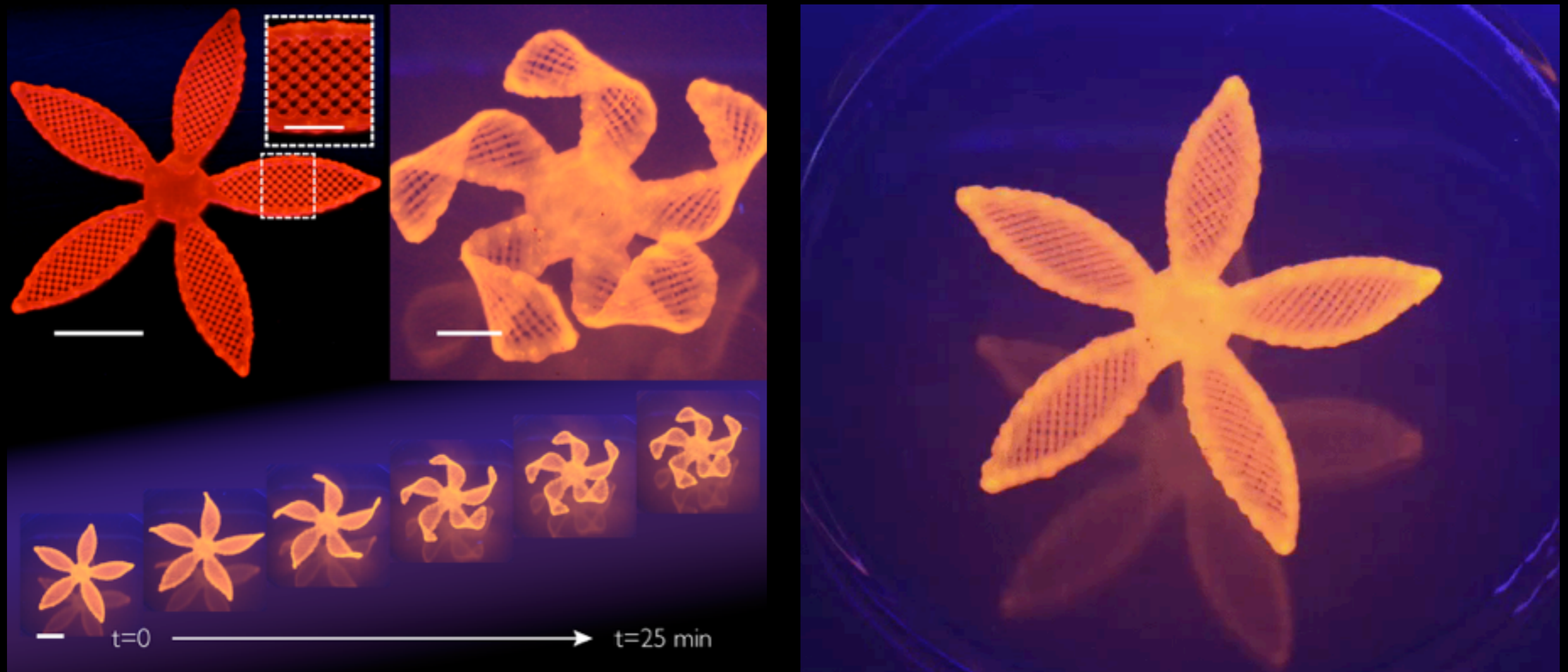
# Forty 4D Folding Flowers



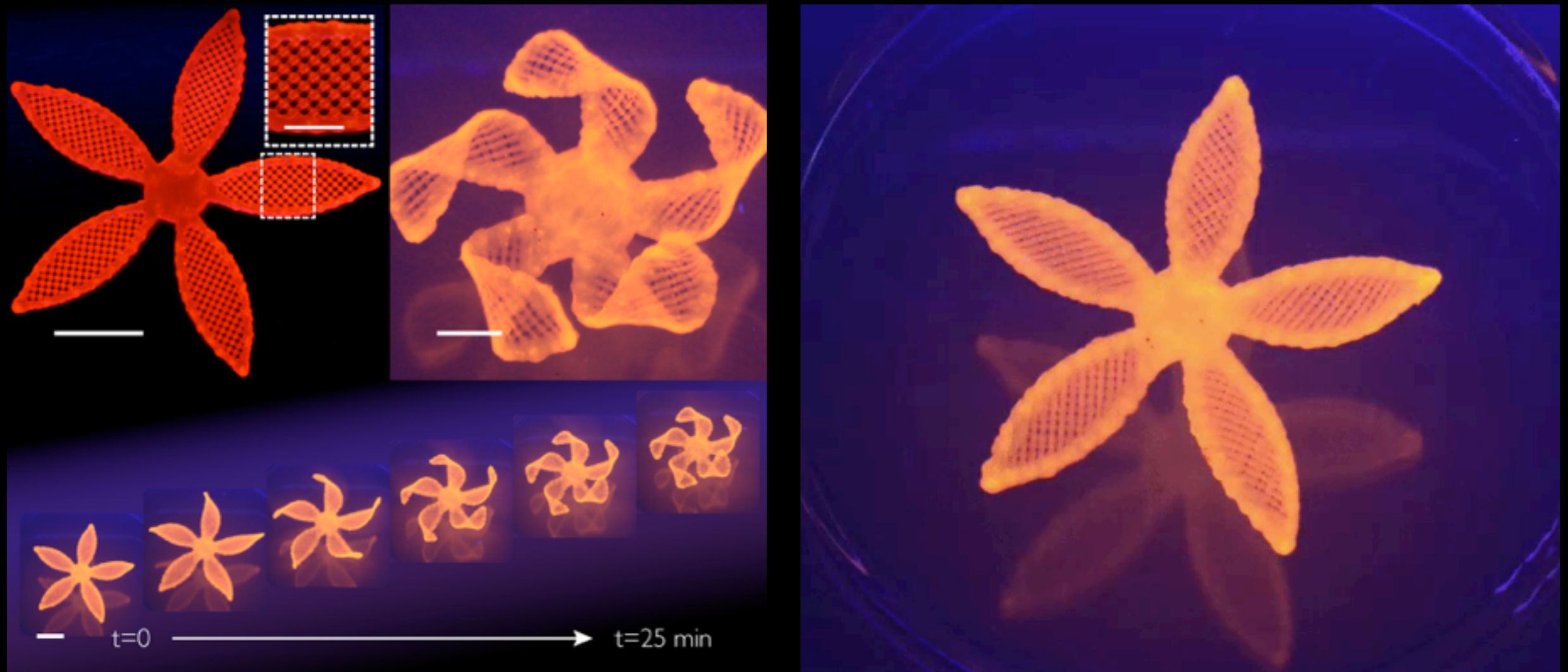
# Forty 4D Folding Flowers



# Forty 4D Folding Flowers

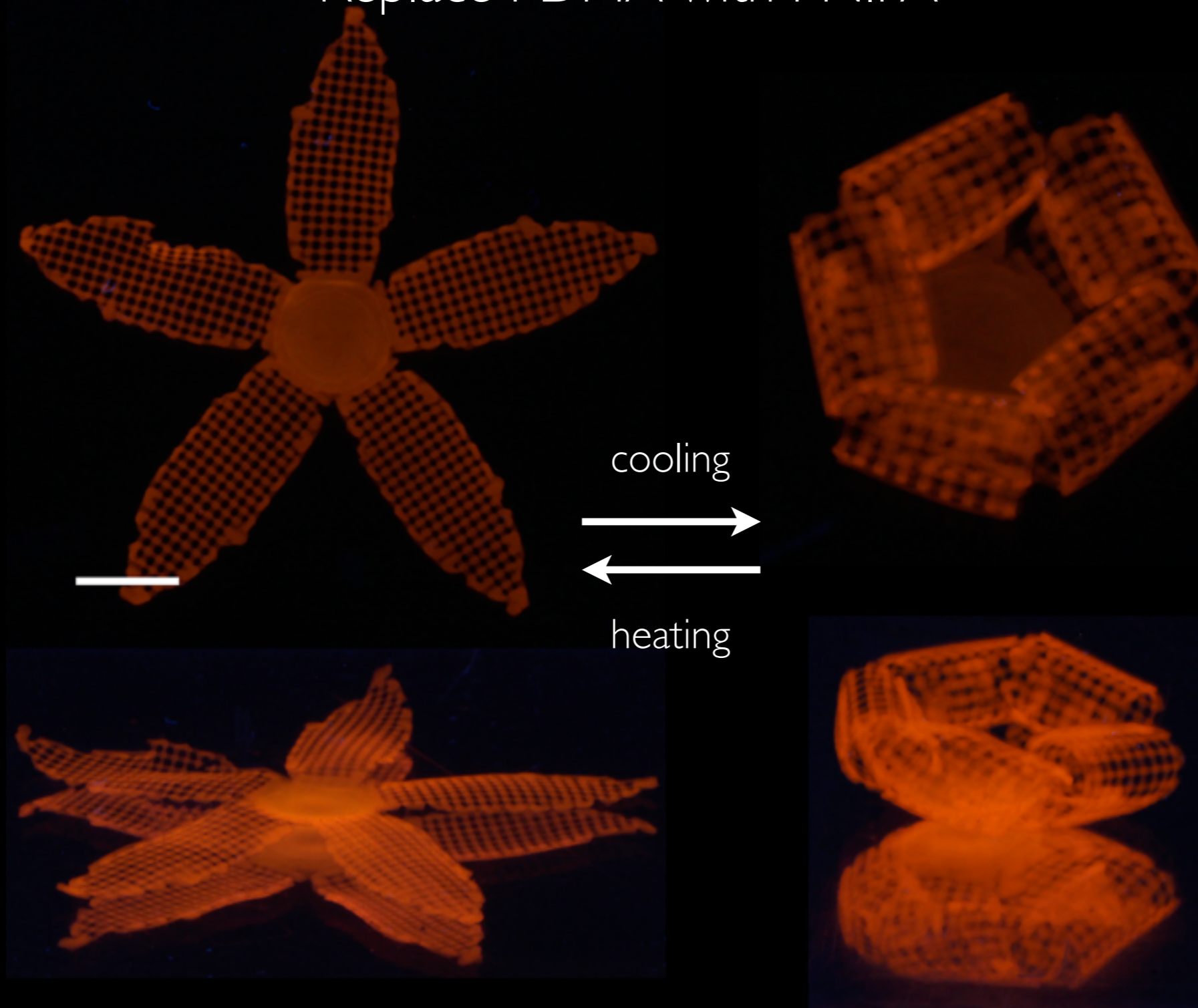


# Forty 4D Folding Flowers



# Reversibility

Replace PDMA with PNIPA



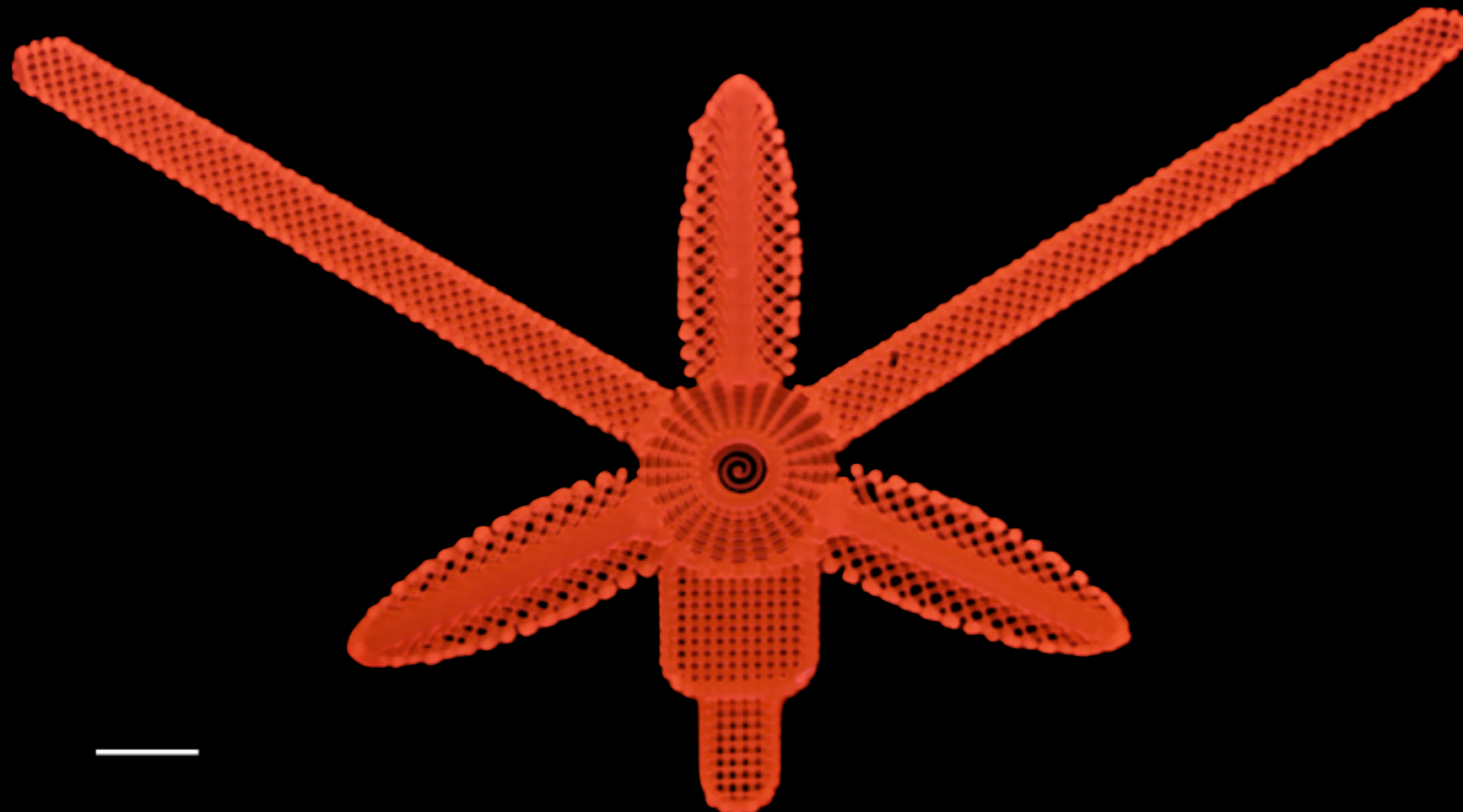
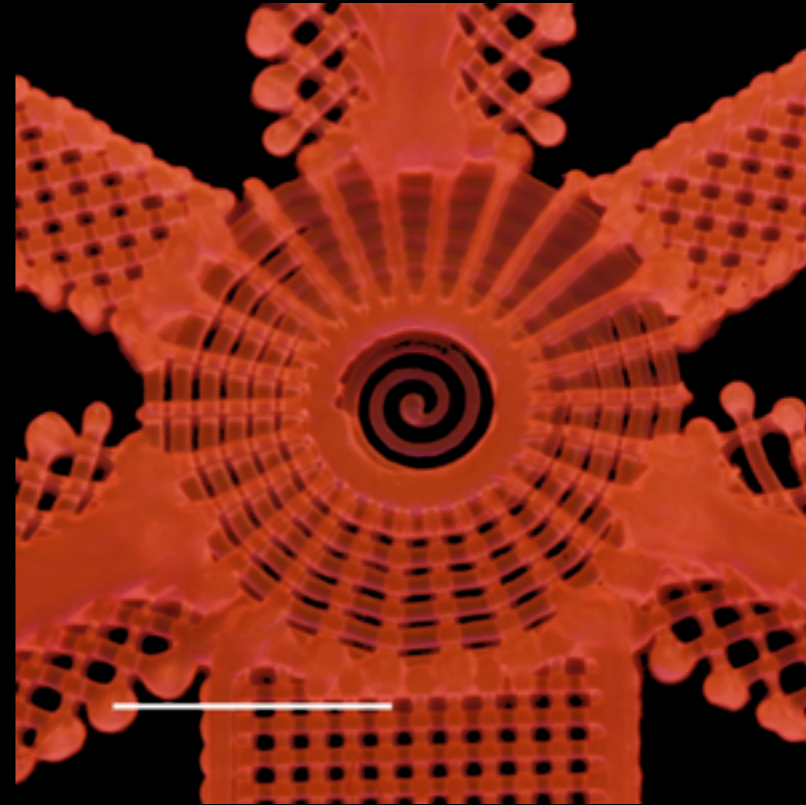
PNIPA undergoes a hydrophilic-hydrophobic transition at 40°C.



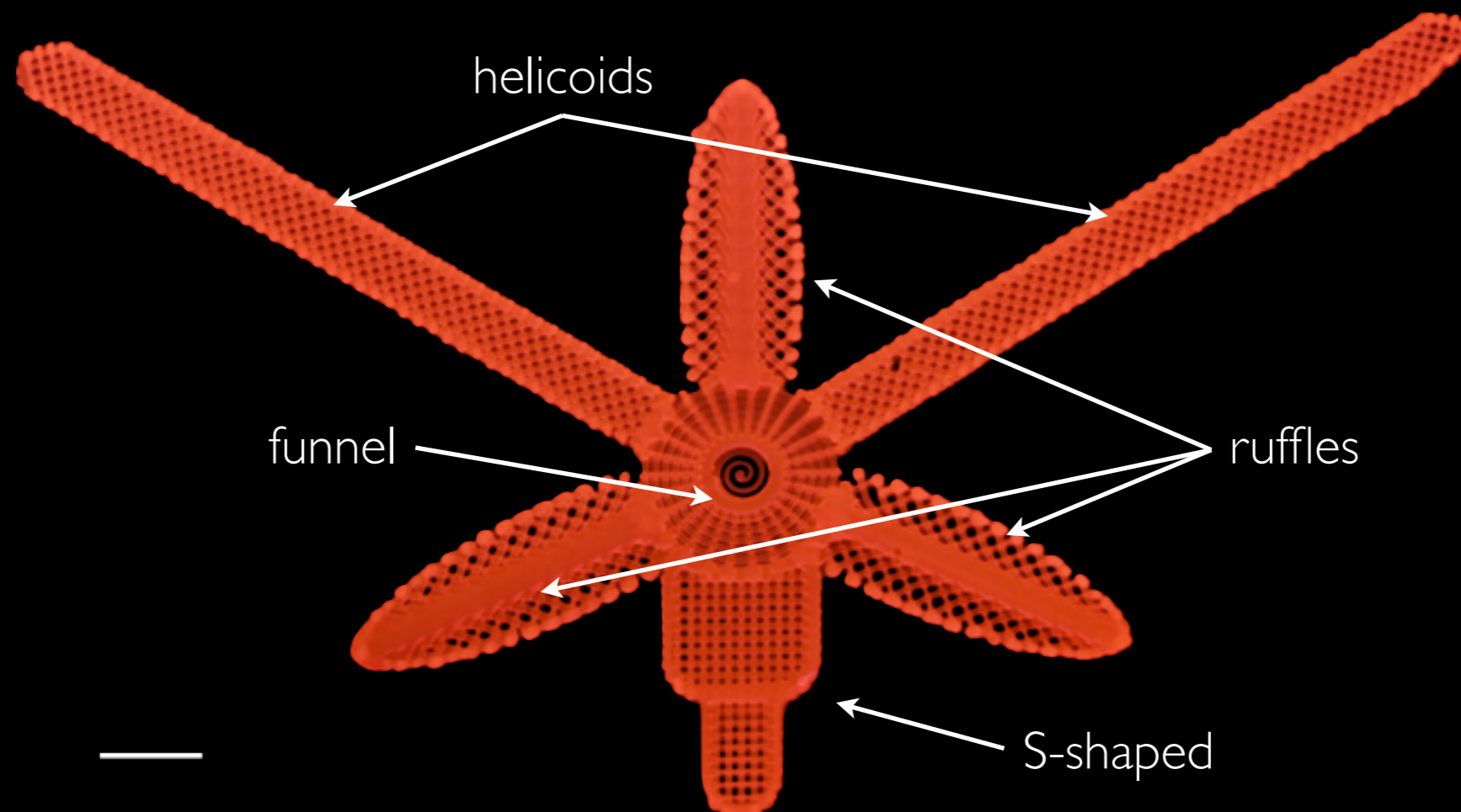
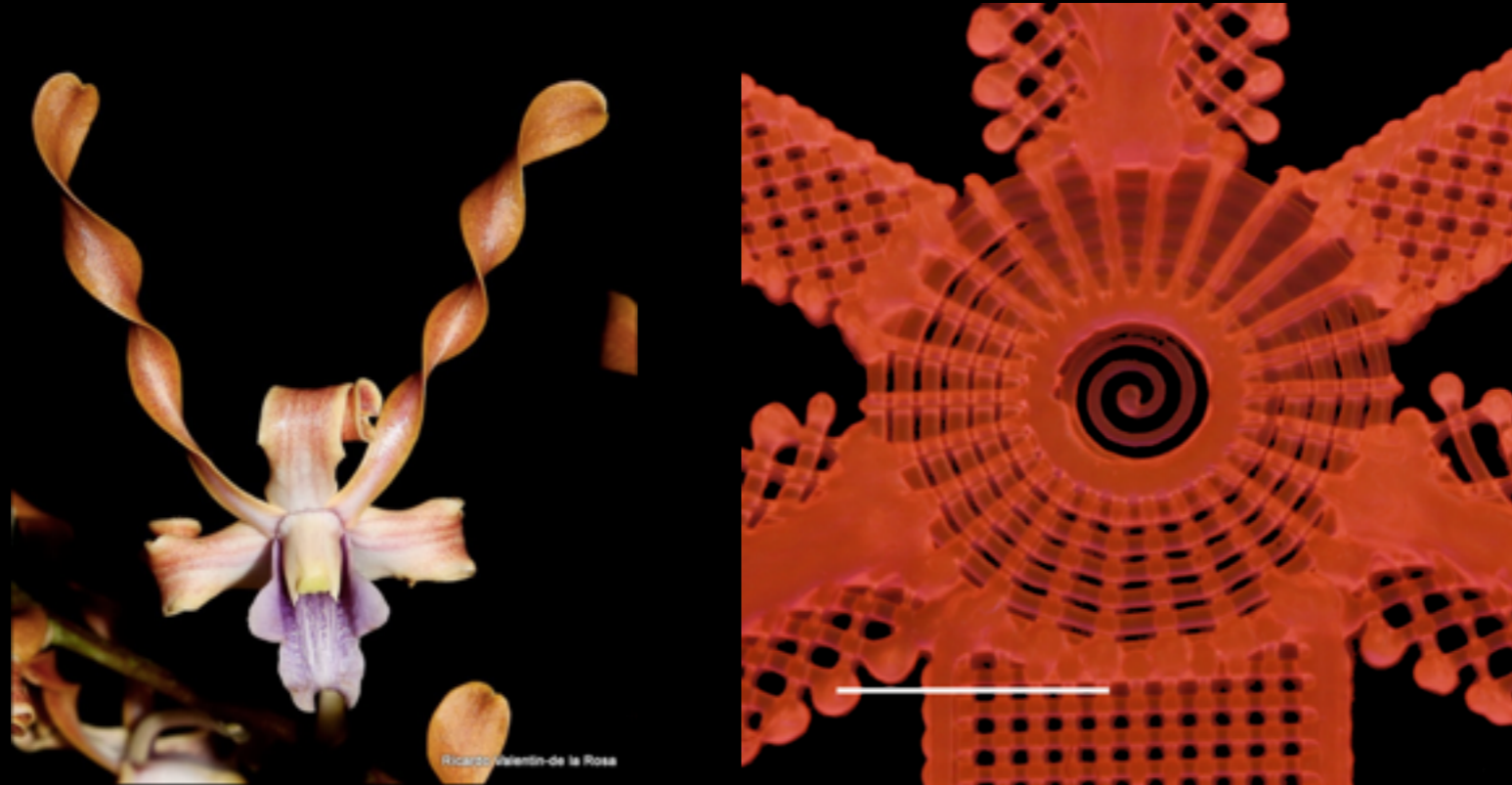
# Forty 4D Folding Flowers



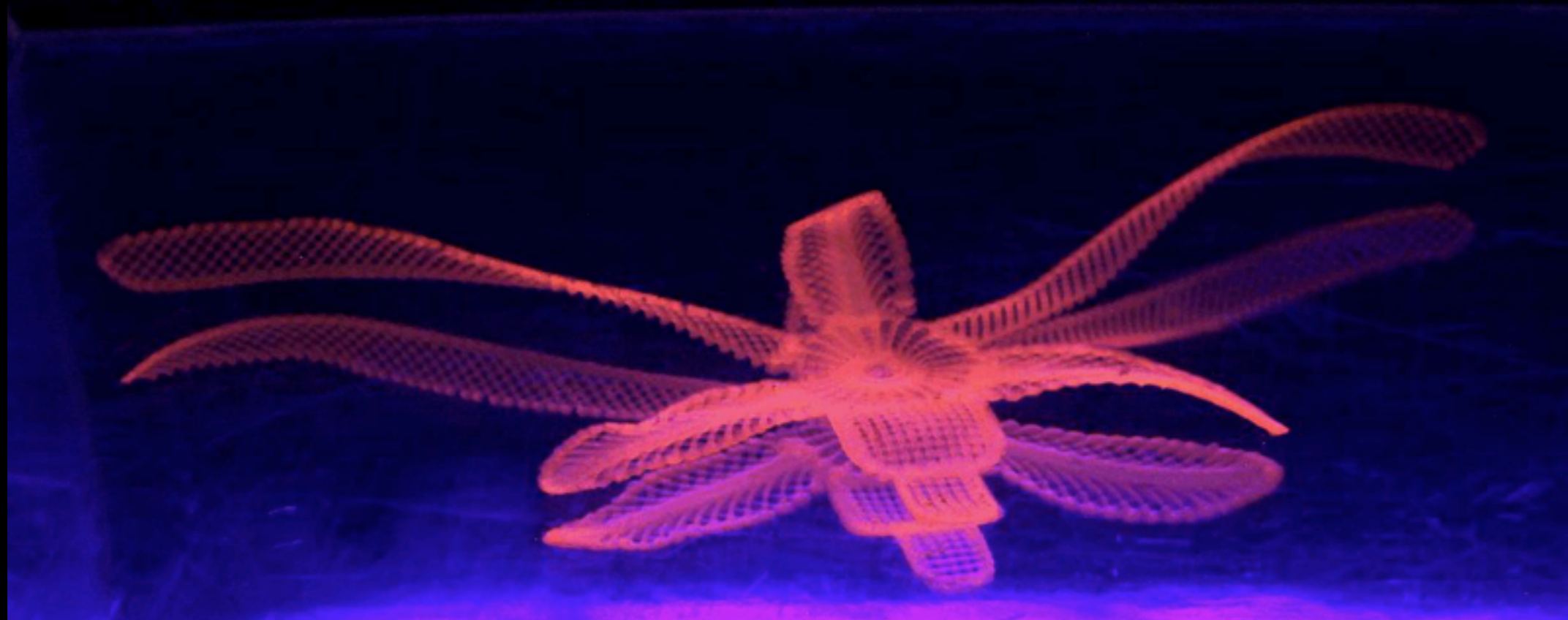
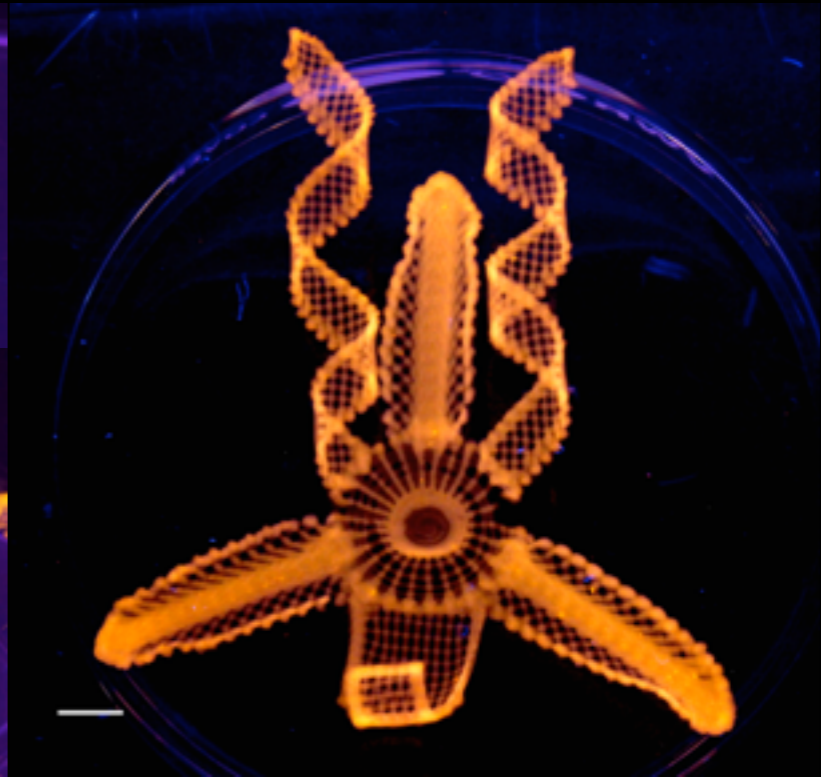
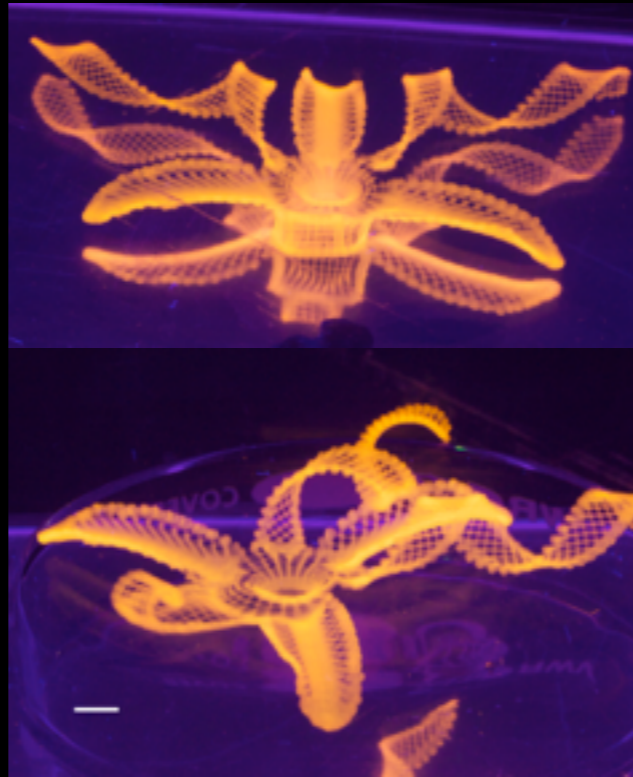
# Forty 4D Folding Flowers



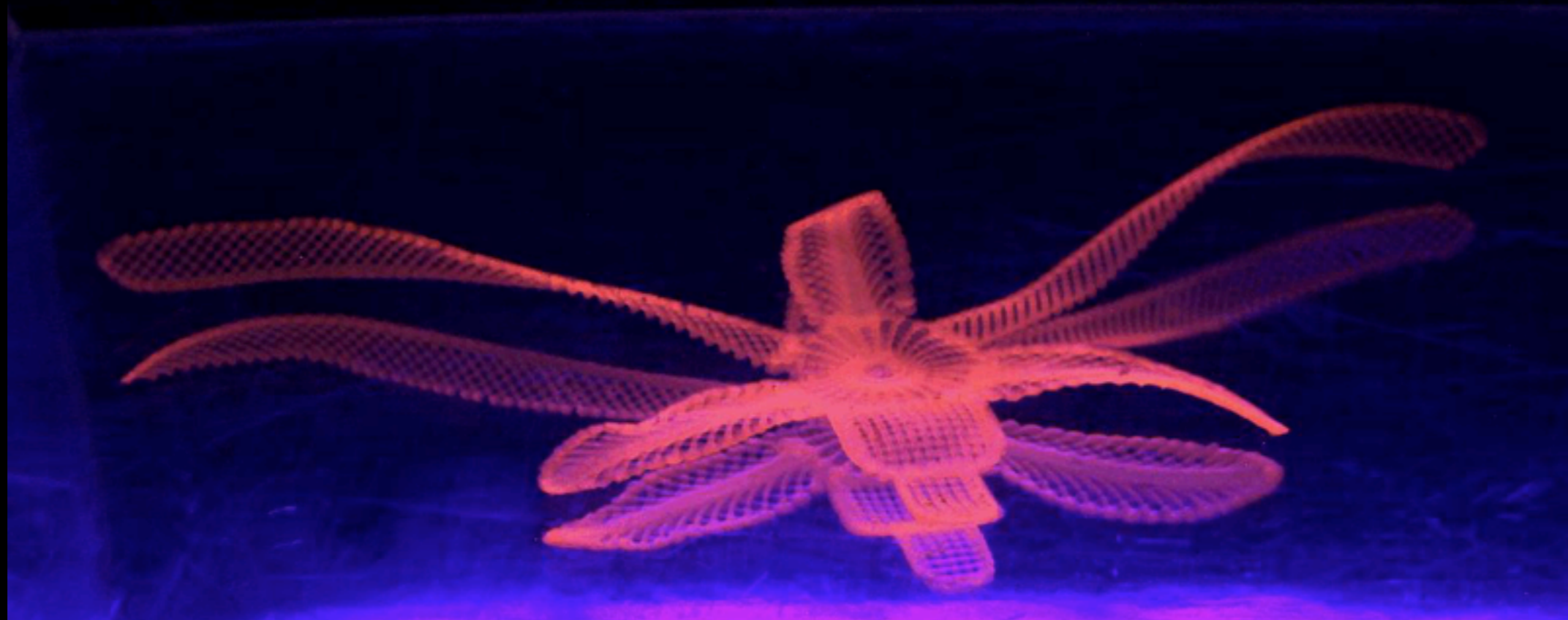
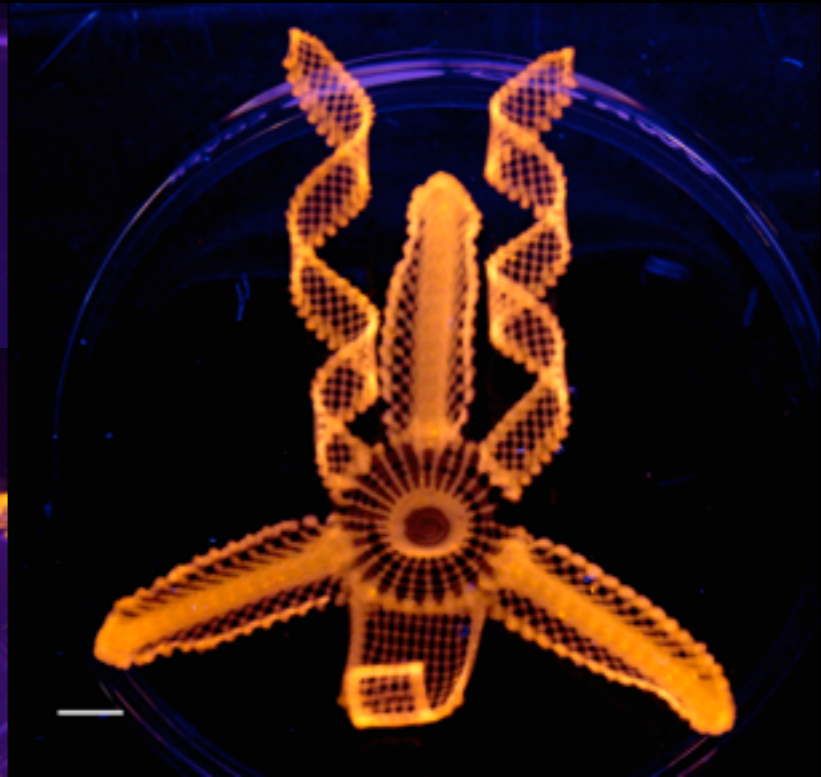
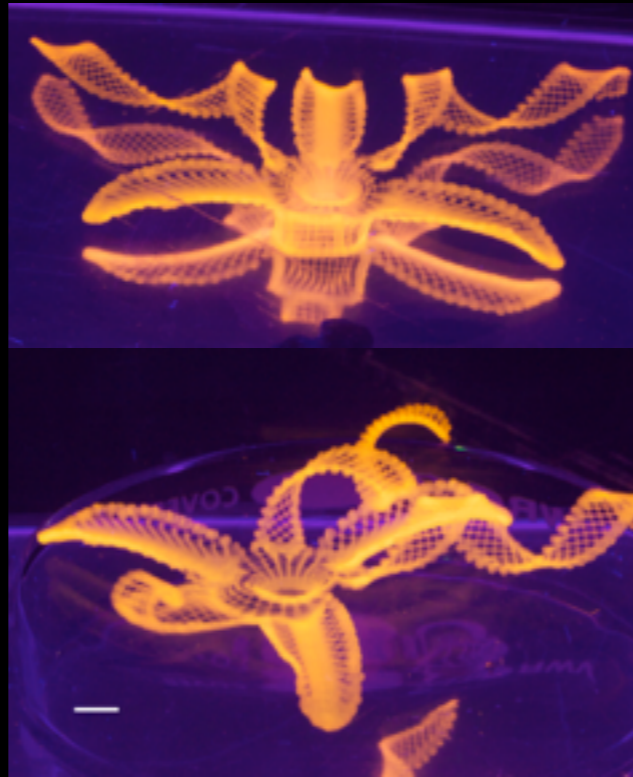
# Forty 4D Folding Flowers



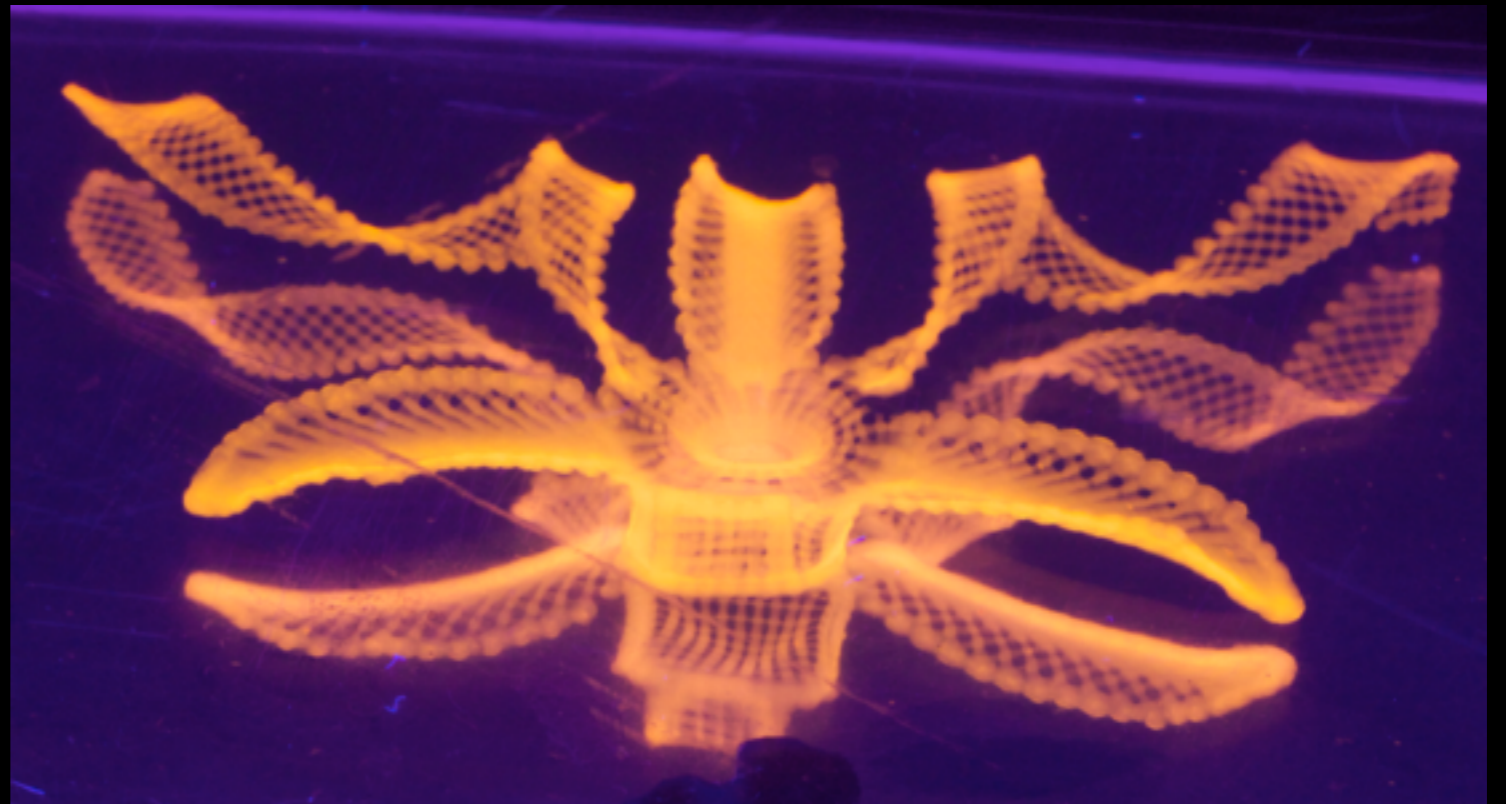
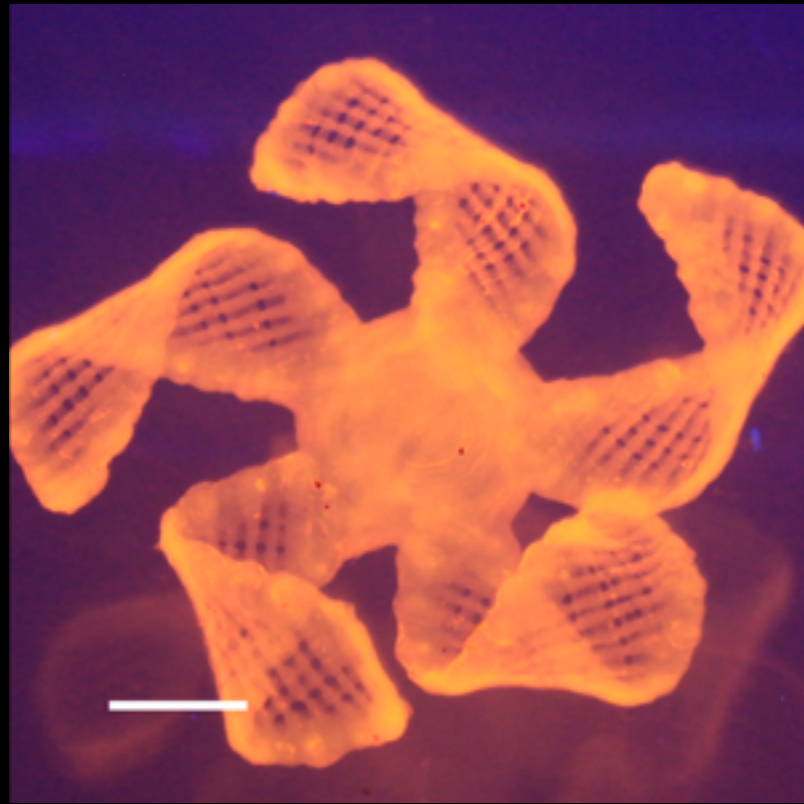
# Forty 4D Folding Flowers



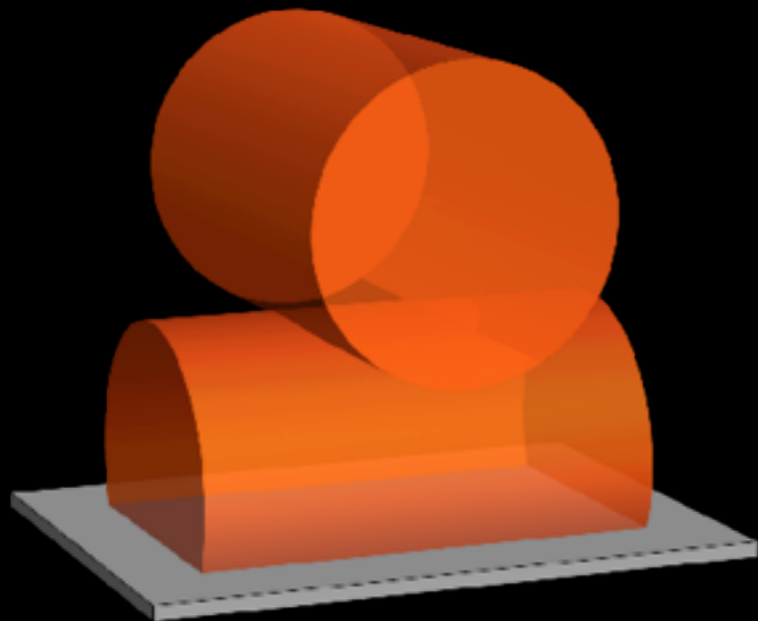
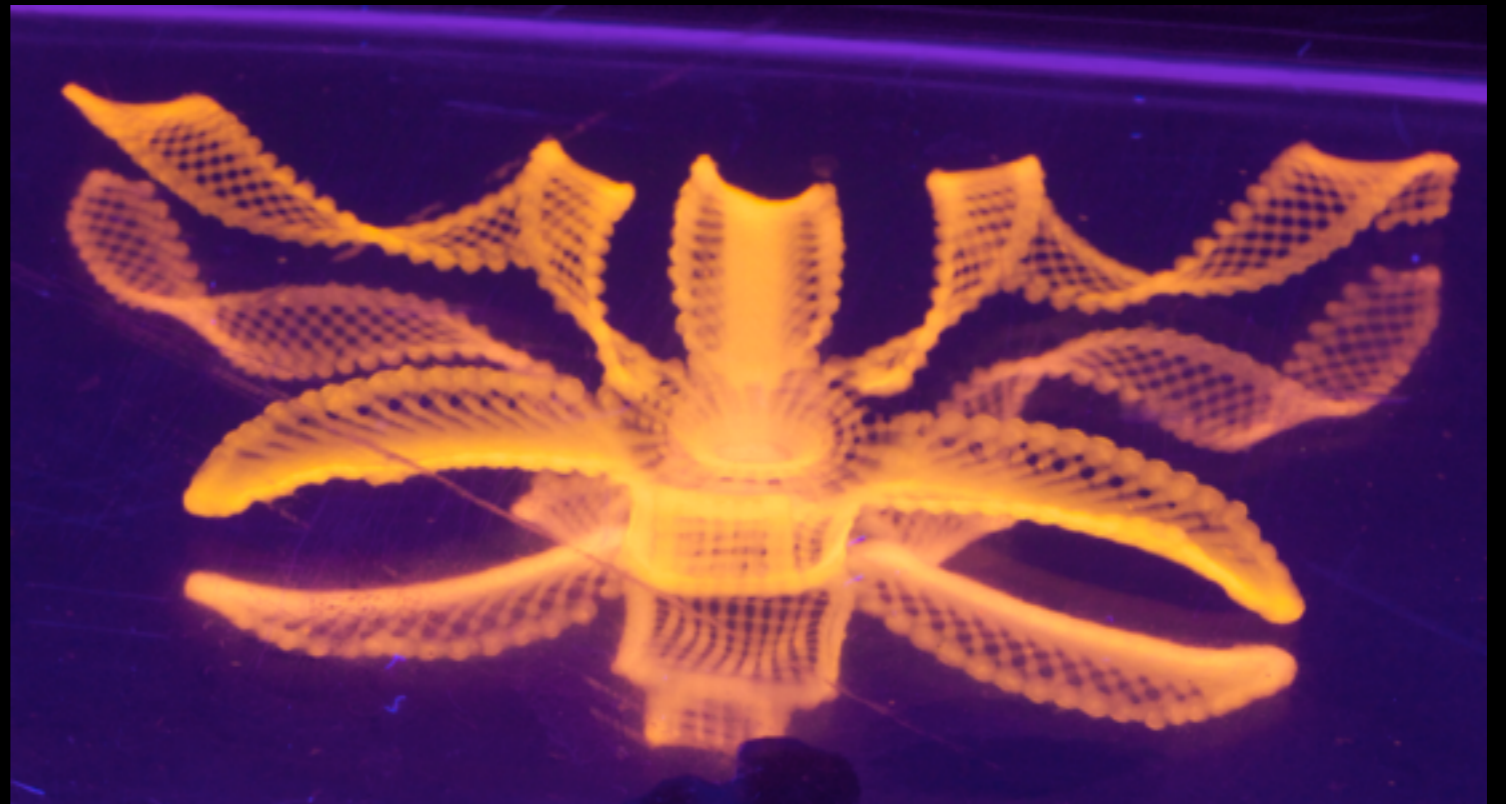
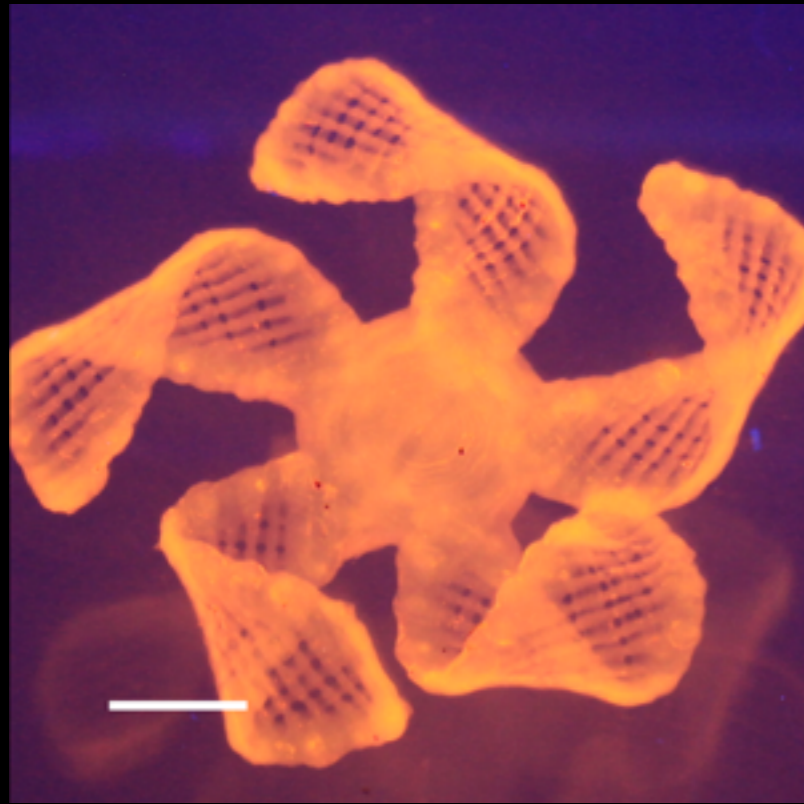
# Forty 4D Folding Flowers



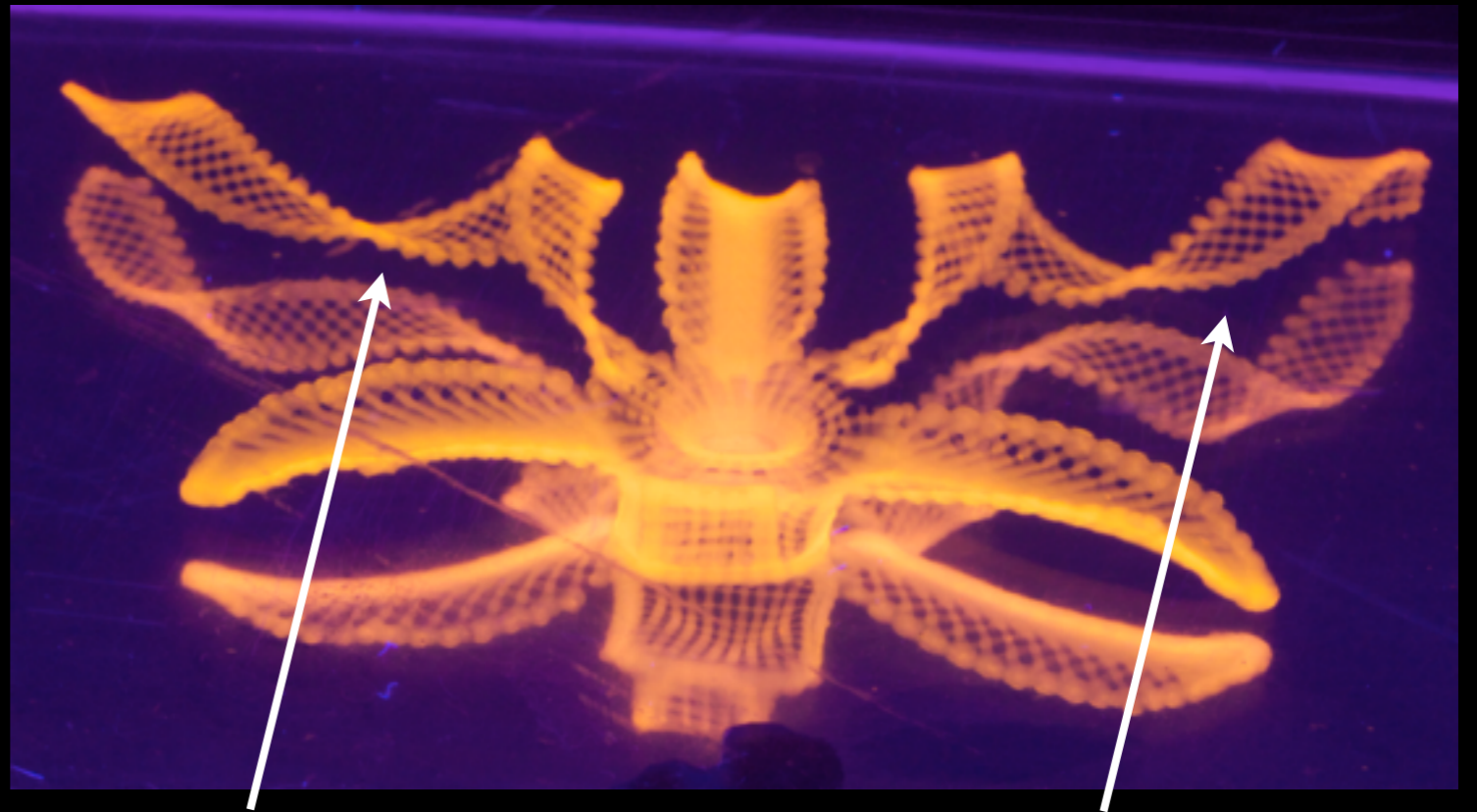
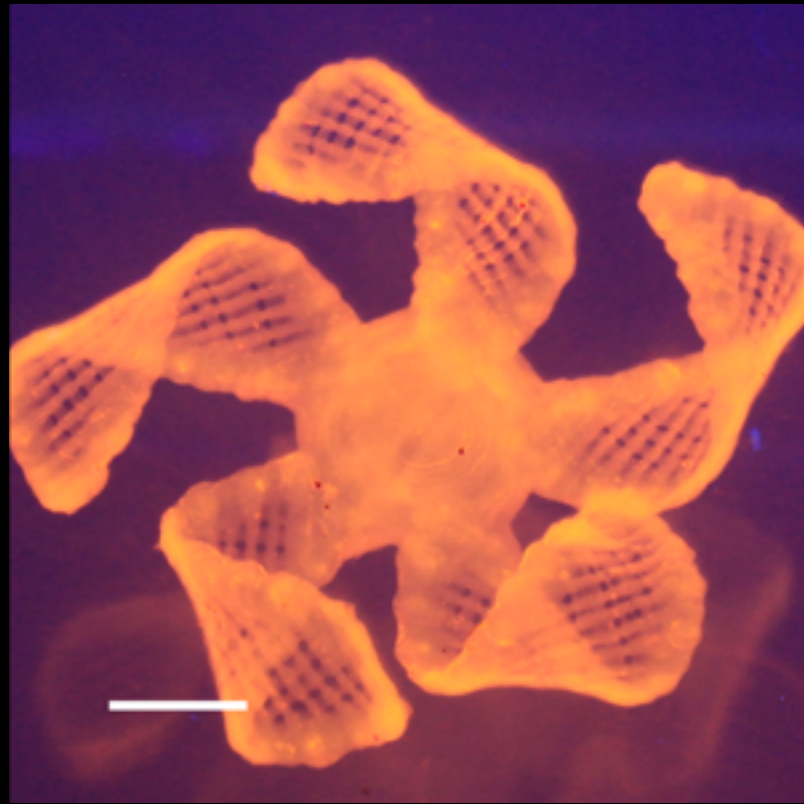
# Left-handed or Right-handed?



# Left-handed or Right-handed?

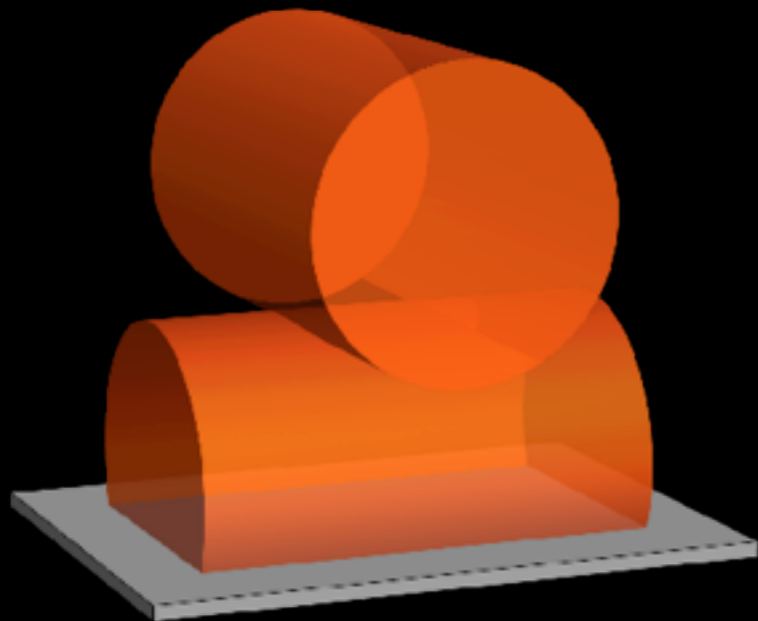


# Left-handed or Right-handed?



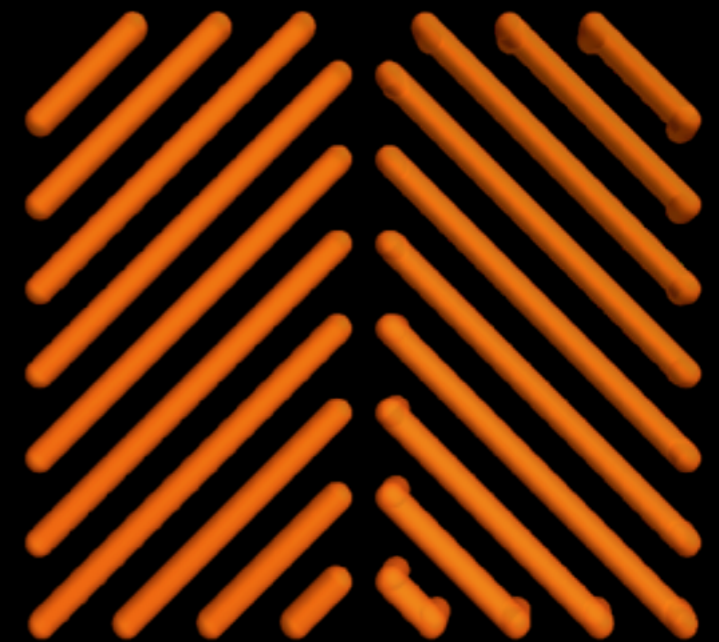
Left-handed

Right-handed



bottom

top

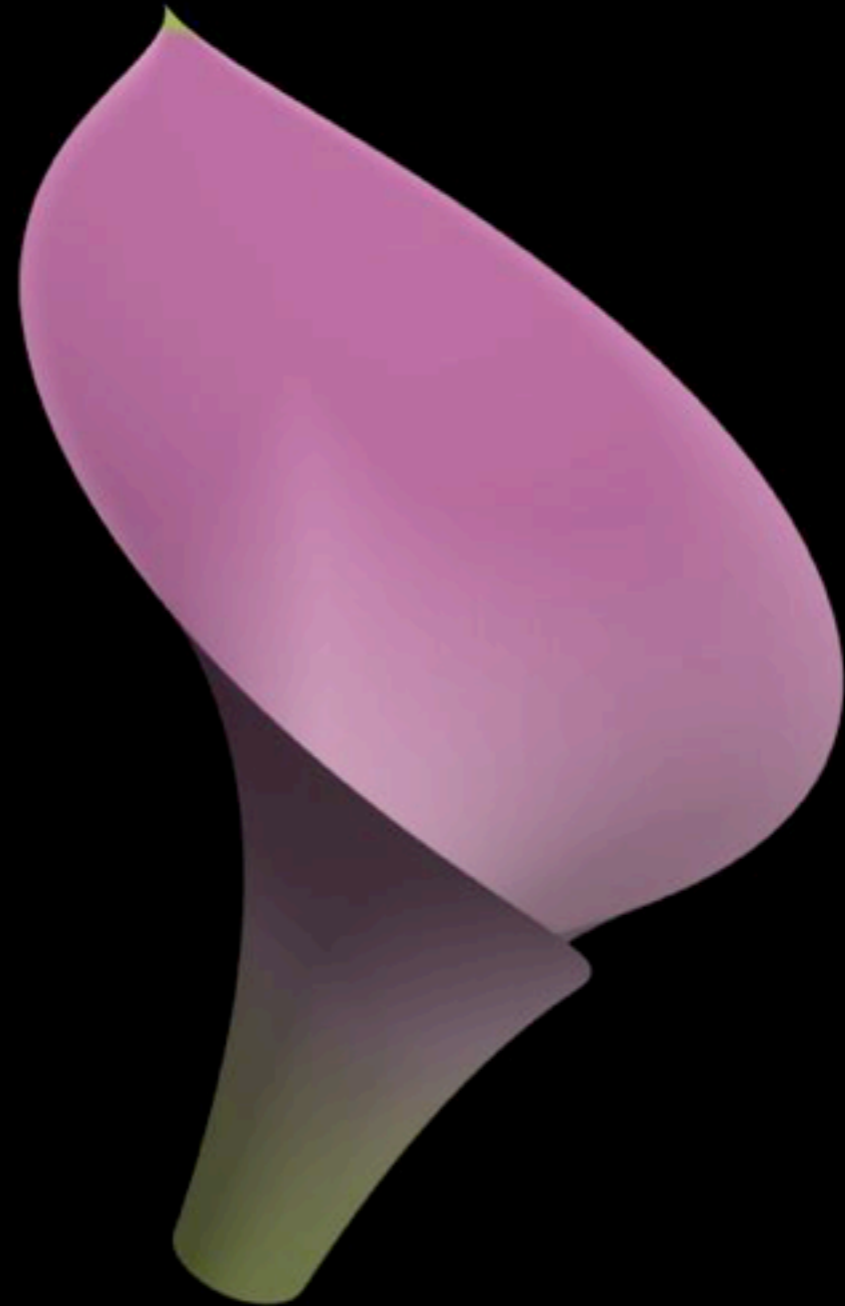


bottom

top

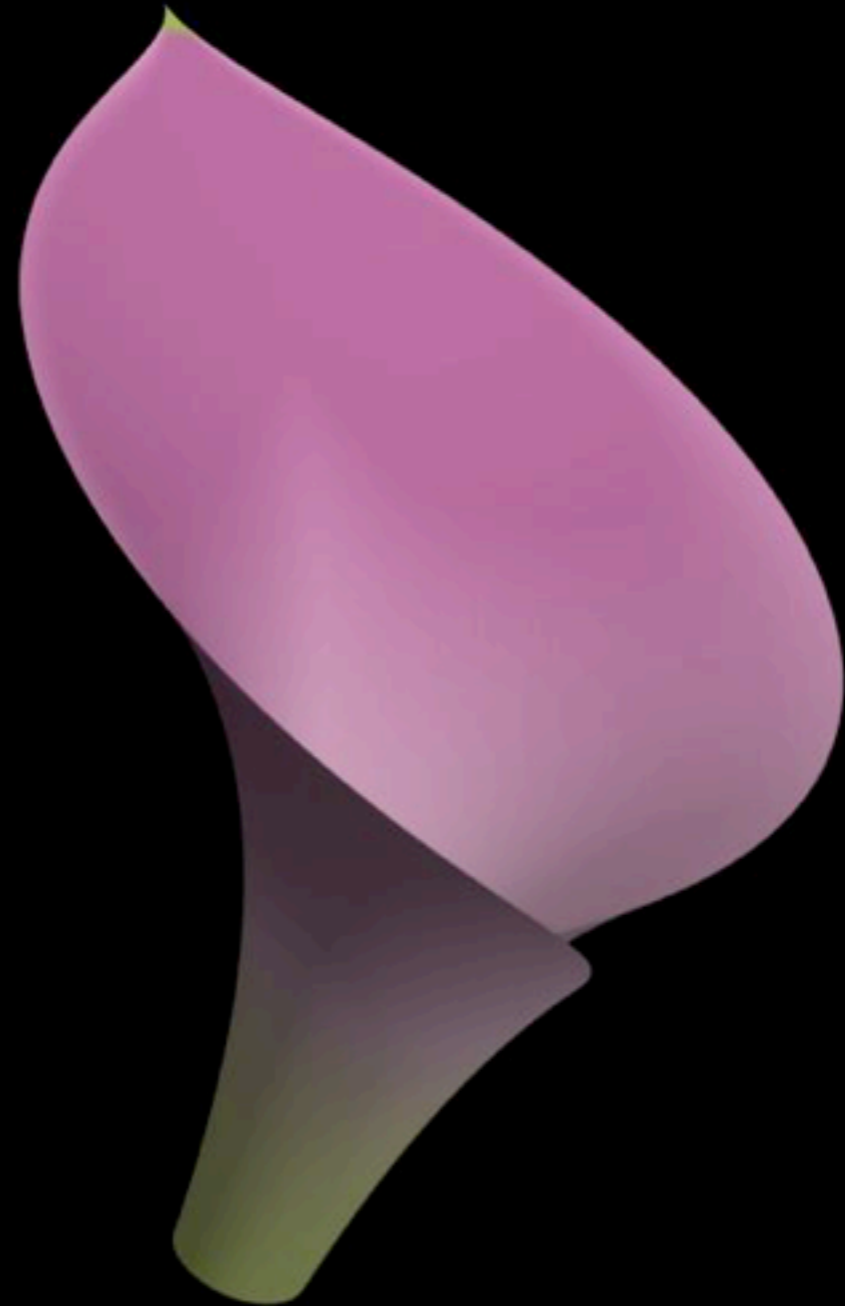


# The Inverse Problem



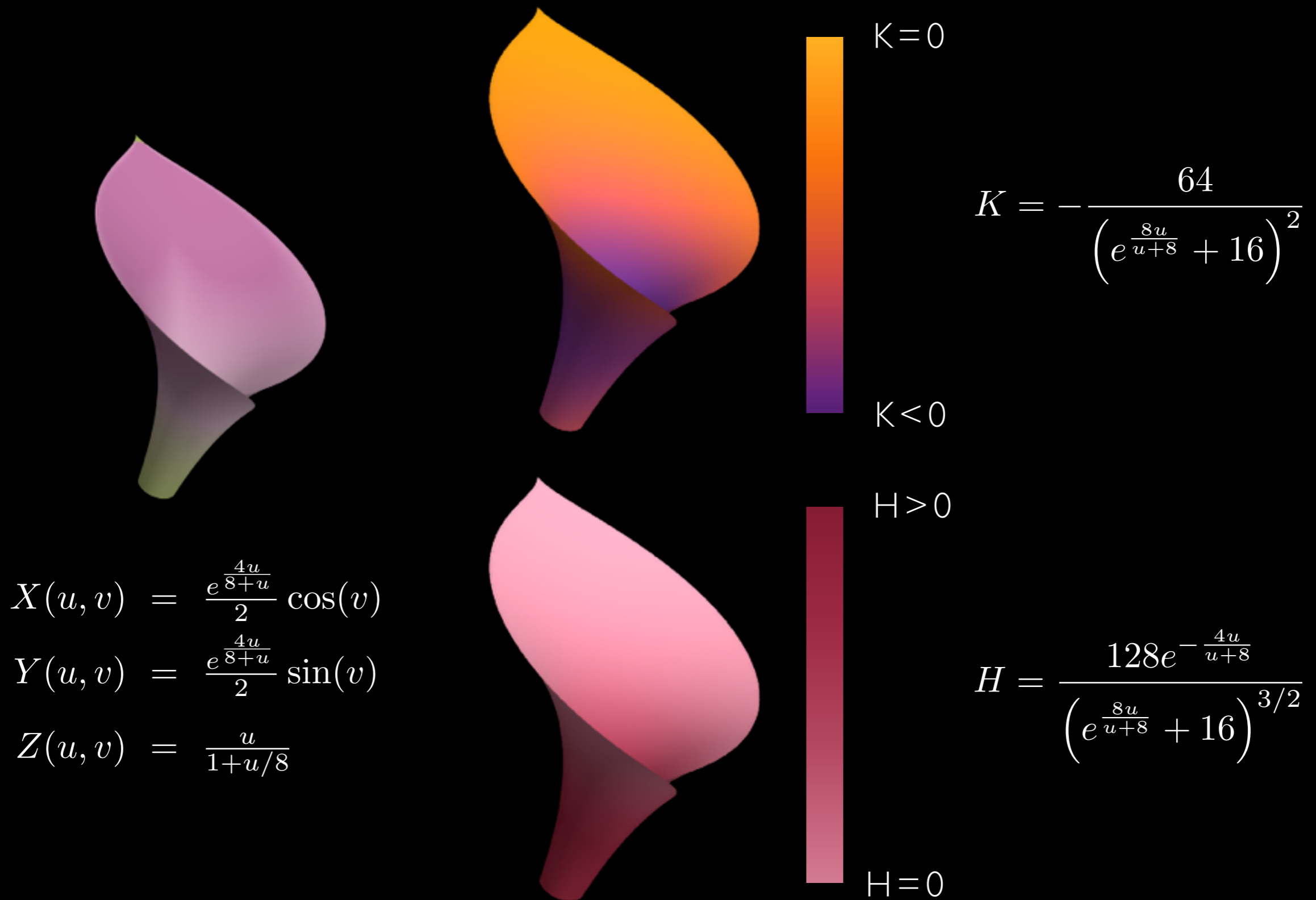
$$\Phi(u, v) = \left\{ \frac{e^{\frac{4u}{8+u}}}{2} \cos(v), \frac{e^{\frac{4u}{8+u}}}{2} \sin(v), \frac{u}{1 + u/8} \right\}$$

# The Inverse Problem



$$\Phi(u, v) = \left\{ \frac{e^{\frac{4u}{8+u}}}{2} \cos(v), \frac{e^{\frac{4u}{8+u}}}{2} \sin(v), \frac{u}{1 + u/8} \right\}$$

# The Inverse Problem



# Programming Local Curvatures

$$H = \frac{\alpha_{\perp} - \alpha_{\parallel}}{h} \frac{c_1 \sin^2(\theta)}{c_2 - c_3 \cos(2\theta) + m^4 \cos(4\theta)}, \quad K = -\frac{(\alpha_{\perp} - \alpha_{\parallel})^2}{h^2} \frac{c_4 \sin^2(\theta)}{c_5 - c_6 \cos(2\theta) + m^4 \cos(4\theta)}$$

Given:  $H, K, \alpha_{\parallel}, \alpha_{\perp}, \mathbf{E}^{(1)}, \mathbf{E}^{(2)}$     Solve for:  $\theta, m = a_1/a_2$



bottom



top

# Programming Local Curvatures

$$H = \frac{\alpha_{\perp} - \alpha_{\parallel}}{h} \frac{c_1 \sin^2(\theta)}{c_2 - c_3 \cos(2\theta) + m^4 \cos(4\theta)}, \quad K = -\frac{(\alpha_{\perp} - \alpha_{\parallel})^2}{h^2} \frac{c_4 \sin^2(\theta)}{c_5 - c_6 \cos(2\theta) + m^4 \cos(4\theta)}$$

Given:  $H, K, \alpha_{\parallel}, \alpha_{\perp}, \mathbf{E}^{(1)}, \mathbf{E}^{(2)}$     Solve for:  $\theta, m = a_1/a_2$



bottom

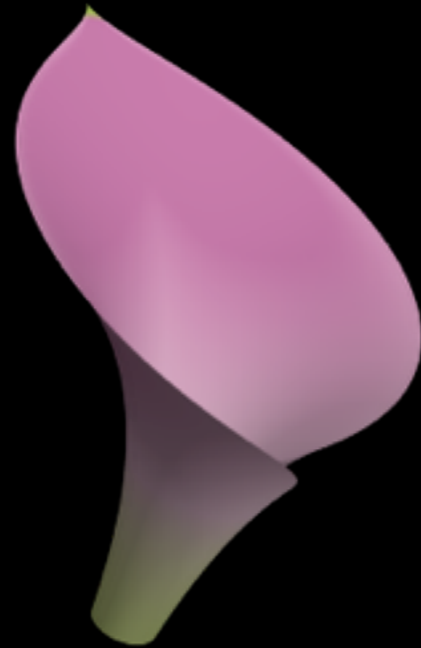


top

# Programming Local Curvatures



# Programming Local Curvatures



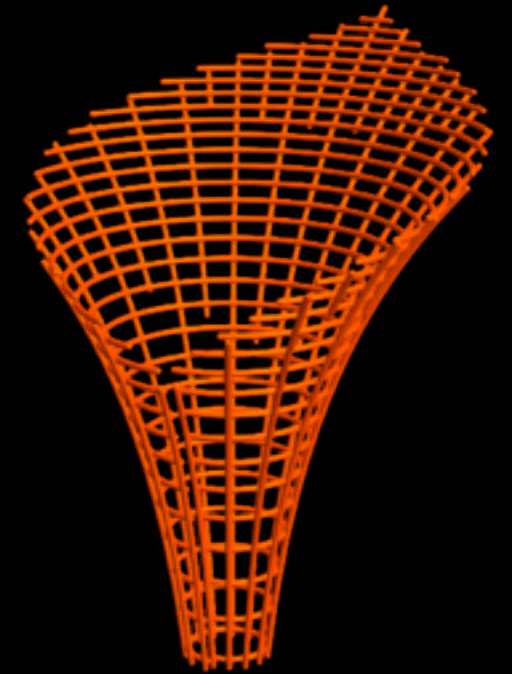
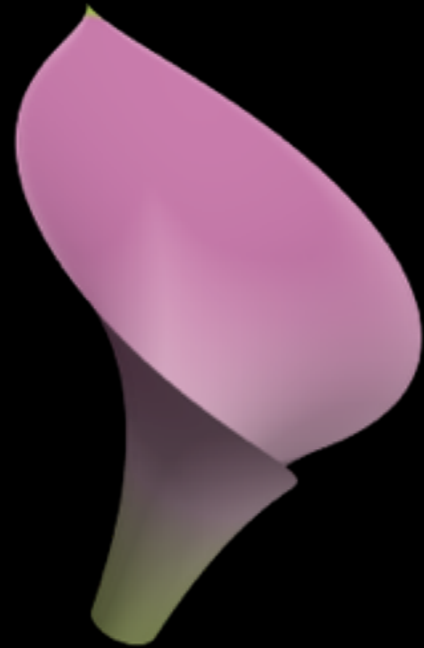
bottom



top



# Programming Local Curvatures





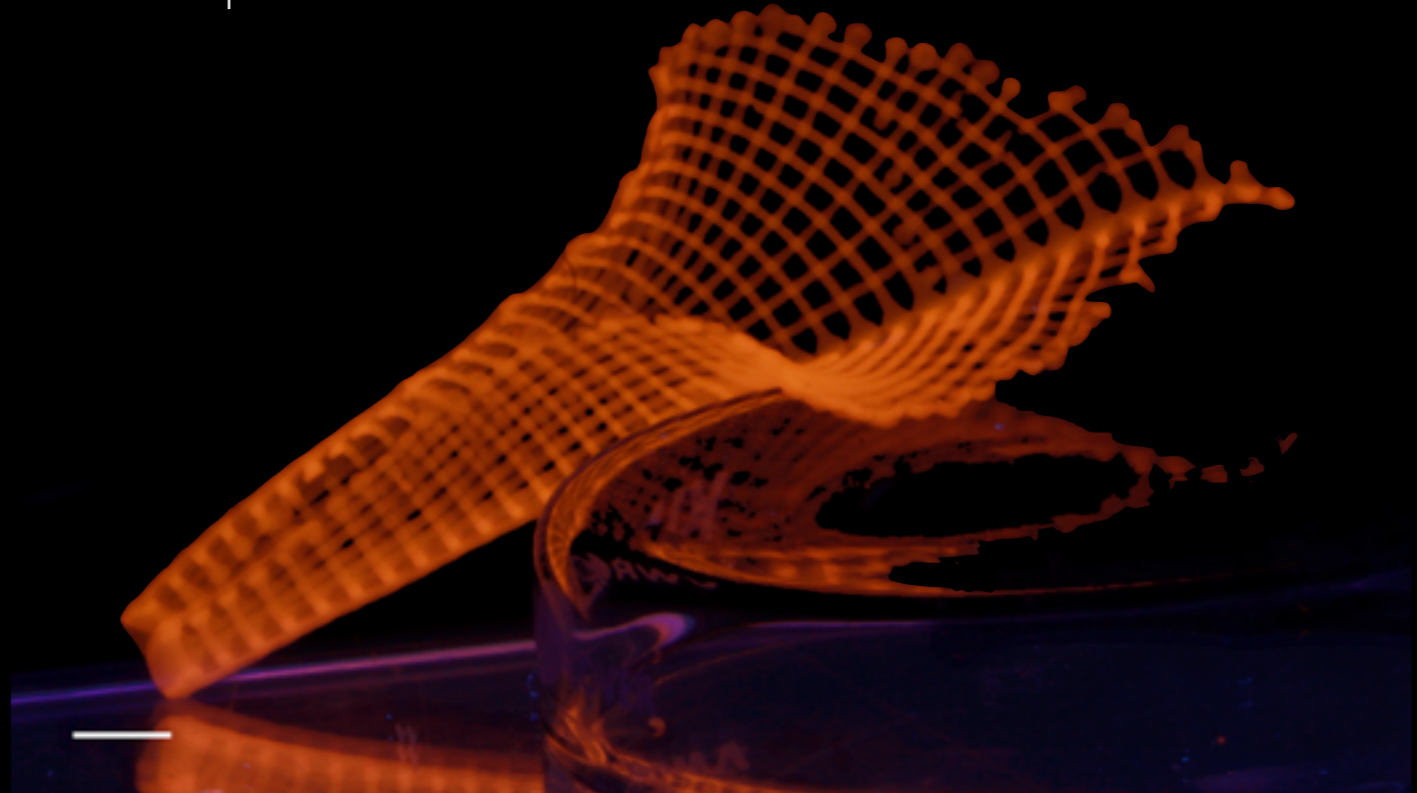
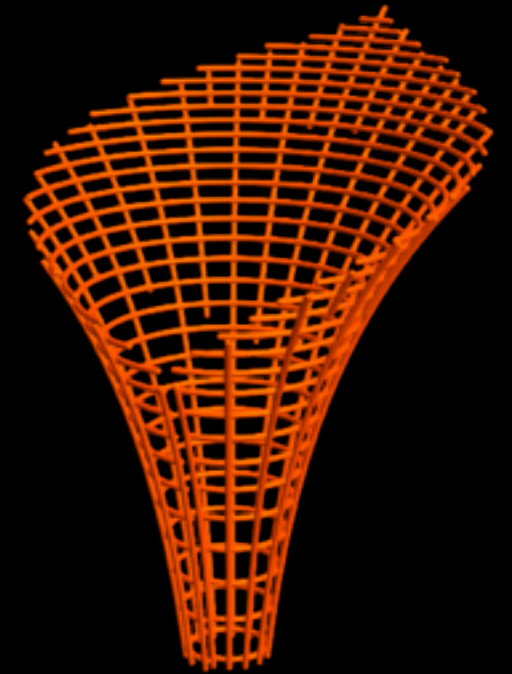
# Programming Local Curvatures



bottom

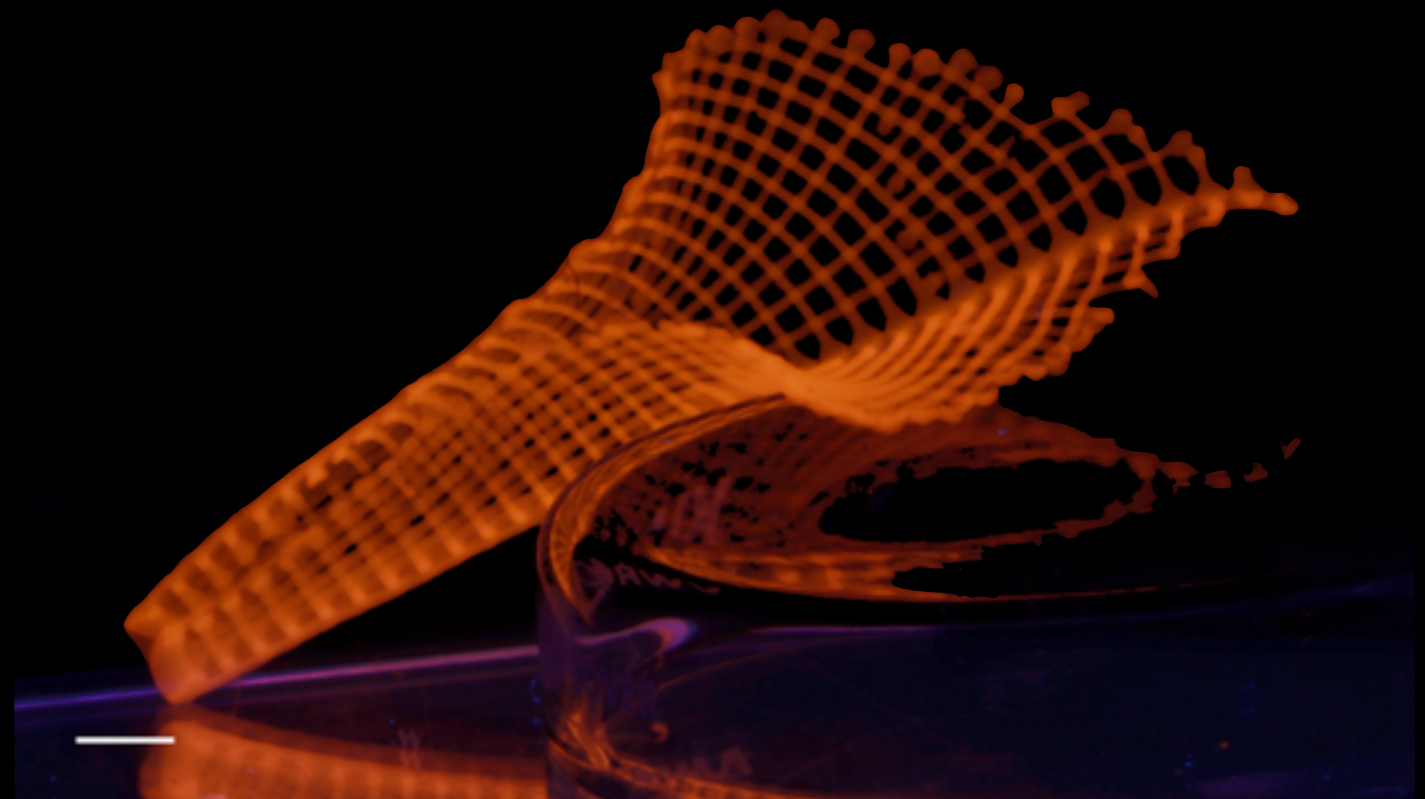


top



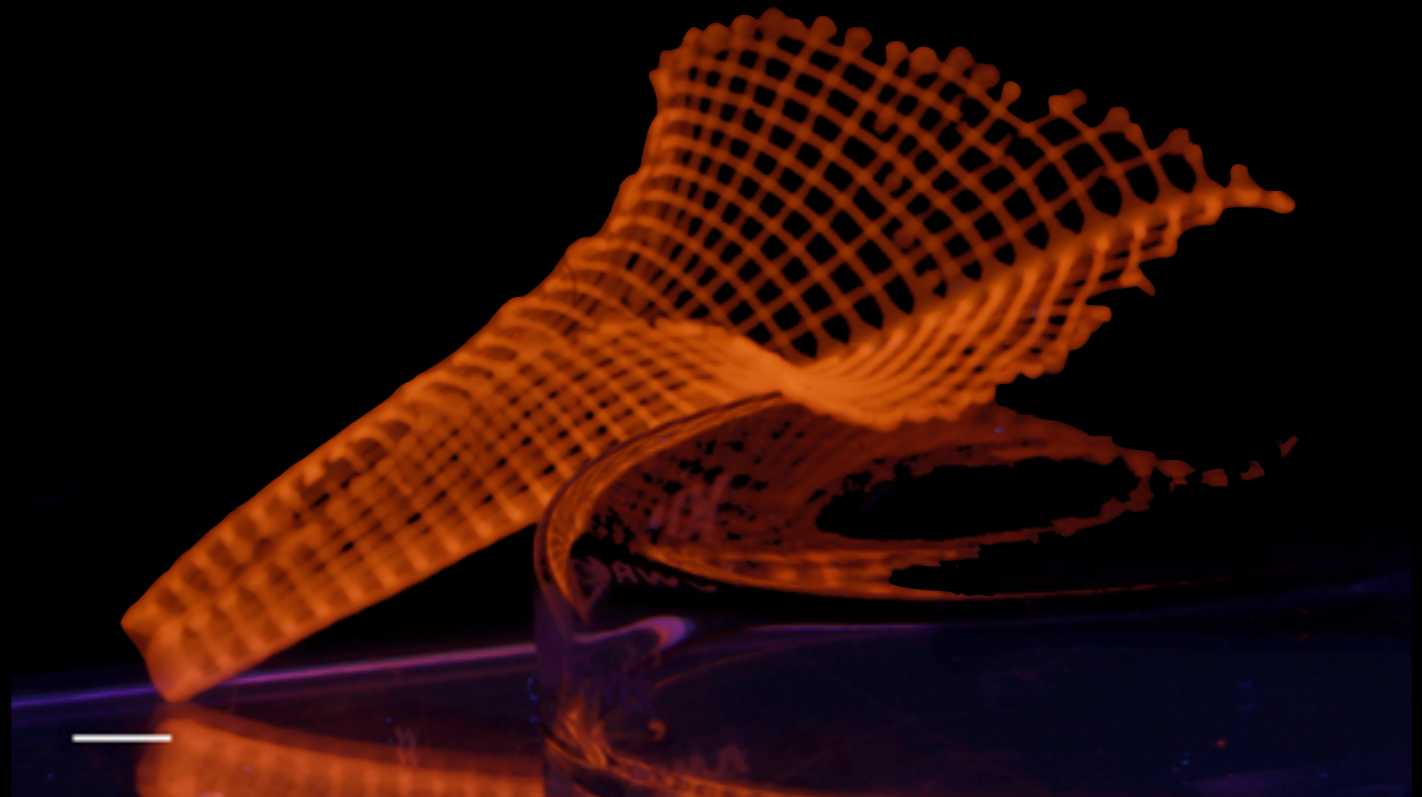
# Conclusions and Future Directions

- 3D printing hydrogel ink + cellulose nanofibrils simultaneously encodes anisotropy in swelling and elastic modulus. Complexity is free with additive manufacturing techniques.
- Local swelling anisotropy in a bilayer system generates curvature.
- Elasticity theory of anisotropic plates and shells allows us to predict mean and Gaussian curvatures.
- The inverse problem: How may we design print paths associated with specific target surfaces?
- Platform technology can be used with multi-stimuli responsive inks: light, temperature, electric field, hydration.



# Acknowledgements

- **Ms. A. Sydney Gladman** - Harvard SEAS
- Prof. L. Mahadevan - Harvard SEAS
- Prof. Jennifer Lewis - Harvard SEAS
- NSF MRSEC DMR 14-20570
- NSF DMREF 15-33985
- Army Research Office Award W911NF-13-0489



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Thank you!

